

A FINITE DIFFERENCE BASED FINITE STRIP

METHOD FOR THE ANALYSIS OF

TRANSLATIONAL SHELL STRUCTURES

by

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A thesis submitted in partial fulfilment of the requirements for the Degree of Master of Science in the Faculty of Engineering, University of Cape Town.

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SYNOPSIS

A numerical method for the analysis of translational shell structures is presented. The finite strip concept is utilised together with finite difference approximations to the differential equations along nodal lines. Numerical examples include open and closed translational shells, with various end conditions and continuity over intermediate supports.

DECLARATION OF CANDIDATE

I, Roger Barker, hereby declare that this thesis is my own work and that it has not been submitted for a degree at another University.

Signed by candidate

September, 1976.

This is my commandment, That ye love one another,
as I have loved you.

John 15:12

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NOTATIONUPPER CASE CHARACTERS

A, B, C, D	Bending stiffness matrices.
D''''	Fourth derivative of displacement vector D .
D_x, D_y, D_{xy}	Flexural rigidities for orthotropic plates.
E	Load vector for bending forces.
E_x, E_y	Young's modulus in x and y directions.
G	Modulus of elasticity in shear.
J	Load vector for in-plane forces.
K, L, M	In-plane stiffness matrices.
M_x, M_y	Bending moments per unit length perpendicular to x and y axes respectively.
M_{xy}	Twisting moment per unit length perpendicular to x axis.
M'_x, M'_y, M'_{xy}	Bending and twisting forces in local co-ordinate system.
T	Transformation matrix.
U_p, U_b	Total potential energy for in-plane and bending conditions respectively.
$\delta U_p, \delta U_b$	First variation of Total Potential Energy for in-plane and bending conditions respectively.
$\Delta \delta U_p, \Delta \delta U_b$	"Part of" expressions given above.
X, Y, Z	Rectangular co-ordinate axes.
X_i, Y_i	Edge forces on strip in x and y directions respectively.

LOWER CASE CHARACTERS

d_x, d_y, d_{xy}	Modified Young's modulus values for in-plane forces.
h	Strip thickness.
r_x, r_y, r_{xy}	Radii of curvature in x and y directions and for twisting moment respectively.
u, v, w	Global displacements in x, y and z directions respectively.

GREEK CHARACTERS

γ_{xy}	Shear strain.
ϵ_x, ϵ_y	Strains in x and y directions respectively.
θ	Global rotation about x axis.
σ_x, σ_y	Direct stresses in x and y direction respectively in global co-ordinates.
σ_x^i, σ_y^i	Direct stresses in x and y directions respectively in local co-ordinates.
τ_{xy}	Shear stress in global co-ordinates.
τ_{xy}^i	Shear stress in local co-ordinates.
ν_x, ν_y	Poisson's ratio in x and y directions respectively.
ϕ	Angle strip makes with global axis system.

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CHAPTER IINTRODUCTION

The literature on the analysis of folded plate structures may be considered to fall into seven principal categories with regard to method of analysis. They are:-

- (1) Beam method.
- (2) Folded plate theory, neglecting relative joint displacements.
- (3) Folded plate theory considering relative joint displacements.
- (4) "Elasticity" method.
- (5) Finite difference method.
- (6) Finite element method.
- (7) Finite strip method.

The Beam Method uses conventional beam theory, which requires all cross-sections to remain the same under load. If intermediate transverse stiffening ribs are frequent, this method yields good results, but more rigorous analyses and experiments have shown that this method cannot be applied to more intricate structures.

Methods (2) and (3) both consider the longitudinal supporting action of each plate to be governed by beam theory and the transverse supporting action to be that of a continuous one-way slab. Method (2) assumes that the changes in transverse bending moment and in longitudinal stresses due to relative joint displacements are negligible in comparison with the values of these moments and stresses computed on the basis of no relative joint displacements. For the general case however, these stresses are not negligible and the method is not recommended.

Folded Plate Theory Considering Relative Joint Displacements (3) takes into account the effect of relative displacement of the joints or the transverse moments and membrane stresses. Practical methods of analysis include those by V.Z. Vlasov, which uses a Fourier Series approximation; the Portland Cement Association; Gaafar, which uses the principle of superposition; and Yitzhaki who has also included the application of plasticity to folded plate structures.

The "Elasticity" Method, developed by Goldberg and Leve combines the equations of classical plate theory, for loads normal to the plane of the plates, and

the elasticity equations defining the plane stress problems, for loads in the plane of the plates. Applied loading is approximated by a Fourier Series.

Methods (1) to (4) are generally restricted to single spans with simple loading and edge conditions. (Reference [1] contains a detailed discussion on these methods.)

The application of Finite Difference Equations to shell analysis was performed in the 1950's, the advent of high speed digital computers making it possible to solve the resulting linear algebraic equations.

The method of Finite Elements was developed by many authors, and folded plate problems have been solved using triangular, rectangular, quadrilateral, isoparametric and thick-shell elements. With this method all possible boundary conditions are soluble [14].

A Finite Strip method of analysis was developed by Y.K. Cheung, first for flat plates [20] and then for folded plate structures [4]. In this method the structure is sub-divided into longitudinal strips which extend from one boundary to the other, making the method ideal for the solution of constant cross-section shell structures. Linear displacement functions are used for the in-plane displacements and cubic polynomials for the displacements normal to the strip surface. The displacements along the length of the strip are approximated by Fourier Series harmonics, likewise the strip loading. The Principle of Minimum Potential Energy is utilised to give the strip stiffness matrix in explicit form. The solution of the displacements is achieved in the normal manner.

Since Fourier Series utilisation becomes impractical for structures with more than three spans, and free boundary conditions cannot be used, du Preez [7] developed a general finite strip method which is capable of solving continuous structures with no boundary condition restriction. The interconnection of strips and other elements is also claimed.

Louw [8] subsequently developed a finite strip method of analysis for flat plates using finite difference approximations to the differential equations along the strip edges. Clamped and simple supported boundaries were considered.

This thesis is an extension of Louw's work and includes in-plane forces

enabling constant cross-section folded plates, closed box structures and translational shells to be analysed. Support conditions include simple, fixed and guided ends as well as continuity over interior supports.

CHAPTER 2

GOVERNING DIFFERENTIAL EQUATIONS FOR FINITE STRIP ANALYSIS

Figure 2.1 shows a typical finite strip element. The strip is of constant cross-sectional shape and has four degrees of freedom per edge, (i.e. u_i, v_i, w_i, θ_i). The right-hand axis system, and the corresponding displacements are also shown.

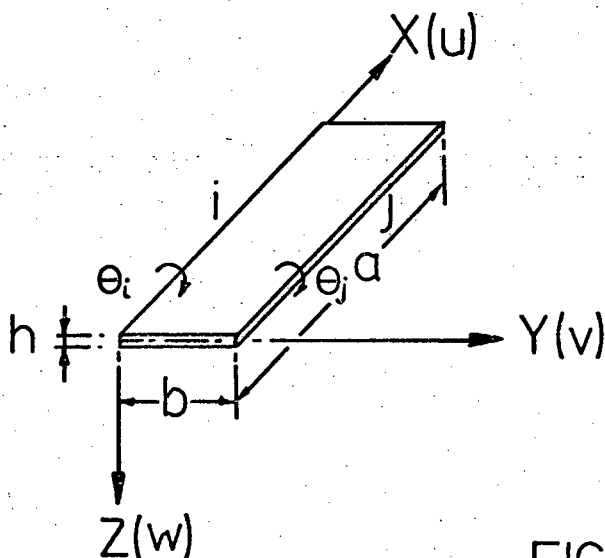


FIG. 2.1

Since in-plane, or membrane stresses, and bending stresses are not coupled, it is convenient to derive the equations for the two cases separately.

2.1 In-plane stiffness of strip

Consider the plate element in Figure 2.2 subjected to in-plane or membrane stresses as shown.

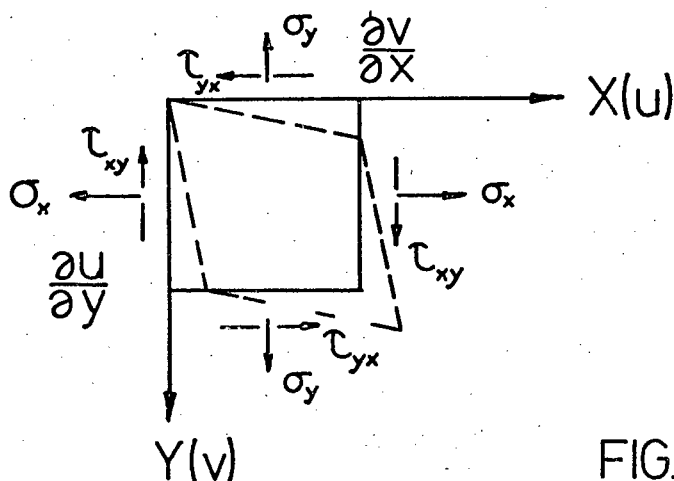


FIG. 2.2

Strains in the X, Y directions and the shear strain can be given by the following linear functions of the displacement gradients:-

$$\begin{aligned}\epsilon_x &= \frac{\partial u}{\partial x} \\ \epsilon_y &= \frac{\partial v}{\partial y} \\ \gamma_{xy} &= \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y}\end{aligned}\tag{2.1}$$

Writing these equations in terms of stresses and including the effect of Poisson's Ratio:-

$$\begin{aligned}\epsilon_x &= \frac{\sigma_x}{E_x} - \nu_y \frac{\sigma_y}{E_y} \\ \epsilon_y &= \frac{\sigma_y}{E_y} - \nu_x \frac{\sigma_x}{E_x}\end{aligned}$$

The stress terms can be made the dependent variables, hence:-

$$\begin{aligned}\sigma_x &= \frac{E_x}{(1 - \nu_x \nu_y)} [\epsilon_x + \nu_y \epsilon_y] \\ \sigma_y &= \frac{E_y}{(1 - \nu_x \nu_y)} [\epsilon_y + \nu_x \epsilon_x]\end{aligned}\tag{2.2}$$

Also $\tau_{xy} = G \cdot \gamma_{xy}$

Consider the strip as shown in Figure 2.1. Let the in-plane displacements at edge i be u_i and v_i , and those at edge j be u_j and v_j . The displacement at any point on the strip surface in the u direction can be made up of linear combinations of u_i and u_j .

Thus $u = f_1(u_i) + f_2(u_j)$,

where f_1 and f_2 are linear functions. Since u is a function of x and y , it is convenient to separate their dependence so that u_i and u_j are functions of x only and f_1 and f_2 , functions of y only.

Hence $u(x,y) = f_1(y) \cdot u_i(x) + f_2(y) \cdot u_j(x)$.

At this stage, only y dependent functions need be given any specific form as the x dependent functions are dealt with using finite difference approximations.

Since y increases from edge i to edge j , and $u(x,y)$ must have the value of $u_i(x)$ at edge i and $u_j(x)$ at edge j , $f_1(y)$ can be replaced by $(1 - y/b)$ and $f_2(y)$ by (y/b) ,

$$\text{i.e. } u(x,y) = \left(1 - \frac{y}{b}\right) \cdot u_i(x) + \left(\frac{y}{b}\right) \cdot u_j(x).$$

By similar reasoning:-

$$v(x,y) = \left(1 - \frac{y}{b}\right) \cdot v_i(x) + \left(\frac{y}{b}\right) \cdot v_j(x).$$

Dropping the (x,y) post-script for u and v and expressing the displacements in matrix form:-

$$\begin{bmatrix} u \\ v \end{bmatrix} = C_p \cdot \psi_p \quad (2.3)$$

$$\text{where } C_p = \begin{bmatrix} \left(1 - \frac{y}{b}\right) & 0 & \left(\frac{y}{b}\right) & 0 \\ 0 & \left(1 - \frac{y}{b}\right) & 0 & \left(\frac{y}{b}\right) \end{bmatrix} \quad (2.4)$$

$$\text{and } \psi_p = [u_i(x) \ v_i(x) \ u_j(x) \ v_j(x)]^T \quad (2.5)$$

(The subscript p indicates in-plane consideration).

Let the equations in (2.1) be represented by one strain matrix:-

$$[\epsilon] = \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{bmatrix} \quad (2.6)$$

Calculating partial derivatives and substituting into (2.6):-

$$\epsilon = \begin{bmatrix} \left(1 - \frac{y}{b}\right) \cdot u_i'(x) + \left(\frac{y}{b}\right) \cdot u_j'(x) \\ \left(-\frac{1}{b}\right) \cdot v_i(x) + \left(\frac{1}{b}\right) \cdot v_j(x) \\ \left(\left(-\frac{1}{b}\right) \cdot u_i(x) + \left(\frac{1}{b}\right) \cdot u_j(x)\right) + \left(\left(1 - \frac{y}{b}\right) \cdot v_i'(x) + \left(\frac{y}{b}\right) \cdot v_j'(x)\right) \end{bmatrix} \quad (2.7)$$

The stresses in (2.2) can be represented by one stress matrix:-

$$[\sigma] = \begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = [D_p][\epsilon] \quad (2.8)$$

Where

$$[D_p] = \begin{bmatrix} \frac{E}{1-\nu\nu} & \frac{\nu E}{1-\nu\nu} & 0 \\ \frac{\nu E}{1-\nu\nu} & \frac{E}{1-\nu\nu} & 0 \\ 0 & 0 & G \end{bmatrix} \quad (2.9)$$

and $[\epsilon]$ is as given in (2.6).

The Total Potential Energy of a finite strip for in-plane forces can be written:-

$$\begin{aligned} U_p &= \frac{h}{2} \int_X \int_Y (\sigma_x \cdot \epsilon_x + \sigma_y \cdot \epsilon_y + \tau_{xy} \cdot \gamma_{xy}) dy \cdot dx \quad (\text{Internal Strain Energy}) \\ &- \int_X (X_i \cdot u_i + Y_i \cdot v_i + X_j \cdot u_j + Y_j \cdot v_j) dx \quad (\text{Edge Forces}) \\ &- \int_X \int_Y (X \cdot u + Y \cdot v) dy \cdot dx \quad (\text{Surface Forces}) \end{aligned} \quad (2.10)$$

In matrix notation:-

$$\begin{aligned} U_p &= \frac{h}{2} \int_X \int_Y \epsilon^T \cdot D_p \cdot \epsilon \cdot dy \cdot dx - \int_X (u_i \ v_i \ u_j \ v_j) \begin{bmatrix} X_i \\ Y_i \\ X_j \\ Y_j \end{bmatrix} dx \\ &- \int_X \int_Y (u \ v) \cdot \begin{bmatrix} X \\ Y \end{bmatrix} dx \cdot dy \end{aligned} \quad (2.11)$$

Equation (2.11) is expanded term by term in its three natural subdivisions. The following substitutions are also made to avoid unnecessary complication:-

$$\begin{aligned} d_x &= \frac{E_x}{(1 - \nu_x \nu_y)} \\ d_y &= \frac{E_y}{(1 - \nu_x \nu_y)} \\ d_{xy} &= \frac{\nu_x E_y}{(1 - \nu_x \nu_y)} \end{aligned} \quad (2.12)$$

Expansion of Internal Strain Energy Term

Only an outline of the process is given as it involves excessive arithmetic.

- (a) Expansion of $D_p \cdot \epsilon$.
- (b) Evaluation of ϵ^T .
- (c) Multiplication of $\epsilon^T \cdot D_p \cdot \epsilon$.
- (d) Integrating with respect to y , across the strip.
- (e) Using the Principle of Minimum Potential Energy, the first variation of U_p with respect to the displacements involved ($u_i, v_i, u_j, v_j; u'_i, v'_i, u'_j, v'_j$) must be calculated and equated to zero.

The first variation of U_p with respect to u_i will be a partial differentiation of U_p with respect to u_i , which must be multiplied by the first variation of u_i itself, i.e. δu_i .

The process is repeated for each of the displacements mentioned which have first variations $\delta v_i, \delta u_j, \delta v_j; \delta u'_i, \delta v'_i, \delta u'_j$ and $\delta v'_j$.

- (f) The result of the process outlined in (e) above is then sorted into three matrices; a row vector involving the first variations of the displacements (1×8), a square non-symmetrical matrix involving the elements of equation (2.12), G , and b as well as the constant h (8×8) and a column vector involving the displacements (8×1).
- (g) The non-symmetrical (8×8) matrix is subdivided into four (4×4) matrices (K, L, M and M^T) as given in Appendix 'A'.

(h) Substituting (2.5) and grouping gives: (The Δ prefix to δU_p is used to indicate a part of the total δU_p).

$$\Delta \delta U_p = \int_X \left\{ \delta \psi_p^T [K \cdot \psi_p + M \cdot \psi_p'] + \delta \psi_p'^T [M^T \cdot \psi_p + L \cdot \psi_p'] \right\} dx \quad (2.13)$$

Expansion of Edge and Surface Force Terms

From equation (2.11) the edge force term can be written:

$$\Delta U_p = - \int_X \psi_p^T \cdot \begin{bmatrix} X_i \\ Y_i \\ X_j \\ Y_j \end{bmatrix} dx.$$

Let the load vector above be represented by $[J]$. When using finite difference approximations to the differential equations $[J]$ becomes:

$$[J] = \begin{bmatrix} \frac{X_i}{2} & \frac{Y_i}{SA} & \frac{X_j}{2} & \frac{Y_j}{SA} \end{bmatrix}^T,$$

where SA is the distance between nodes in the longitudinal direction.

The surface force terms are required, for analysis purposes, to be proportioned and placed on the adjoining strip edges. Thus all in-plane surface loads can be included in the J matrix above. Thus edge and surface forces may be represented by:

$$\Delta U_p = - \int_X \psi_p^T \cdot J \cdot dx. \quad (2.14)$$

The first variation of ΔU_p with respect to the displacements contained in ψ_p^T is:

$$\delta \Delta U_p = - \int_X \delta \psi_p^T \cdot J \cdot dx. \quad (2.15)$$

Integration with Respect to the Variable X

Equations (2.13) and (2.15) are added giving the total expression for the first variation of Total Potential Energy for one strip, i.e.

$$\delta U_p = \int_X \left\{ \delta \psi_p^T [K \cdot \psi_p + M \cdot \psi_p'] + \delta \psi_p'^T [M^T \cdot \psi_p + L \cdot \psi_p'] - \delta \psi_p^T \cdot J \right\} dx \quad (2.16)$$

Integration of term involving $\delta \psi_p'^T$ by parts

$$\delta U_p = \delta \psi_p^T [M^T \cdot \psi_p + L \cdot \psi_p'] \Big|_0^a + \int_X \delta \psi_p^T [-(M^T \cdot \psi_p' + L \psi_p'') + (K \cdot \psi_p + M \psi_p') - J] dx$$

$\delta \psi_p^T$ is an arbitrary displacement vector which can be zero or any small finite quantity depending on the boundary conditions prescribed. For the Total Potential Energy to be a minimum, U_p is differentiated with respect to the generalised displacements, one at a time and each must be zero, hence:

$$\delta \psi_p^T [M^T \cdot \psi_p + L \cdot \psi_p'] \Big|_0^a = 0 \quad (2.17)$$

and

$$\int_X \delta \psi_p^T [-(M^T \cdot \psi_p' + L \cdot \psi_p'') + (K \cdot \psi_p + M \cdot \psi_p') - J] dx = 0 \quad (2.18)$$

Equation (2.17) is a boundary condition equation which must be satisfied at $x = 0$ or $x = a$. Equation (2.18) represents the strip section away from the boundaries where the displacement vector, $\delta \psi_p^T$ is generally non-zero, and hence for equation (2.18) to be satisfied, the integrand must be zero, giving:

$$\delta \psi_p^T [M^T \cdot \psi_p + L \cdot \psi_p'] \Big|_0^a = 0 \quad (2.19)$$

$$-L \cdot \psi_p'' + (M - M^T) \cdot \psi_p' + K \cdot \psi_p - J = 0$$

2.2 BENDING STIFFNESS OF STRIP

Consider a strip cross-section as shown in Figure 2.3 below:

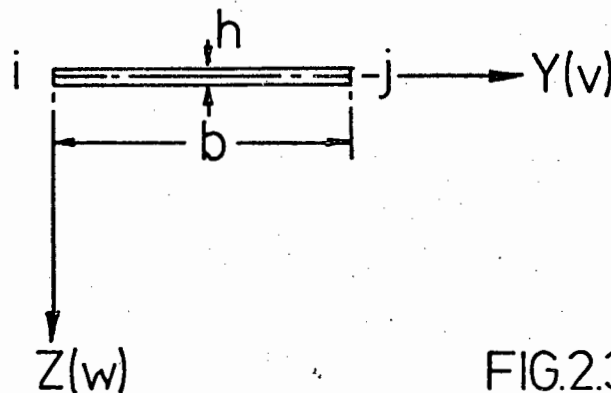


FIG.2.3

All possible displacements of this cross-section which cause bending or twisting stresses can be represented by combinations of the following displacements. The appropriate cubic polynomial displacement functions are also given: (Φ)

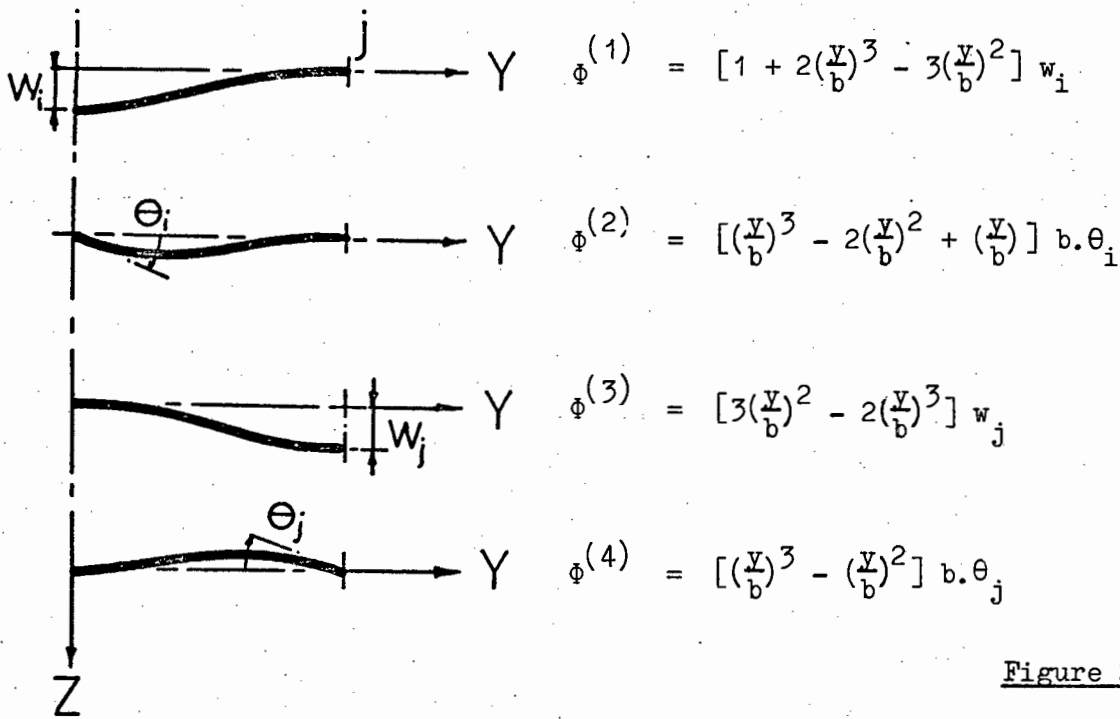


Figure 2.4

As with the in-plane displacements, it is convenient to separate the x and y dependence and make the edge displacements functions of x alone. The displacement of any point on the strip surface is then given by the summation of the four displacements. In matrix form:

$$w = C_b \cdot u_b, \quad (2.20)$$

where

$$C_b = \left[\left(1 - \frac{3y^2}{b^2} + \frac{2y^3}{b^3}\right) \left(y - \frac{2y^2}{b} + \frac{y^3}{b^2}\right) \left(\frac{3y^2}{b^2} - \frac{2y^3}{b^3}\right) \left(-\frac{y^2}{b} + \frac{y^3}{b^2}\right) \right] \quad (2.21)$$

and

$$u_b = [w_i(x) \theta_i(x) w_j(x) \theta_j(x)]^T \quad (2.22)$$

The curvature displacement relationship for thin plates may be written:

$$\chi = \left[-\frac{\partial^2 w}{\partial x^2} \mid -\frac{\partial^2 w}{\partial y^2} \mid \frac{2\partial^2 w}{\partial x \partial y} \right]^T \quad (2.23)$$

and the relationship between moment and curvature may be expressed as

$$M = D \chi \quad (2.24)$$

where

$$D = \begin{bmatrix} D_x & D_1 & 0 \\ D_1 & D_y & 0 \\ 0 & 0 & D_{xy} \end{bmatrix} \quad (2.25)$$

and has individual elements:

$$\begin{aligned} D_x &= \frac{E_x h^3}{12(1 - \nu_x \nu_y)} \\ D_y &= \frac{E_y h^3}{12(1 - \nu_x \nu_y)} \\ D_1 &= \frac{\nu_x E_y h^3}{12(1 - \nu_x \nu_y)} \\ D_{xy} &= \frac{Gh^3}{12} \end{aligned} \quad (2.26)$$

The Total Potential Energy for bending forces acting on a finite strip can be written: (the subscript b indicates bending consideration)

$$\begin{aligned} U_b &= \frac{1}{2} \int_X \int_Y (M_x \cdot \frac{1}{r_x} + M_y \cdot \frac{1}{r_y} + 2M_{xy} \cdot \frac{1}{r_{xy}}) dy dx - \int_X \int_Y w \cdot p \cdot dy dx \\ &\quad - \int_0^a (M_{xx} \cdot \theta_y + M_{yy} \cdot \theta_x + Q \cdot w) dx \end{aligned} \quad (2.27)$$

Rewriting equation 2.27 with the surface load expression included in the nodal load expression; and using matrix notation (Shear Strains have been neglected)

$$U_b = \frac{1}{2} \int_X \int_Y \chi^T \cdot D \cdot \chi dy dx - \int_X \int_Y C_b \cdot u_b \cdot p \cdot dy dx$$

or

$$U_b = \frac{1}{2} \int_X \int_Y \chi^T \cdot D \cdot \chi dy dx - \int_X \int_Y u_b^T \cdot C_b^T \cdot p \cdot dy dx \quad (2.28)$$

When use is made of finite differences, loading terms become simple manipulations which are performed when filling out the load matrix. Thus $C_b^T \cdot p$ can be replaced by a matrix E, which at present requires no further definition.

Expansion of Matrices

As with the "in-plane" matrix expansion, the calculations involved in expanding

(2.28) are both tedious and voluminous. An outline of the process is:-

- (a) Multiplication of D and χ .
- (b) Evaluation of χ^T .
- (c) Pre-multiplying the result from (a) by that from (b).
- (d) Integrating with respect to y and evaluating the integrand from 0 to b .
- (e) Differentiating with respect to each displacement present ($w_i, \theta_i, w_j, \theta_j; w'_i, \theta'_i, w'_j, \theta'_j; w''_i, \theta''_i, w''_j, \theta''_j$).
- (f) Grouping the result to give:

$$\delta U_b = \int_X (\delta u^{T[A.u'' + B.u]} + \delta u'^T[C.u'] + \delta u^T[B^T.u'' + D.u - E]) dx \quad (2.29)$$

The matrices A, B, C and D are given in Appendix B.

Integrating by parts:-

$$\begin{aligned} \delta U_b = & \left\{ \delta u'^T(A.u'' + B.u) \Big|_0^a - [\delta u^T[A.u'''' + B.u']] \Big|_0^a - \int_X \delta u^T(A.u'''' + B.u'') dx \right\} \\ & + \left\{ \delta u^T.C.u' \Big|_0^a - \int_X \delta u^T.C.u'' dx \right\} \\ & + \int_X \delta u^T(B^T.u'' + D.u - E) dx \end{aligned} \quad (2.30)$$

Grouping (2.30) as products of $\delta u'^T$, and δu^T

$$\begin{aligned} \delta U_b = & \delta u'^T[Au'' + Bu] \Big|_0^a - \delta u^T[Au'''' + (B - C) u'] \Big|_0^a \\ & + \int_X \delta u^T[Au'''' + (B - C + B^T) u'' + Du - E] dx \end{aligned} \quad (2.31)$$

Using the same reasoning as in formulating equations (2.19), to satisfy the requirements that Total Potential Energy be a minimum:-

$$\begin{aligned} \delta u'^T[A.u'' + B.u] \Big|_0^a & = 0 \\ \delta u^T[A.u'''' + (B - C) u'] \Big|_0^a & = 0 \\ A.u'''' + (B - C + B^T) u'' + D.u - E & = 0 \end{aligned} \quad (2.32)$$

2.3 COMBINATION OF IN-PLANE AND BENDING FORCES

The in-plane and bending stiffness matrices have been developed separately as there is no connection between the two systems of forces. In the combination of these two sets of forces, matrices are simply enlarged with a separate space for each system. This is shown in Figure 2.4 below.

u_i	v_i	w_i	θ_i	u_j	v_j	w_j	θ_j	
In-Plane		0		In-Plane		0		u_i
0		Bending		0		Bending		v_i
In-Plane		0		In-Plane		0		w_i
0		Bending		0		Bending		θ_i
In-Plane		0		In-Plane		0		u_j
0		Bending		0		Bending		v_j
In-Plane		0		In-Plane		0		w_j
0		Bending		0		Bending		θ_j

Figure 2.4

All strip stiffness matrices are now (8×8) , compared with (4×4) previously.

The governing differential equation for a single finite strip is composed of the summation of the last equations of (2.19) and (2.32).

In general terms, this summation can be expressed

$$\alpha D'''' + \beta D''' + \gamma D'' + \epsilon D' + \zeta D = \lambda \quad (2.33)$$

where: $\alpha = A$
 $\beta = 0$
 $\gamma = (B - C + B^T) - (L)$
 $\epsilon = (M - M^T)$
 $\zeta = (D + K)$
 $\lambda = (E + J)$ (Appendix B)

(2.34)

The full α , γ , ϵ , ζ and λ matrices are simply an addition, as indicated in Figure 2.4, and are given in Appendix C.

With $\beta = 0$, (2.33) becomes:-

$$\alpha D'''' + \gamma D'' + \epsilon D' + \zeta D = \lambda \quad (2.35)$$

2.4 TRANSFORMATION MATRICES

Equation (2.35) is only suitable for solving flat plate problems. The Transformation Matrix derived below enables folded plate problems to be solved as well.

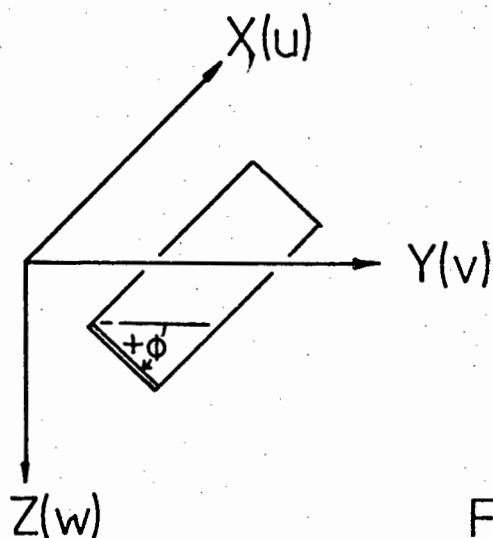


FIG.2.5

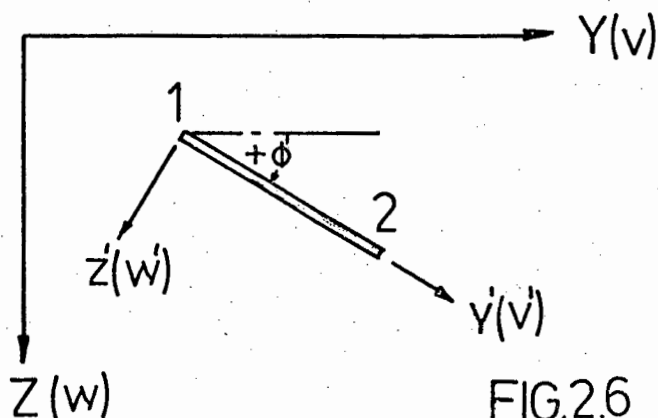


FIG.2.6

The angle ϕ is measured from the main-axis direction to the local-axis direction. This angle is considered positive if clockwise and negative if anti-clockwise. The Y' axis is parallel to the strip and in the direction of increasing node number.

Consider Figure 2.6; the local axis displacements can be derived from global displacements: (Let local axis displacements be indicated by a prime)

$$\begin{aligned} u' &= u \\ v' &= v \cos \phi + w \sin \phi \\ w' &= -v \sin \phi + w \cos \phi \\ \theta' &= \theta. \end{aligned} \tag{2.36}$$

Or in matrix form:

$$\begin{aligned} u' \\ v' \\ w' \\ \theta' \end{aligned} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & \cos \phi & \sin \phi & 0 \\ 0 & -\sin \phi & \cos \phi & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} u \\ v \\ w \\ \theta \end{bmatrix} = R.D$$

This is the Transformation Matrix for one edge of the strip; for both edges, the matrix is duplicated thus:-

$$T = \begin{bmatrix} R & | & 0 \\ \hline 0 & | & R \end{bmatrix} \tag{2.37} \tag{2.37}$$

Following the normal transformation procedures, equation (2.35) is written:

$$T^T \alpha T D''' + T^T \gamma T D'' + T^T \epsilon T D' + T^T \zeta T D = \lambda \tag{2.38}$$

This equation makes possible the solution of folded plate problems. λ is the load matrix where nodal-global loads are substituted. The matrices D''' , D'' and D' have global finite difference approximations to the differential equation along the length of the strip.

2.5 FINITE DIFFERENCE APPROXIMATIONS

The finite difference approximations to the differential equation along a nodal line are derived as for beam elements. All the boundary conditions required in equations (2.19) and (2.32) are satisfied by the use of standard Operator Patterns as given in Appendix D.

The three boundary conditions which occur in this type of structure are; rigidly fixed, simply supported and guided. Each of the four displacements (u, v, w, θ) must be given an operator pattern for each of the above cases.

The displacements v and w become interchanged for a strip on edge, and for any strip which makes an angle to the global co-ordinate system there is the corresponding reciprocation between these two displacements. Thus v and w displacements must always have the same operator patterns. The operator pattern for the θ displacement has the same form as the w displacement.

The u displacement may be either fixed or free. The free operator pattern is derived for a simple rod element having the same values of displacement beyond the boundary as at the boundary node.

The three cases are as follows:-

Rigidly Fixed: u Fixed
 v Fixed
 w Fixed
 θ Fixed

Simply Supported: u Free
 v Simply Supported
 w Simply Supported
 θ Simply Supported

Guided Support: u Fixed
 v Guided
 w Guided
 θ Guided

All nodes at the end of a structure are usually given the same boundary condition.

2.6 EXTENSION OF EQUATION (2.39) FOR COMPUTER USE

Consider the following folded plate structure shown in Figure 2.7.

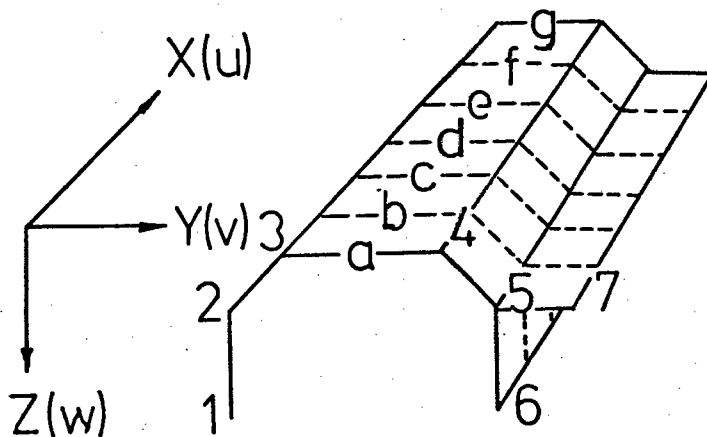


FIG.27

Steps in the solution of nodal displacements are:-

- Matrices α , γ , ϵ and ζ in equation (2.38) are evaluated.
- The transformation matrices and their transposes are calculated for each strip.
- Matrix multiplication yields the following condensed form of equation (2.38) for each strip:-

$$(AL) D'''' + (FA) D'' + (EP) D' + (ZE) D = \lambda$$

- To compile this equation for the complete cross-section the matrices are added where strip boundaries coincide as shown in Figure 2.8.

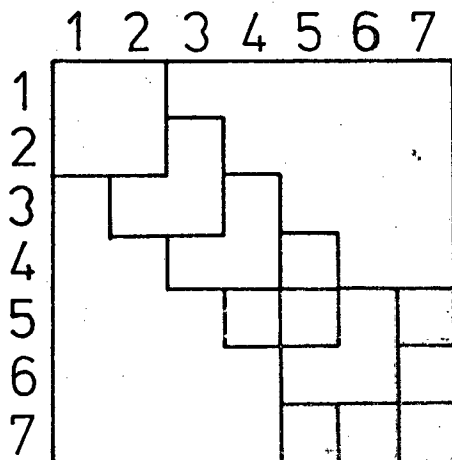


FIG.28

- Notes:
1. Each small block represents a (4×4) matrix; with corresponding u, v, w and θ displacements.
 2. The (8×8) matrix covering nodes 1 and 2 represents the first strip going from node 1 to 2, the second (8×8) block represents the second strip going from node 2 to 3, and so on.
 3. The strip going from node 5 to 7 is broken into (4×4) matrices and is placed as shown by the shaded squares.

- (e) The Finite Difference approximations run along the strip boundaries and always represent global displacements. The total equation to be solved is shown in Figure 2.9.
- (f) The combined stiffness matrix is found by the addition of all components in the equation under step (c).
- (g) The nodal loads are represented in vector form (λ) .
- (h) When compiling the stiffness matrix $[K]$, the cross-section matrices are derived for the unit strip length, only a portion of these matrices must be considered. Thus the value $S/(\text{Number of cross-sections})$ must multiply all the matrices (AL, FA, EP and ZE), where S is the nodal spacing.

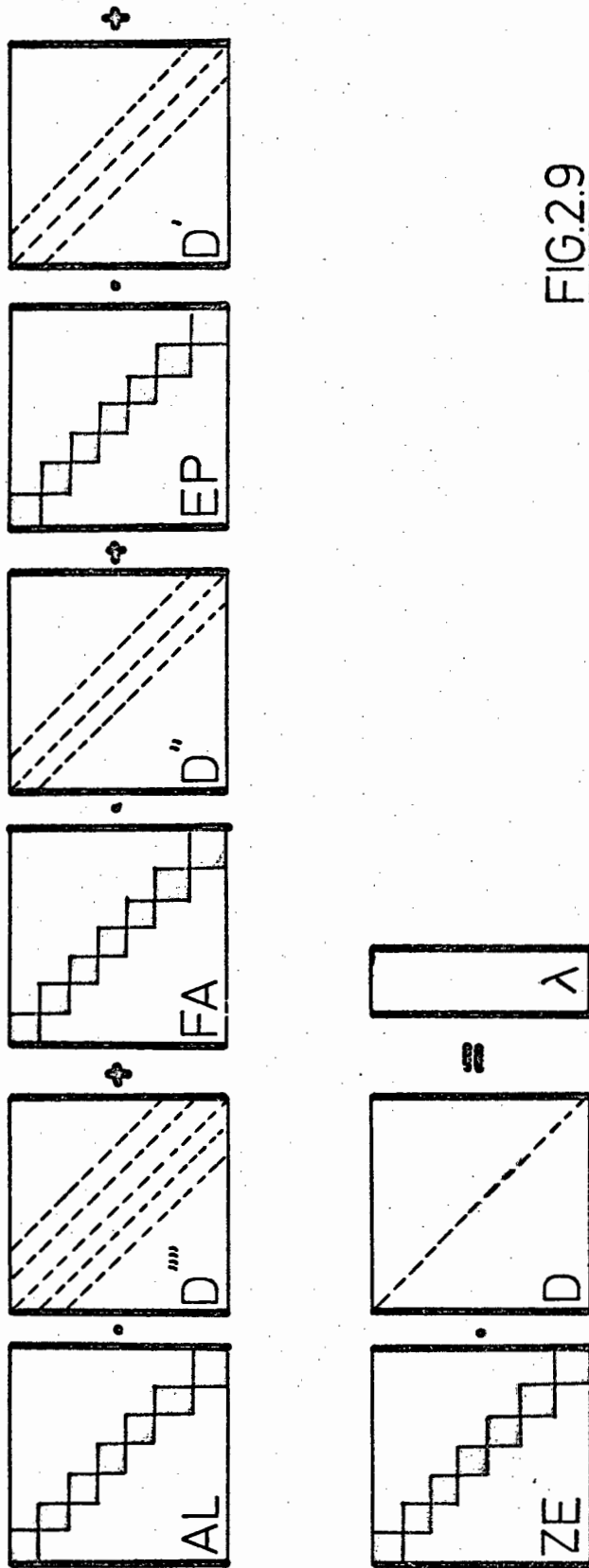


FIG.2.9

- Note:
1. Each shaded block in the AL, FA, EP, and ZE matrices are compiled as shown in Fig. 2.8.
 2. The Derivative Matrices are sparsely populated and hence are more compactly stored in the computer analysis. The D matrix is a scalar matrix.
 3. When matrix arithmetic is done the form of this equation is $[K][D] = [\lambda]$, where K is a symmetric stiffness matrix.

2.7 STEPS IN SOLUTION OF STRESSES AND MOMENTS FROM DISPLACEMENTS

- (a) All displacements are transformed into their respective local axis system:-

$$\begin{bmatrix} D'_1 \\ D'_2 \end{bmatrix} = \begin{bmatrix} T_{12} & 0 \\ 0 & T_{12} \end{bmatrix} \cdot \begin{bmatrix} D_1 \\ D_2 \end{bmatrix} \quad (2.39)$$

where the strip is envisaged as going from node 1 to node 2.

- (b) The stresses and moments are then calculated by the following formulae:-

$$\sigma'_x = d_x \frac{\partial u}{\partial x} + d_{xy} \frac{\partial v}{\partial y}$$

$$\sigma'_y = d_{xy} \frac{\partial u}{\partial x} + d_y \frac{\partial v}{\partial y}$$

$$\tau'_{xy} = G \left[\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right]$$

(2.40)

$$M'_x = - \left[D_x \frac{\partial^2 w}{\partial x^2} + D_1 \frac{\partial^2 w}{\partial y^2} \right]$$

$$M'_y = - \left[D_1 \frac{\partial^2 w}{\partial x^2} + D_y \frac{\partial^2 w}{\partial y^2} \right]$$

$$M'_{xy} = D_{xy} 2 \frac{\partial^2 w}{\partial x \partial y}$$

CHAPTER 3

NUMERICAL EXAMPLES

3.1 Numerical Examples

This chapter contains various numerical examples to illustrate the various displacement, stress and force results as obtained from the computer program.

The results of the first example are compared with those obtained by using conventional beam theory, while the more complicated examples are compared with the results obtained by other researchers in the same field.

EXAMPLE NO. 1:

This example was chosen to illustrate the three different boundary conditions discussed previously, as well as relevant stress and deflection curves.

Ex. 1a

Cross-section : Thick walled box section as shown.
 Span : 12,000 m
 Boundary conditions : Simply supported - simply supported.
 Loading : Uniformly distributed load of 9,96 kN/m.
 Young's modulus (X and Y): 31,0E09
 Poisson's ratio (X and Y): 0,00
 Program C.P.U. time : 1 min 6 sec.

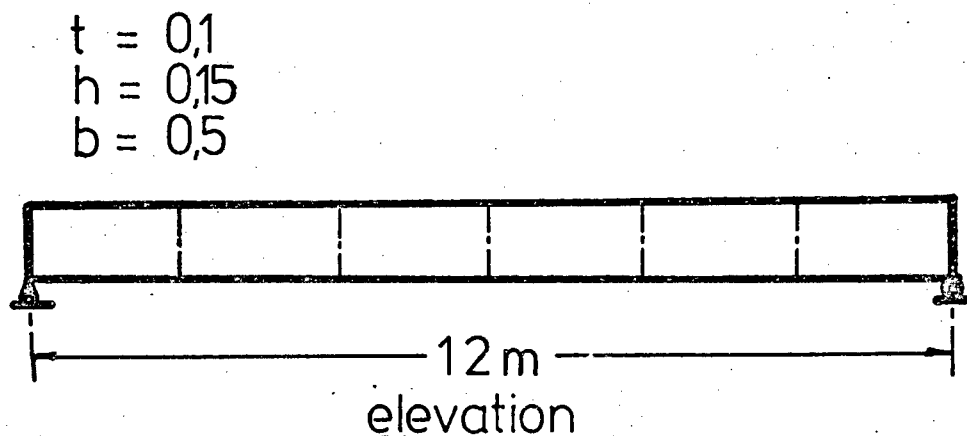
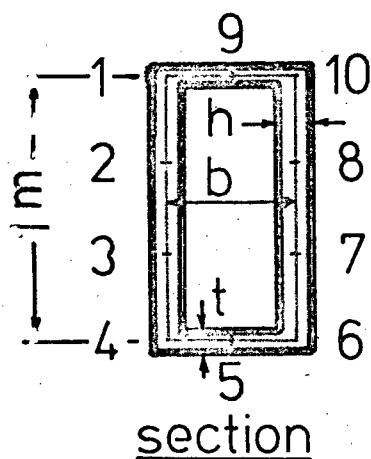


FIG.3.1

Shear lag at beam ends

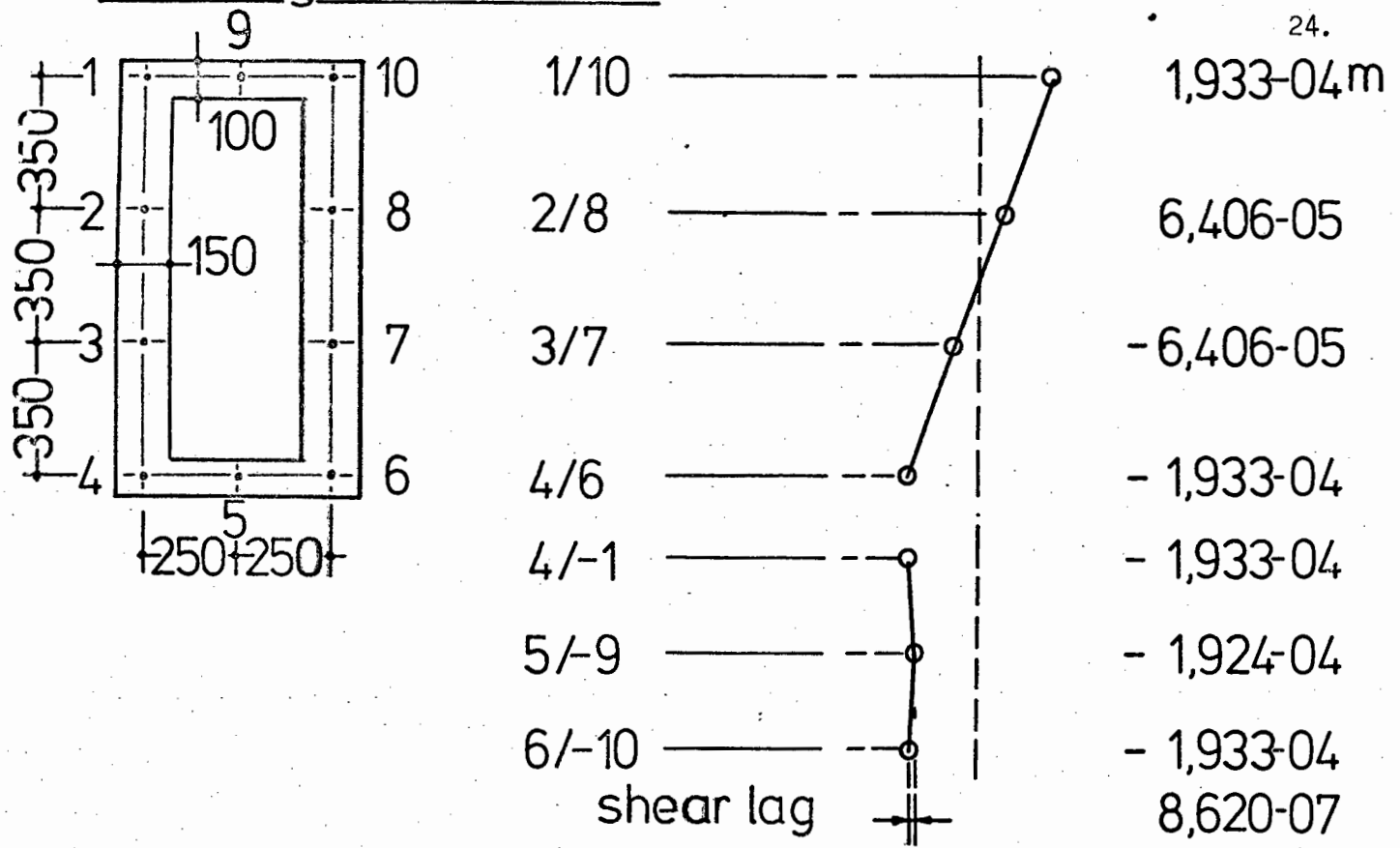


FIG.3.4

Ex. 1b

- Cross-section : As in example 1a.
- Span : 12,000 m
- Boundary conditions : Simply supported - Guided.
- Loading : Uniformly distributed load of 9,96 kN/m.
- Young's Modulus (X and Y) : 31,0E09.
- Poisson's Ratio (X and Y) : 0,00.
- Program C.P.U. time : 58 sec.

Vertical deflection

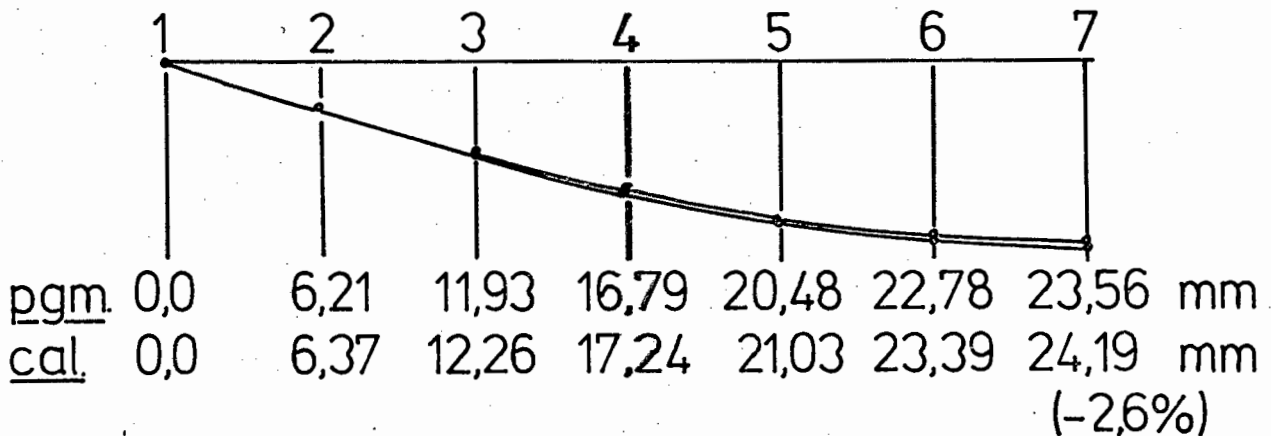


FIG.3.5

σ_x on centre line of horizontal strips

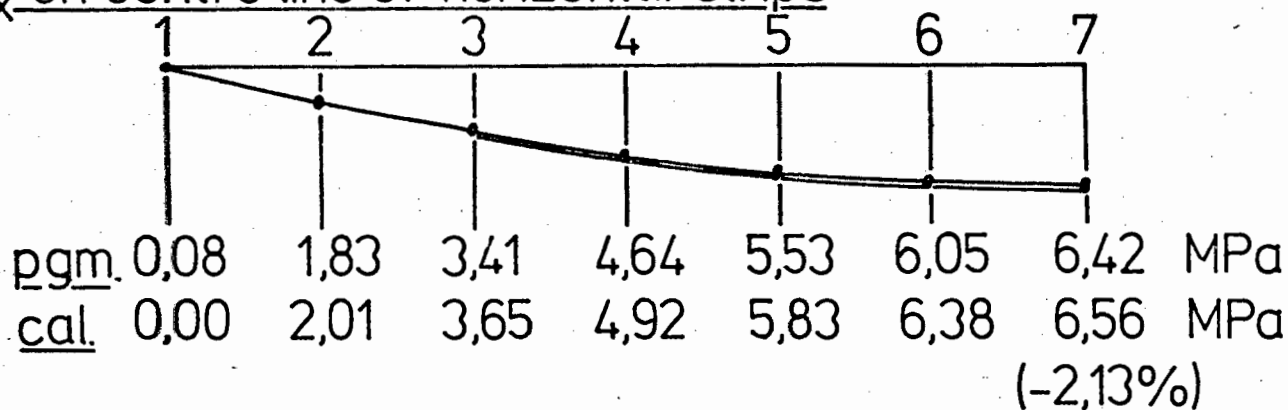


FIG.3.6

Shear stress distribution 2m from l.h. end

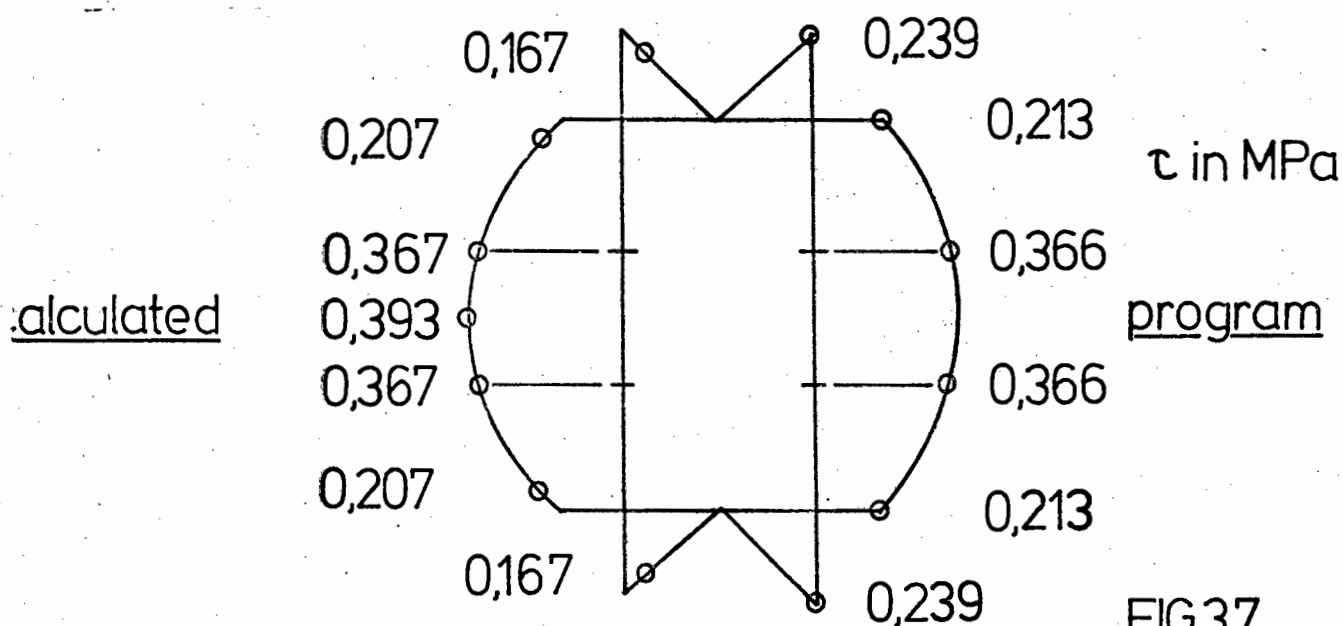


FIG.3.7

Ex. 1c

Cross-section	: As in example 1a
Span	: 12,000 m.
Boundary conditions	: Fixed - Simply supported.
Loading	: Uniformly distributed loading of 9,96 kN/m.
Young's Modulus (X and Y)	: 31,0E09.
Poisson's Ratio (X and Y)	: 0,00
Program C.P.U. time	: 1 min 8 sec.

Vertical deflection

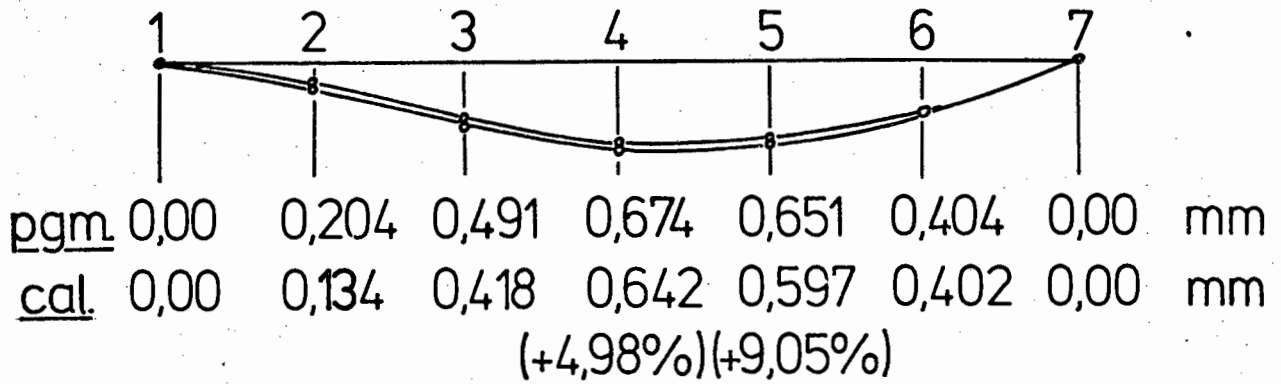


FIG.3.8

σ_x on centre line of horizontal strips

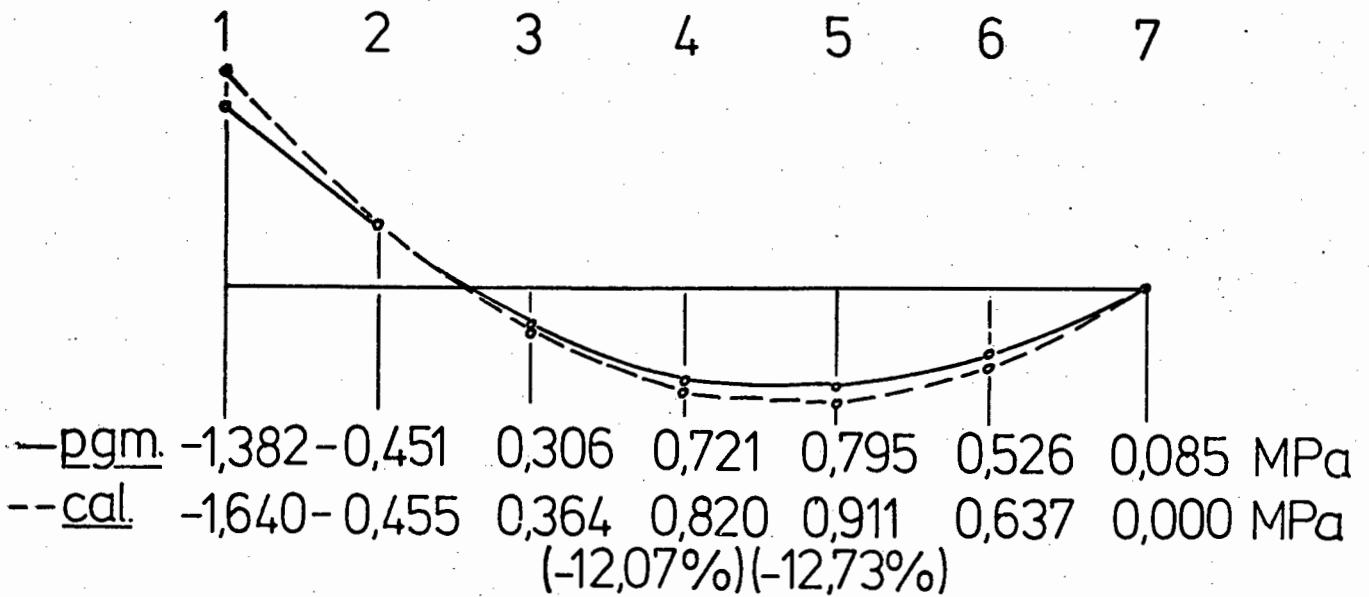


FIG.3.9

u displacement curves (x direction)

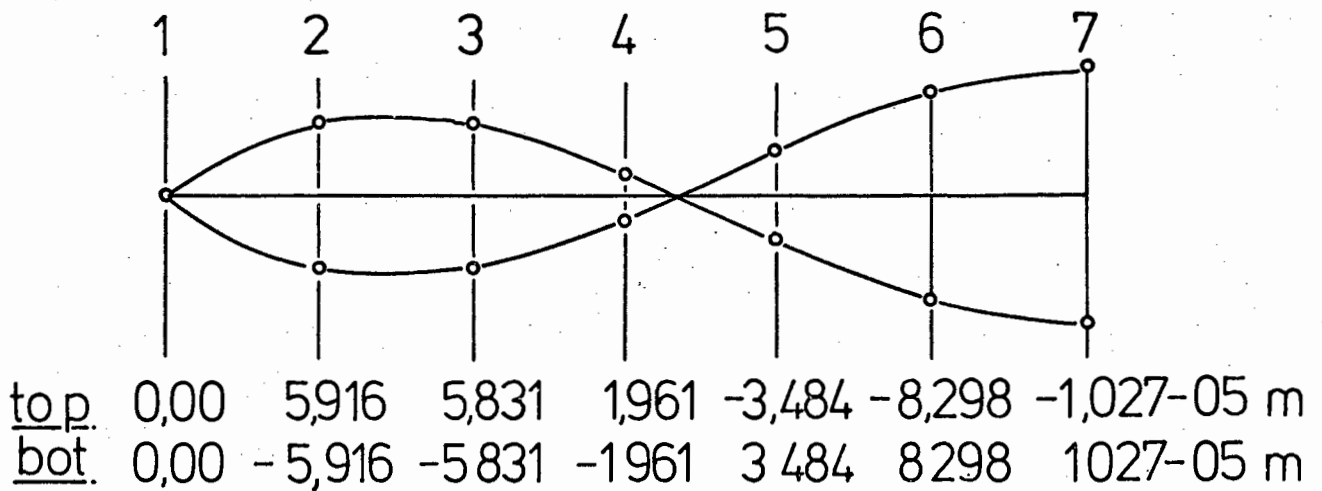
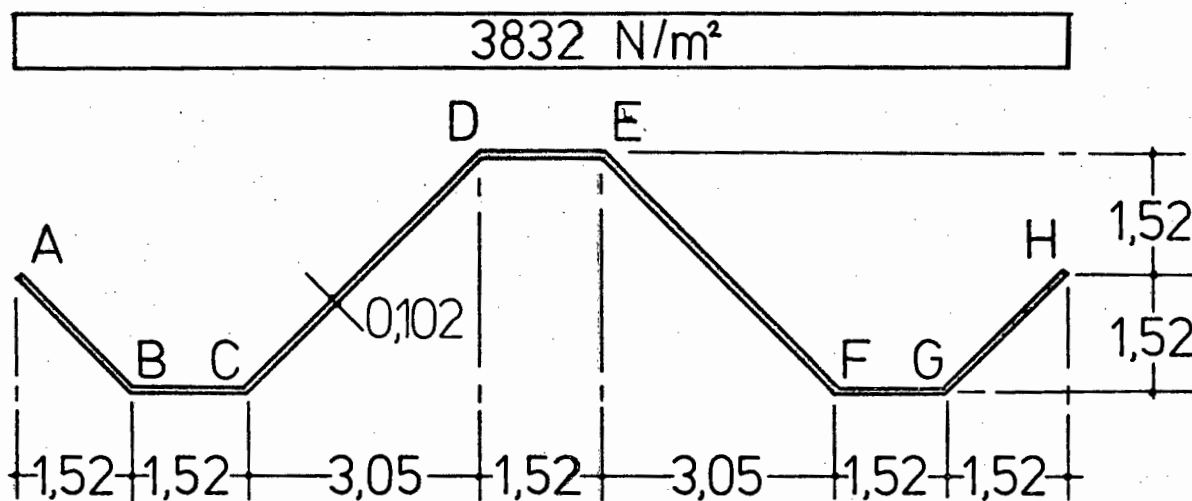
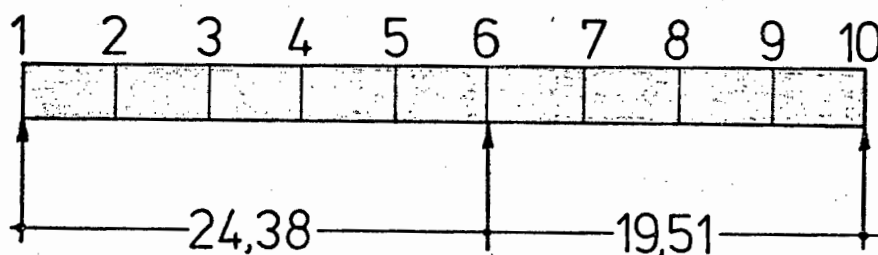


FIG.3.10

EXAMPLE NO. 2:

This example is an open cell, thin walled continuous structure. The ability of the theory and program to cater for an interior support is illustrated. Longitudinal stresses and transverse bending moments are compared with other published work.

Reference	: F.W. Beaufait; Analysis of Continuous Folded Plate Surface, A.S.C.E., Dec. 1965. A.C. Scordelis and K. Lo; Analysis of Continuous Folded Plate Surface, A.S.C.E., June 1966.
Cross-section:	: As shown in Fig. 3.11.
Span	: 24,38 m and 19,507.m.
Boundary conditions	: Simply supported - Simply supported with interior support.
Loading	: Uniformly distributed load of 3832 N/m^2 .
Young's modulus (X and Y)	: $2,52443 \text{ E } 10 \text{ N/m}^2$.
Poisson's ratio (X and Y)	: 0,00.
Program C.P.U. time	: 2 min 21 sec.

Cross-sectionFIG.3.11ElevationFIG.3.12

Section AB has 2 strips

Section BC has 2 strips

Section CD has 4 strips

Section D to centre line has 2 strips

Only half the cross-section was analysed, the rotations and lateral displacements at the node on the centre line were zeroed.

Longitudinal distribution of transverse moment on fold line C

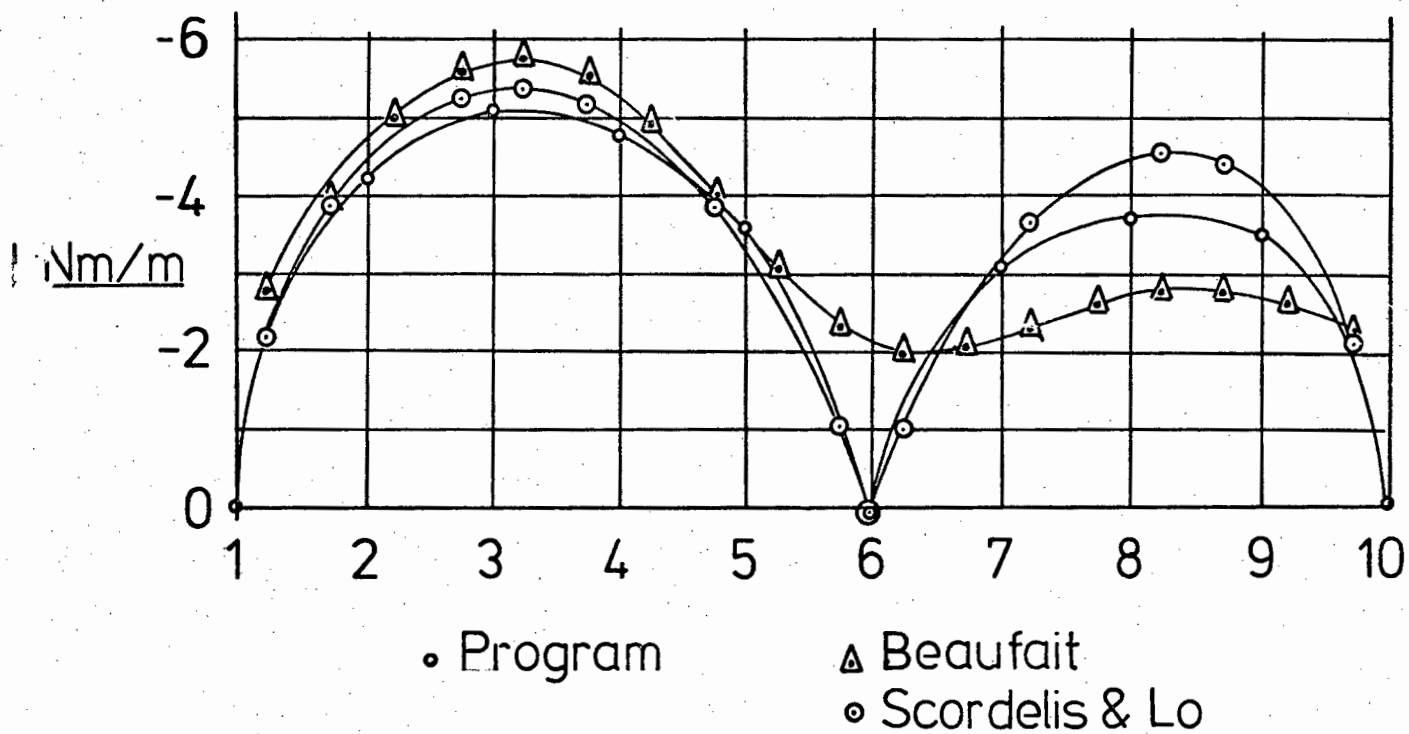


FIG.3.13.

Transverse bending moment at section 3

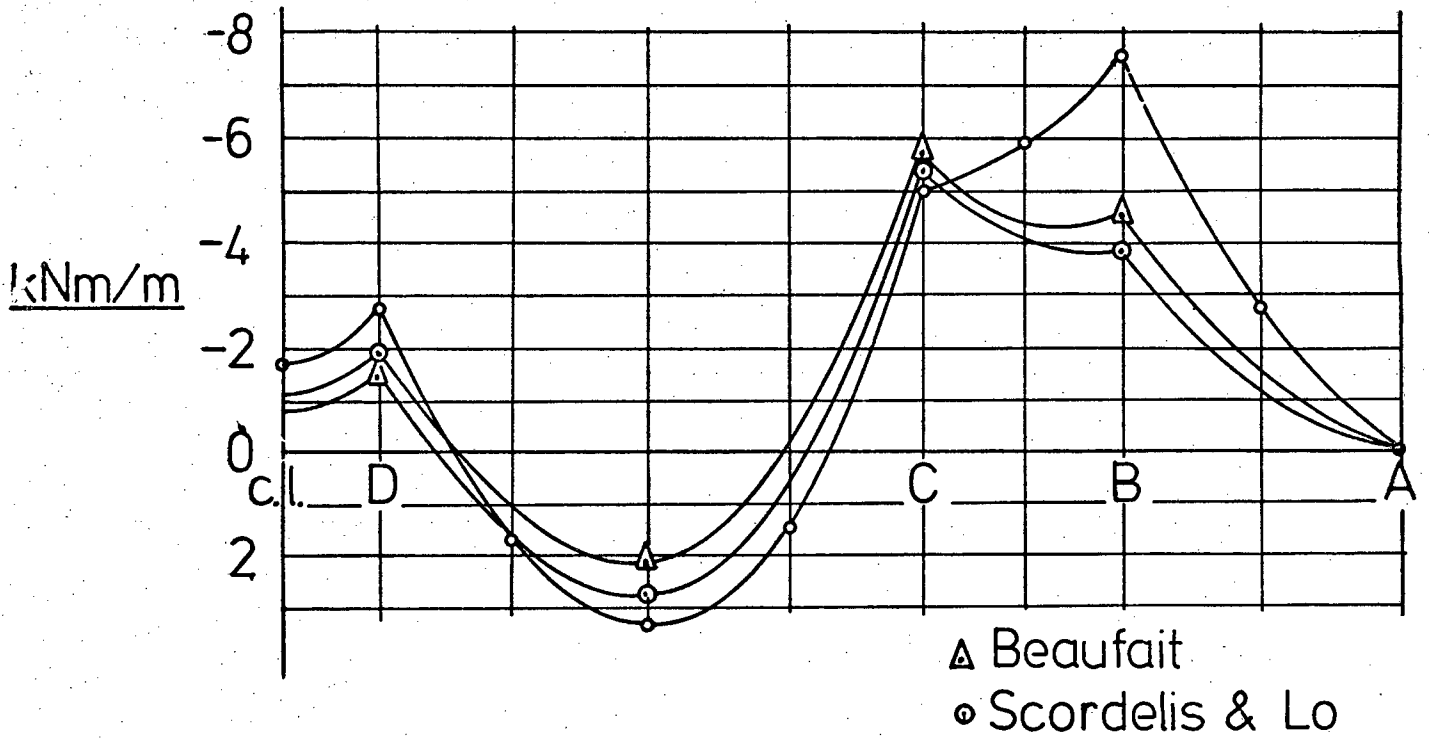


FIG 3.14.

σ_x on foldline D

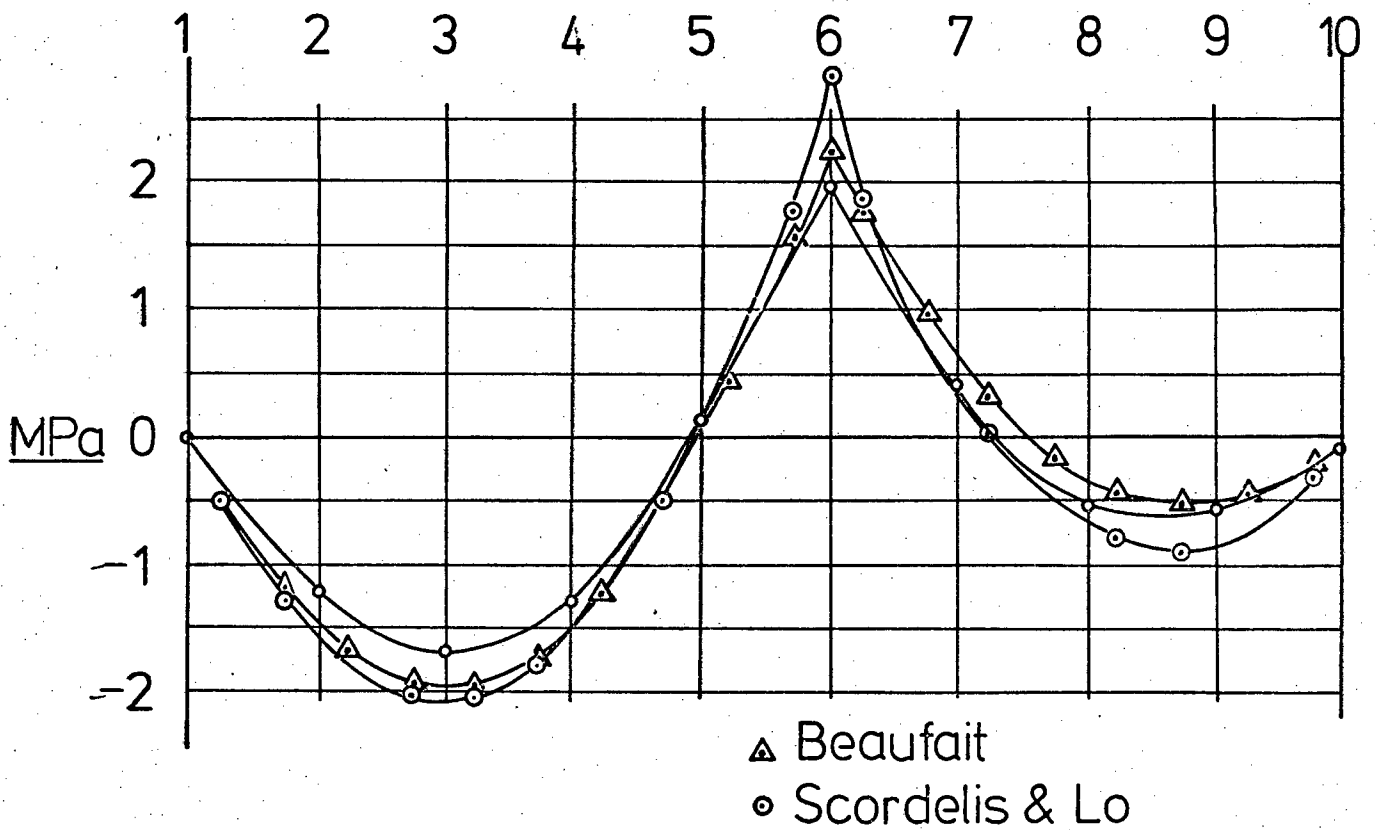
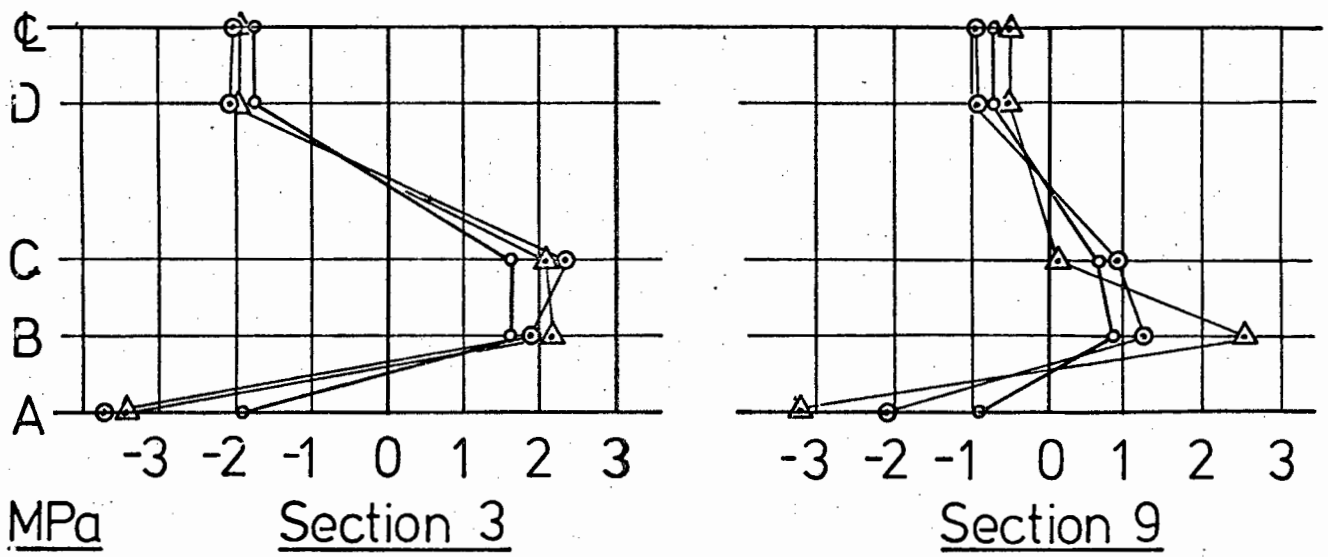


FIG 3.15.

Transverse distribution of longitudinal stress



- ▲ Beaufait
- Scordelis & Lo

FIG.3.16

EXAMPLE NO. 3:

The structure shown is an open cell, thin walled translational shell which has members of two different thicknesses.

Reference : A. de Fries-Skene and A.C. Scordelis; Direct Stiffness Solution for Folded Plates, A.S.C.E., August, 1964.
 Y.K. Cheung; Folded Plate Structures by Finite Strip Method, A.S.C.E., Dec. 1969.

Cross-section : As shown in Fig. 3.

Span : 30,480 m

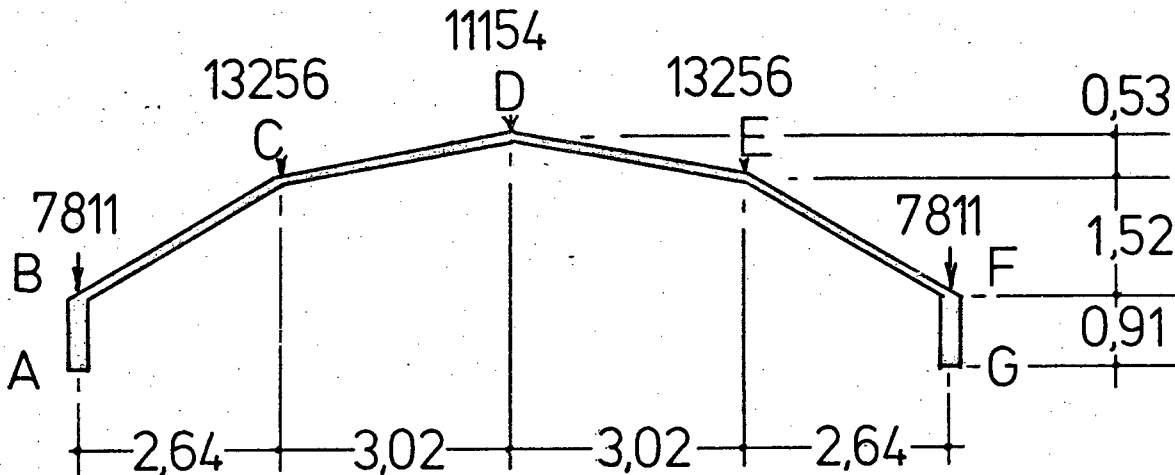
Boundary conditions : Simply supported - Simply supported.

Loading : Ridge Line loading as shown in Fig. 3.17.

Young's modulus (X and Y) : $13,795 \times 10^9 \text{ N/m}^2$.

Poisson's ratio (X and Y) : 0,00.

Program C.P.U. time : 2 min 38 sec.



Cross-section

FIG.3.17

All plate elements were divided into 2 strips each. The C.P.U. time given is for the analysis of the complete structure. The program, however, is quite capable of analysing one quarter of such "two way symmetrical" structures. A time of 1 min 29 sec was recorded for such an analysis.

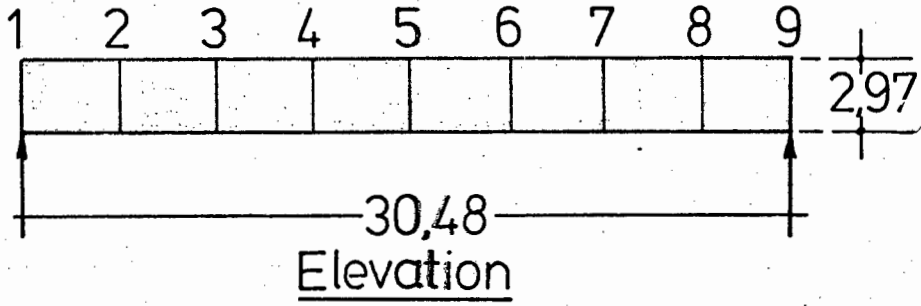
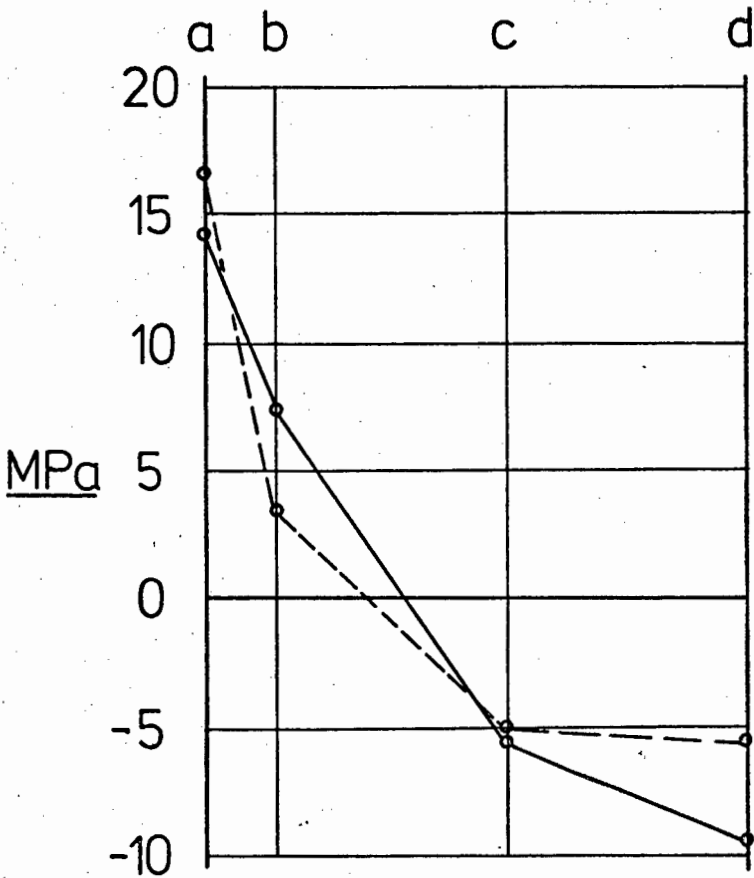


FIG.3.18

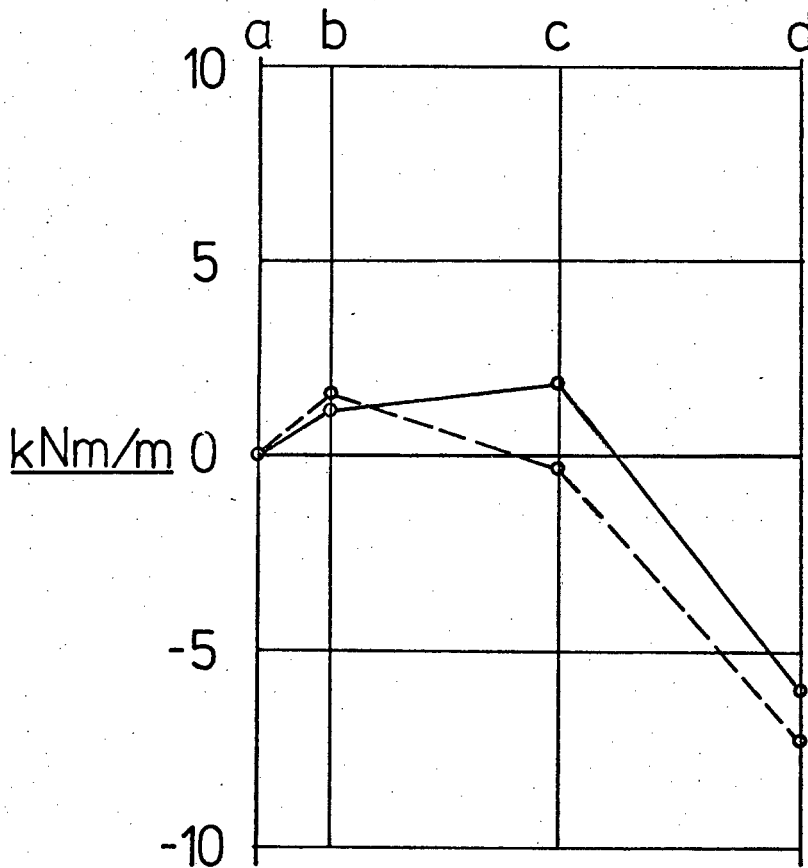
σ_x at midspan



dotted: Elasticity theory
Ref: 16

FIG. 3.19

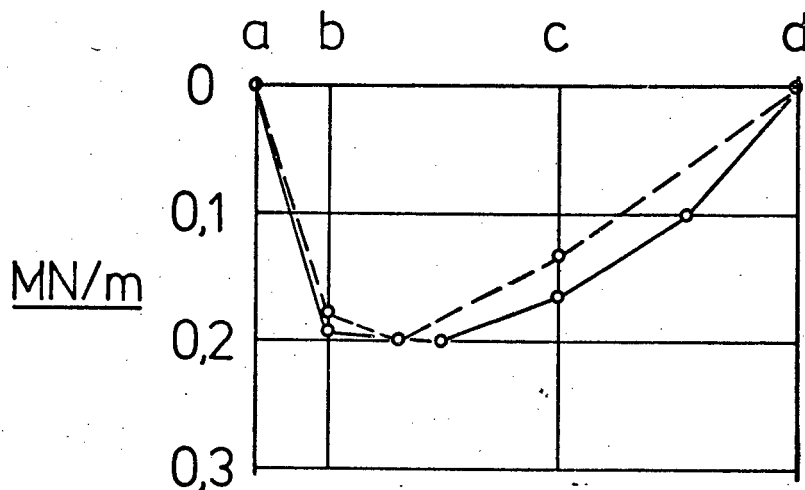
Transverse moment at midspan



dotted: Ref. 16

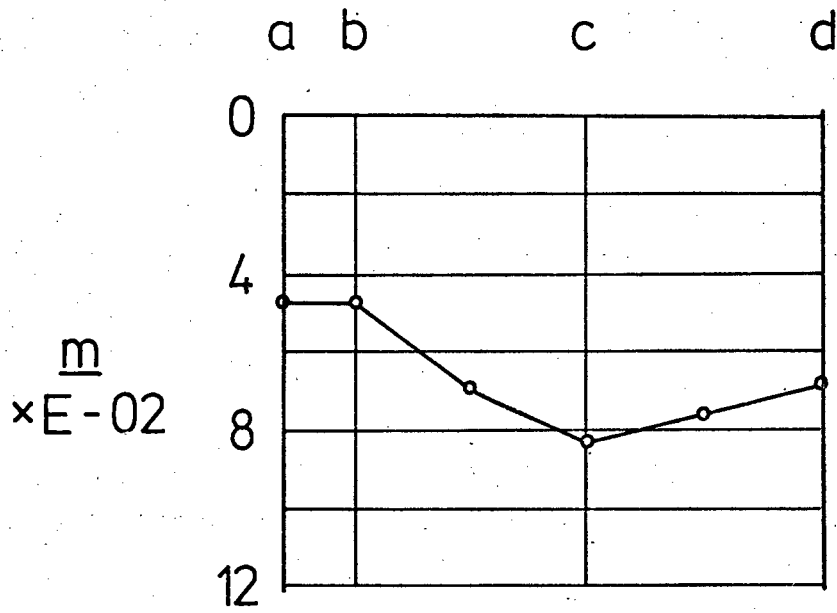
FIG. 3.20

Plate membrane shear at support



dotted: Ref. 16

FIG. 3.21

Vertical deflection at midspanFIG.3.22

3.2 Discussion of Results

In example 1, the box beam webs and flanges were kept relatively thick so as to reduce the effect of secondary stresses and so make the comparison with simple beam theory realistic.

In all cases the deflection and bending stress curves were within acceptable limits. Fig. 3.4 shows that the method of analysis is capable of detecting shear lag, which is important in the analysis of thin walled box structures. Fig. 3.7 shows shear stresses in the beam near the supports. The computer analysis overestimates these stresses at the ends of the structure and it was found that extrapolating the results for nodes near the ends gives acceptable answers. Fig. 3.10 shows the form of the in-plane u displacement for the propped cantilever case. It can be seen that the slope of this displacement with respect to the longitudinal axis is zero at the right hand support, fulfilling the requirement of zero stress at the boundary ($\sigma_x \neq d_x \partial u / \partial x$).

Example 2 was compared with published work. It shows that the method is quite capable of dealing with internal supports and gives reasonably accurate results. The values, as plotted, indicate that there is a fairly large range between results obtained by Beaufait and those obtained by Scordelis and Lo.

Fewer nodes were used in the longitudinal direction than in the two references and thus it was not possible to compare results at exactly the same cross-sections. The number of cross-sections was limited by practical considerations and the maximum array size handled by the computer.

Fig. 3.13 shows that Beaufait's solution has a relatively high transverse moment over the internal support. (Cross-section 6). The exact nature of the support is not described in the literature, but Scordelis gives a very small bending moment at this section, indicating a diaphragm type support. The present analysis also makes use of a diaphragm type support, and is capable of simulating any type of internal fixity.

Fig. 3.14 shows the transverse bending moment at Section 3. The results agree reasonably well except at fold line B, where the bending moment is substantially more but still within acceptable design limits.

Example 3 shows essentially the same variables as example 2. The accuracy of the results, compared with published work, is also approximately the same.

CONCLUSIONS

This chapter contains general conclusions in respect of the method of analysis and its applications. The main points of interest which arose with the application of finite differences are discussed, as well as computer implications and advantages over the more usual Fourier Series Method are listed.

4.1 Finite Difference Operator Patterns

The patterns employed in the computer program are given in Appendix D. The following facts governed the choice of patterns and the resulting accuracy.

- (a) The half-band width of the stiffness matrix is three times the number of degrees of freedom per cross-section. This is because five point central difference operator patterns were used for the fourth derivative, of which only the central and right hand values were actually used in the analysis. Thus these three values, which determine the stiffness matrix band-width, are the minimum required for the fourth derivative.
- (b) The various strip stiffness matrices and their multipliers are as follows:-

<u>Matrix</u>	<u>Symmetry</u>	<u>Multiplier</u>	<u>Symmetry</u>
α	Symmetrical	D'''	Symmetrical
γ	Symmetrical	D''	Symmetrical
ϵ	Skew-Symmetrical	D'	Skew-Symmetrical
ζ	Symmetrical	D	Symmetrical

It is noticed that every matrix that is symmetrical must be multiplied by a symmetrical multiplier and vice-versa. This dictates that no odd pattern may be inserted at the end of the structure to increase accuracy, as it would not be compatible with the rest of the multiplier matrix. The maximum number of values that are actually used in these multiplier matrices are limited to three, as more would give values outside the already rather broad band-width.

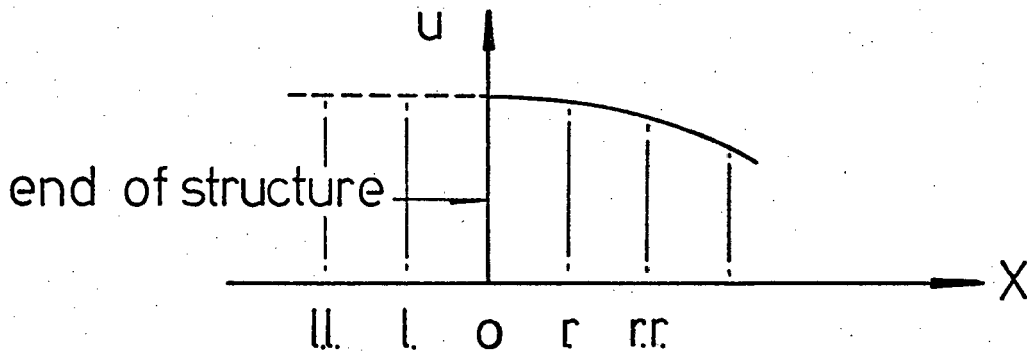
- (c) The values in the scalar matrix (D) may only be different from 1.0 at the extreme ends of the structure. This is because boundary conditions need only be satisfied at the ends of the structure.
- (d) The in-plane u displacement boundary condition is satisfied by using the fact that the longitudinal stress is zero at the free edge. The longitudinal stress is given by:

$$\sigma_x = d_x \frac{\partial u}{\partial x} + d_{xy} \frac{\partial v}{\partial y}.$$

Since the second term makes only a small contribution, and none at all when $v = 0$, the stress may be approximated by:

$$\sigma_x \doteq d_x \frac{\partial u}{\partial x}.$$

At the free edge, $\sigma_x = 0$, which implies that $\partial u / \partial x$ must be zero. This is satisfied by the following displacement model;



- (e) The final stresses are calculated from the displacements obtained, and it was found that small differences in corresponding displacements resulted in unacceptably large differences in stresses. Thus it was absolutely necessary to use operator patterns which gave very good symmetrical displacements.
- (f) External reactions and other forces not given in the computer analysis can be readily calculated using the correct finite difference operator pattern together with the displacements given.

4.2 Computer Program Implications

- (a) The computer program, as written, uses a large amount of core storage because of the necessarily broad band width of the stiffness matrix. However, the solution time compares very favourably with those given in the references listed. (This may be because the computer used, UNIVAC 1106, is a relatively high speed machine).
- (b) A small amount of core storage is required for the CA, CG, CE and CZ matrices, and the finite difference values are stored as efficiently as possible. This method can hence be used on a fairly small core machine if the formation of the stiffness matrix is altered so that blocks of figures are transferred to disc storage and are later recalled as required.
- (c) Tapering members can be given an equivalent thickness which corresponds to an identical second moment of area.

4.3 Advantages over Fourier Series Method

The method of using finite differences together with the normal finite strip concept has distinct advantages over the use of Fourier Series. These are:

- (a) The global displacements are solved directly, as compared with the Fourier Series Method where substantially smaller calculations are done repeatedly and added, giving the required displacements. Up to seventy, and more, harmonics are used in the papers referred to.
- (b) Point loadings are distributed over one nodal length, this will give more accurate answers than using a number of Fourier Series harmonics.
- (c) Different boundary conditions are employed without any difficulty and are relatively easily understood.
- (d) Interior supports along the length of a structure may be as numerous as desired. Fourier Series becomes impractical for more than three spans.
- (e) This method may also be used for strips of non-uniform thickness in the longitudinal direction, i.e. cut-outs and the change of thickness

of members over supports. This simply leads to the necessity of calculating and storing CA, CG, CE and CZ matrices for as many different cross-sections as are required.

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APPENDIX A:

STIFFNESS SUB-MATRICES (In-Plane Forces)

$$[K] = \begin{bmatrix} G \frac{h}{b} & 0 & -G \frac{h}{b} & 0 \\ 0 & d_y \frac{h}{b} & 0 & -d_y \frac{h}{b} \\ -G \frac{h}{b} & 0 & G \frac{h}{b} & 0 \\ 0 & -d_y \frac{h}{b} & 0 & d_y \frac{h}{b} \end{bmatrix}$$

$$[L] = \begin{bmatrix} d_x \frac{bh}{3} & 0 & d_x \frac{bh}{6} & 0 \\ 0 & G \frac{bh}{3} & 0 & G \frac{bh}{6} \\ d_x \frac{bh}{6} & 0 & d_x \frac{bh}{3} & 0 \\ 0 & G \frac{bh}{6} & 0 & G \frac{bh}{3} \end{bmatrix}$$

$$[M] = \begin{bmatrix} 0 & -G \frac{h}{2} & 0 & -G \frac{h}{2} \\ -d_{xy} \frac{h}{2} & 0 & -d_{xy} \frac{h}{2} & 0 \\ 0 & G \frac{h}{2} & 0 & G \frac{h}{2} \\ d_{xy} \frac{h}{2} & 0 & d_{xy} \frac{h}{2} & 0 \end{bmatrix}$$

APPENDIX B:

STIFFNESS SUB-MATRICES (Bending Forces)

$$[A] = \begin{bmatrix} \frac{13b D}{35} & \frac{11b^2 D}{210} & \frac{9b D}{70} & -\frac{13b^2 D}{420} \\ \frac{11b^2 D}{210} & \frac{b^3 D}{105} & \frac{13b^2 D}{420} & -\frac{b^3 D}{140} \\ \frac{9b D}{70} & \frac{13b^2 D}{420} & \frac{13b D}{35} & -\frac{11b^2 D}{210} \\ -\frac{13b^2 D}{420} & -\frac{b^3 D}{140} & -\frac{11b^2 D}{210} & \frac{b^3 D}{105} \end{bmatrix}$$

$$[B] = \begin{bmatrix} -\frac{6D}{5b} & -\frac{11D}{10} & \frac{6D}{5b} & -\frac{D}{10} \\ -\frac{D}{10} & -\frac{2b D}{15} & \frac{D}{10} & \frac{b D}{30} \\ \frac{6D}{5b} & \frac{D}{10} & -\frac{6D}{5b} & \frac{11D}{10} \\ -\frac{D}{10} & \frac{b D}{30} & \frac{D}{10} & -\frac{2b D}{15} \end{bmatrix}$$

$$[c] = \begin{bmatrix} \frac{24D}{5b} & \frac{4D}{10} & -\frac{24D}{5b} & \frac{4D}{10} \\ \frac{4D}{10} & \frac{8bD}{15} & -\frac{4D}{10} & -\frac{4bD}{30} \\ -\frac{24D}{5b} & -\frac{4D}{10} & \frac{24D}{5b} & \frac{4D}{10} \\ \frac{4D}{10} & -\frac{4bD}{30} & -\frac{4D}{10} & \frac{8bD}{15} \end{bmatrix}$$

$$[D] = \begin{bmatrix} \frac{12D}{b^3} & \frac{6D}{b^2} & -\frac{12D}{b^3} & \frac{6D}{b^2} \\ \frac{6D}{b^2} & \frac{4D}{b} & -\frac{6D}{b^2} & \frac{2D}{b} \\ -\frac{12D}{b^3} & -\frac{6D}{b^2} & \frac{12D}{b^3} & -\frac{6D}{b^2} \\ \frac{6D}{b^2} & \frac{2D}{b} & -\frac{6D}{b^2} & \frac{4D}{b} \end{bmatrix}$$

$$[\lambda] = \begin{bmatrix} N \\ N/m \\ N/m \\ Nm/m \end{bmatrix} \quad \begin{array}{l} u \quad (\text{Force}) \\ v \quad (\text{Force per unit length}) \\ w \quad (\text{Force per unit length}) \\ \theta \quad (\text{Moment per unit length}) \end{array}$$

APPENDIX C: α , γ , ϵ and ζ MATRICESTHE α MATRIX

$$\begin{array}{cccc|cccc}
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & \frac{13b D}{35} x & \frac{11b^2 D}{210} x & 0 & 0 & \frac{9b D}{70} x - \frac{13b^2 D}{420} x & \\
 0 & 0 & \frac{11b^2 D}{210} x & \frac{b^3 D}{105} x & 0 & 0 & \frac{13b^2 D}{420} x - \frac{b^3 D}{140} x & \\
 \hline
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & \frac{9b D}{70} x & \frac{13b^2 D}{420} x & 0 & 0 & \frac{13b D}{35} x - \frac{11b^2 D}{210} x & \\
 0 & 0 & -\frac{13b^2 D}{420} x & -\frac{b^3 D}{140} x & 0 & 0 & -\frac{11b^2 D}{210} x & \frac{b^3 D}{105} x
 \end{array}$$

THE Y MATRIX

$-\frac{hd_x b}{3}$	0	0	$-\frac{hd_x b}{6}$	0	0	0
0	$-\frac{hg b}{3}$	0	0	$-\frac{hg b}{6}$	0	0
0	0	$-\frac{6D_1}{5b} - \frac{24D_{xy}}{5b} - \frac{6D_1}{5b}$	0	$-\frac{11D_1}{10} - \frac{4D_{xy}}{10}$	$-\frac{6D_1}{5b} + \frac{24D_{xy}}{5b} + \frac{6D_1}{5b}$	$-\frac{D_1}{10} - \frac{4D_{xy}}{10}$
0	0	$-\frac{11D_1}{10} - \frac{2bD_1}{15}$	0	$-\frac{2bD_1}{15} - \frac{8bD_{xy}}{15}$	$-\frac{D_1}{10} + \frac{4D_{xy}}{10}$	$-\frac{bD_1}{30} + \frac{4bD_{xy}}{30}$
$-\frac{hd_x b}{6}$	0	0	$-\frac{hd_x b}{3}$	0	0	0
0	$-\frac{hg b}{6}$	0	0	$-\frac{hg b}{3}$	0	0
0	0	$-\frac{6D_1}{5b} + \frac{24D_{xy}}{5b} + \frac{6D_1}{5b}$	0	$-\frac{D_1}{10} + \frac{4D_{xy}}{10}$	$-\frac{6D_1}{5b} - \frac{24D_{xy}}{5b} - \frac{6D_1}{5b}$	$-\frac{11D_1}{10} + \frac{4D_{xy}}{10}$
0	0	$-\frac{D_1}{10} - \frac{4D_{xy}}{10}$	0	$-\frac{2bD_1}{15} - \frac{8bD_{xy}}{15}$	$-\frac{D_1}{10} + \frac{4D_{xy}}{10}$	$-\frac{bD_1}{30} + \frac{4bD_{xy}}{30}$

THE C MATRIX

$\frac{Gh}{b}$	0	0	0	$-\frac{Gh}{b}$	0	0	0
0	$\frac{d h}{b}$	0	0	0	$-\frac{d h}{b}$	0	0
0	0	$\frac{12D}{b^3}y$	$\frac{6D}{b^2}y$	0	0	$-\frac{12D}{b^3}y$	$\frac{6D}{b^2}y$
0	0	$\frac{6D}{b^2}y$	$\frac{4D}{b}y$	0	0	$-\frac{6D}{b^2}y$	$\frac{2D}{b}y$
$-\frac{Gh}{b}$	0	0	0	$\frac{Gh}{b}$	0	0	0
0	$-\frac{d h}{b}$	0	0	0	$\frac{d h}{b}$	0	0
0	0	$-\frac{12D}{b^3}y$	$-\frac{6D}{b^2}y$	0	0	$\frac{12D}{b^3}y$	$-\frac{6D}{b^2}y$
0	0	$\frac{6D}{b^2}y$	$\frac{2D}{b}y$	0	0	$-\frac{6D}{b^2}y$	$\frac{4D}{b}y$

APPENDIX D:FINITE DIFFERENCE OPERATOR PATTERNS

- Note: 1. SA is the spacing between cross-section nodes.
2. Central difference formulae are used.

$$\text{i.e. } \frac{dy}{dx} = \frac{1}{2SA} [-1 y_L + 0 y_o + 1 y_R]$$

$$\frac{d^2y}{dx^2} = \frac{1}{SA^2} [1 y_L - 2 y_o + 1 y_R]$$

$$\frac{d^4y}{dx^4} = \frac{1}{SA^4} [1 y_{LL} - 4 y_L + 6 y_o - 4 y_R + 1 y_{RR}]$$

4th DERIVATIVE ($\div SA^4$)Simply Supported Ends

$$\begin{array}{ccccccc} \textcircled{0} & 0 & 0 & & & & \\ 0 & \textcircled{5} & -4 & 1 & & & \\ 0 & -4 & \textcircled{6} & -4 & 1 & & \\ & 1 & -4 & \textcircled{6} & -4 & 1 & \\ & & & 1 & -4 & \textcircled{6} & -4 & 1 \\ & & & & 1 & -4 & \textcircled{6} & -4 & 0 \\ & & & & & 1 & -4 & \textcircled{5} & 0 \\ & & & & & & 0 & 0 & \textcircled{0} \end{array}$$

Fixed Ends

$$\begin{array}{ccccccc} \textcircled{0} & 0 & 0 & & & & \\ 0 & \textcircled{7} & -4 & 1 & & & \\ 0 & -4 & \textcircled{6} & -4 & 1 & & \\ & 1 & -4 & \textcircled{6} & -4 & 1 & \\ & & & 1 & -4 & \textcircled{6} & -4 & 1 \\ & & & & 1 & -4 & \textcircled{6} & -4 & 0 \\ & & & & & 1 & -4 & \textcircled{7} & 0 \\ & & & & & & 0 & 0 & \textcircled{0} \end{array}$$

Guided Ends

$$\begin{array}{ccccccc} \textcircled{3} & -4 & 1 & & & & \\ -4 & \textcircled{7} & -4 & 1 & & & \\ 1 & -4 & \textcircled{6} & -4 & 1 & & \\ & 1 & -4 & \textcircled{6} & -4 & 1 & \\ & & & 1 & -4 & \textcircled{6} & -4 & 1 \\ & & & & 1 & -4 & \textcircled{6} & -4 & 1 \\ & & & & & 1 & -4 & \textcircled{7} & -4 \\ & & & & & & 1 & -4 & \textcircled{3} \end{array}$$

2nd DERIVATIVE ($\div SA^2$)Simply Supported Ends

$$\begin{pmatrix} 0 & 0 \\ 0 & -2 & 1 \\ & 1 & -2 & 1 \end{pmatrix}$$

Fixed Ends

$$\begin{pmatrix} 0 & 0 \\ 0 & -2 & 1 \\ & 1 & -2 & 1 \end{pmatrix}$$

$$\begin{pmatrix} & & 1 & -2 & 1 \\ & & & 1 & -2 & 0 \\ & & & & 0 & 0 \end{pmatrix}$$

$$\begin{pmatrix} & & 1 & -2 & 1 \\ & & & 1 & -2 & 0 \\ & & & & 0 & 0 \end{pmatrix}$$

Guided Ends

$$\begin{pmatrix} -1 & 1 \\ 1 & -2 & 1 \\ & 1 & -2 & 1 \end{pmatrix}$$

Function u: Fixed Ends

$$\begin{pmatrix} 0 & 0 \\ 0 & -2 & 1 \\ & 1 & -2 & 1 \end{pmatrix}$$

$$\begin{pmatrix} & & 1 & -2 & 1 \\ & & & 1 & -2 & 1 \\ & & & & 1 & -1 \end{pmatrix}$$

$$\begin{pmatrix} & & 1 & -2 & 1 \\ & & & 1 & -2 & 0 \\ & & & & 0 & 0 \end{pmatrix}$$

Function u: Free Ends:

$$\begin{pmatrix} -1 & 1 \\ 1 & -2 & 1 \\ & 1 & -2 & 1 \end{pmatrix}$$

$$\begin{pmatrix} & & 1 & -2 & 1 \\ & & & 1 & -2 & 1 \\ & & & & 1 & -1 \end{pmatrix}$$

1st DERIVATIVE ($\div 2$ SA)Simply Supported Ends

$$\begin{array}{ccc} \textcircled{0} & 1 & \\ -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \end{array}$$

$$\begin{array}{ccc} -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \\ & & -1 & \textcircled{0} \end{array}$$

$$\begin{array}{ccc} -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 0 \\ & & 0 & \textcircled{0} \end{array}$$

Guided Ends

$$\begin{array}{ccc} \textcircled{0} & 0 & \\ -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \end{array}$$

$$\begin{array}{ccc} 1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \\ & & 0 & \textcircled{0} \end{array}$$

$$\begin{array}{ccc} -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 0 \\ & & 0 & \textcircled{0} \end{array}$$

Fixed Ends

$$\begin{array}{ccc} \textcircled{0} & 0 & \\ 0 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \end{array}$$

Function u: Fixed Ends

$$\begin{array}{ccc} \textcircled{0} & 0 & \\ 0 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \end{array}$$

Function u: Free Ends

$$\begin{array}{ccc} \textcircled{-1} & 1 & \\ -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \end{array}$$

$$\begin{array}{ccc} -1 & \textcircled{0} & 1 \\ & -1 & \textcircled{0} & 1 \\ & & -1 & \textcircled{1} \end{array}$$

The following Finite Difference Operator patterns are used in Sub-routine ECHO to determine stresses from deflections.

2nd DERIVATIVE ($\div SA^2$)

		(2)	-5	4	-1	Forward Difference
	1	(-2)	1			Central Difference
-1	4	-5	(2)			Backward Difference

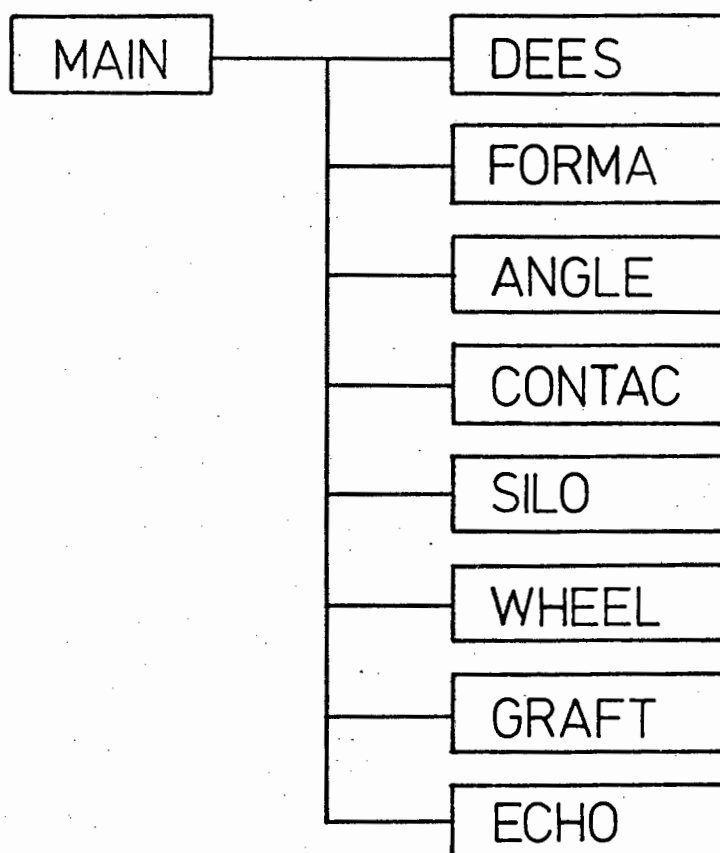
1st DERIVATIVE ($\div 2 SA$)

		(-3)	4	-1	Forward Difference
	-1	(0)	1		Central Difference
1	-4	(3)			Backward Difference

FORTRAN V COMPUTER PROGRAM

MENAI*

*The strait over which Robert Stephenson built the Britannia Tubular Bridge in 1850.

MENAI SUBROUTINE SEQUENCE

PROJECT=MENAI(1).MATN

```

1      COMPILER(XM=3)
2      C      *****
3      C      *MENAI* PROGRAM TO SOLVE TRANSLATIONAL SHELL STRUCTURES
4      C      THE FINITE STRIP METHOD OF ANALYSIS IS USED
5      C      CUBIC POLYNOMIAL DISPLACEMENT FUNCTIONS ARE USED ACROSS THE STRIPS
6      C      FOR THE NORMAL DISPLACEMENTS, AND THE IN-PLANE DISPLACEMENTS ARE
7      C      VARIED LINEARLY;
8      C      FINITE DIFFERENCE APPROXIMATIONS TO THE DIFFERENTIAL EQUATIONS
9      C      ARE USED ALONG THE LENGTH OF THE STRIPS * UNITS ARE ALWAYS N,M,S.*
10     C      *****
11     COMMON/EXT/SKU(440,133),SD4CA(440,5),SD2CG(440,3),SDOZE(440),
12     .SD1CE(440,3),CA(44,44),CG(44,44),CE(44,44),
13     .CZ(44,44),SIGMAX(180),SIGMAY(180),TAUXY(180),
14     .AMX(180),AHY(180),AMXY(180)
15     COMMON DDY(30),DDY(30),DD1(30),DDXY(30),D4(30),D3(30)
16     COMMON FA(8,8,30),FG(8,8,30),FE(8,8,30),FZ(8,8,30)
17     COMMON AFA(8,8,30),AFG(8,8,30),AFE(8,8,30),AFZ(8,8,30)
18     COMMON SD4(6,5,6),SD2(6,3,6),SD1(6,3,6),SD(6,6)
19     COMMON DEX,DEY,DPX,DPY,DG,IGNUM,DLDX,DLXY,DLZY
20     COMMON MHC,MSKD,MSKB,MS3IC,MSZERO(250)
21     COMMON NW27,JENOX,JENUM,MS(31),ME(31)
22     COMMON ANG(30),SA
23     COMMON /B2/ATRANS
24     DOUBLE PRECISION ATRANS(8,8,30)
25     MHC=44
26     MSKD=440
27     MSKB=132
28     NW27=133
29     PARAMETER JE1=720
30     DIMENSION EDV(JE1)
31     PRINT 541
32     541  FORMAT(1H1,6(/),1HD,3DX,6C('**'),/,
33     .31X,'**',13X,'MENAI: TRANSLATIONAL SHELL PROGRAM',11X,'**',/,
34     .31X,'**',58X,'**',/,
35     .31X,'**',3X,'DATE:.....',28X,'**',/,
36     .31X,'**',58X,'**',/,
37     .31X,'**',3X,'PROJECT:.....',27X,'**',/,
38     .31X,'**',58X,'**',/,
39     .31X,'**',3X,'CLIENT:.....',28X,'**')
40     PRINT 567
41     567  FORMAT(1H ,3DX,'**',58X,'**',/,
42     .31X,'**',3X,'GENERAL DESCRIPTION OF STRUCTURE:.....
43     .',1X,'**',/,
44     .31X,'**',58X,'**',/,
45     .31X,'**',3X,52('.'),3X,'**',/,
46     .31X,'**',58X,'**')
47     PRINT 566
48     566  FORMAT(1H ,3DX,'**',3X,'LOADING CASE CONSIDERED',3C('.'),2X,'**',/,
49     .31X,'**',58X,'**',/,
50     .31X,6C('**'))
51     C
52     C      EVALUATION OF ELASTIC CONSTANTS
53     C
54     CALL DEES
55     C
56     PRINT 542
57     542  FORMAT(1H1,'**ALL UNITS USED ARE N.M.S.**')
58     WRITE(5,SD1) DEX,DEY,DPX,DPY,DG
59     501  FORMAT(1HD,'YOUNGS MODULUS X:',8X,1P216.6,/,

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60      . * YOUNG'S MODULUS  Y: ',8X,E16.6, /,
61      . * POISSON RATIO   X: ',12X,0F6.3, /,
62      . * POISSON RATIO   Y: ',12X,F6.3, /,
63      . * SHEAR MODULUS   G: ',8X,1PE16.E, /, )
64      C
65      C   FORMULATION OF THE BASIC STIFFNESS MATRICES
66      C
67      C   CALL FORMA
68      C
69      C   TO COMPILE THE TRANSFORMATION MATRICES FOR INDIVIDUAL STRIPS
70      C   AND TO COMPILE GLOBAL STRIP STIFFNESS MATRICES
71      C
72      C   CALL ANGLE
73      C
74      C   PRINT 543
75      543  FORMAT(1HC, 'STRIP NO      STRIP WIDTH      STRIP THICKNESS      STRIP
76      . * ANGLE(RAD) ', /,
77      . * 62(' - ') )
78      DO 544 I=1, IDNUM
79      544  WRITE(5,545) I, DB(I), DH(I), ANG(I)
80      545  FORMAT(1HC, 4X, I2, 11X, F6.3, 10X, F6.3, 9X, F10.7)
81      C
82      C   TO ESTABLISH THE COMBINED STIFFNESS MATRICES FOR THE TOTAL
83      C   CROSS-SECTION
84      C
85      C   CALL CONTAC
86      C
87      C   TO COMPILE THE FINITE DIFFERENCE EQUATIONS, MULTIPLY WITH CA, CG,
88      C   CE, CZ MATRICES AND COMBINE TO FORM UPPER TRIANGULAR MATRIX SKU
89      C
90      C   CALL SILO
91      C
92      IF(MS(1).EQ.1) PRINT 570
93      IF(MS(1).EQ.3) PRINT 571
94      IF(MS(1).EQ.5) PRINT 572
95      IF(MS(1).EQ.7) PRINT 573
96      570  FORMAT(1HC, 'BOUNDARY CONDITION AT START IS FIXED')
97      571  FORMAT(1HC, 'BOUNDARY CONDITION AT START IS GUIDED')
98      572  FORMAT(1HC, 'BOUNDARY CONDITION AT START IS FREE')
99      573  FORMAT(1HC, 'BOUNDARY CONDITION AT START IS SIMPLY SUPPORTED')
100     IF(ME(1).EQ.1) PRINT 575
101     IF(ME(1).EQ.3) PRINT 576
102     IF(ME(1).EQ.5) PRINT 577
103     IF(ME(1).EQ.7) PRINT 578
104     575  FORMAT(1HC, 'BOUNDARY CONDITION AT END IS FIXED')
105     576  FORMAT(1HC, 'BOUNDARY CONDITION AT END IS GUIDED')
106     577  FORMAT(1HC, 'BOUNDARY CONDITION AT END IS FREE')
107     578  FORMAT(1HC, 'BOUNDARY CONDITION AT END IS SIMPLY SUPPORTED')
108     WRITE(5,580) SA
109     580  FORMAT(1HC, 'NODAL SPACING = ', F6.3)
110     C   FORMULATION OF LOAD MATRIX
111     C
112     C   CALL WHEEL
113     C
114     C   TO SOLVE DISPLACEMENTS
115     C
116     C   CALL GRAFT
117     C
118     C   PRINT 565
119     565  FORMAT(1HC, 'CROSS-SECTION ', 6X, 'NODE ', 29X, 'GLOBAL DISPLACEMENTS

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120      .*)
121      PRINT 546
122      546  FORMAT(1HC,35X,'U',18X,'V',14X,'W',13X,'THETA(RAD)')
123      PRINT 548
124      569  FORMAT(1H ,96('-''))
125      C    NUMBER OF CROSS-SECTIONS
126      KA10=3*MSKD/MSKB
127      C    NUMBER OF NODES PER SECTION
128      KA11=MSKB/12
129      C    DEGREES OF FREEDOM PER SECTION
130      KA16=MSKB/3
131      DO 549 K=1,KA10
132      WRITE(5,547) K
133      547  FORMAT(1H ,6X,I3)
134      DO 549 L=1,KA11
135      KA12=(K-1)*KA16+L+4-3
136      KA13=KA12+1
137      KA14=KA13+1
138      KA15=KA14+1
139      WRITE(5,548) L,SKU(KA12,NW27),SKU(KA13,NW27),
140      .SKU(KA14,NW27),SKU(KA15,NW27)
141      548  FORMAT(1H ,20X,I2,7X,1PE12.6,6X,E12.6,6X,E12.6,6X,E12.6)
142      549  CONTINUE
143      C
144      C    TO SOLVE STRESSES IN LOCAL AXES CO-ORDINATES
145      C
146      CALL ECHO (EDV,JE1)
147      C
148      PRINT 550
149      550  FORMAT(1H1,'CROSS-SECTION SUB-STRUCTURE NODE',35X,'LOCAL STRESSE
150      .S')
151      PRINT 551
152      551  FORMAT(1H ,41X,'SIGMA-X',7X,'SIGMA-Y',6X,'TAU-XY',
153      .12X,'M-X',9X,'M-Y',8X,'M-XY')
154      PRINT 569
155      569  FORMAT(1H ,111('-''))
156      DO 555 I=1,JENOX
157      WRITE(5,552) I
158      552  FORMAT(1HC,6X,I2)
159      DO 555 J=1,JENUM
160      WRITE(5,553) J
161      553  FORMAT(1H ,21X,I2)
162      DO 555 K=1,3
163      JE201=(I-1)*3+JENUM+J+3-2+K-1
164      WRITE(5,554) K,SIGMAX(JE201),SIGMAY(JE201),TAUXY(JE201),
165      .AMX(JE201),AMY(JE201),AMXY(JE201)
166      554  FORMAT(1H ,32X,I2,6X,1PE11.5,2X,E11.5,2X,E11.5,
167      .2X,E11.5,2X,E11.5,2X,E11.5)
168      555  CONTINUE
169      STOP
170      END

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PROJECT=MENAI(1).DEES
1      COMPILER(XM=3)
2      SUBROUTINE DEES
3      COMMON/EXT/SKU(440,133),SD4CA(440,5),SD2CG(440,3),SDCZE(440),
4      .SD1CE(440,3),CA(44,44),CG(44,44),CE(44,44),
5      .CZ(44,44),SIGMAX(180),SIGMAY(180),TAUXY(180),
6      .AMX(180),AMY(180),AMXY(180)
7      COMMON DDX(30),DDY(30),DD1(30),DDXY(30),DH(30),DB(30)
8      COMMON FA(8,8,30),FG(8,8,30),FE(8,8,30),FZ(8,8,30)
9      COMMON AFA(8,8,30),AFG(8,8,30),AFE(8,8,30),AFZ(8,8,30)
10     COMMON SD4(6,5,6),SD2(6,3,6),SD1(6,3,6),SD(6,6)
11     COMMON DEX,DEY,DPX,DPY,DG,IDNUM,DLDX,DLDEXY,DLDY
12     COMMON MHC,MSKD,MSKB,MS3ID,MSZERO(250)
13     COMMON VW27,JENOX,JENUM,MS(31),ME(31)
14     COMMON ANG(30),SA
15     C   READ EX , EY , POISSON X , POISSON Y
16     C   READ (8,100) DEX , DEY , DPX , DPY
17     100  FORMAT (
18     C   READ NUMBER OF STRIPS IN STRUCTURE IDNUM
19     C   READ(8,101) IDNUM
20     101  FORMAT(
21     C   READ STRIP THICKNESS AND WIDTH
22     C   DO 102 I = 1, IDNUM
23     102  READ (8,103) DH(I) , DB(I)
24     103  FORMAT(
25     C   IN ORDER OF CALCULATION: SHEAR MODULUS, LITTLE(DX,DY,DY),
26     C   CAPITAL(DX,DY,D1,DXY)
27     C   DG = (DEX+DEY)/(4.*(1.+(DPX+DPY)/2.))
28     C   DLDX = DEX/(1.-DPX+DPY)
29     C   DLDXY = DPX+DEY/(1.-DPX+DPY)
30     C   DLDY = DEY/(1.-DPX+DPY)
31     C   DO 104 I = 1, IDNUM
32     C   DDY(I) = (DLDY+DH(I)**3.)/12.
33     C   DD1(I) = (DLDXY+DH(I)**3.)/12.
34     C   DDXY(I) = (DG+DH(I)**3.)/12.
35     104  CONTINUE
36     C   RETURN
37     C   END
38

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@PRT,S MENAI.FORMA

PROJECT*MENAI(1).FORMA

```

1     COMPILER(XM=3)
2     SUBROUTINE FORMA
3     COMMON/EXT/SKU(440,133),SD4CA(440,5),SD2CG(440,3),SDCZE(440),
4     .SD1CE(440,3),CA(44,44),CG(44,44),CE(44,44),
5     .CZ(44,44),SIGMAX(180),SIGMAY(180),TAUXY(180),
6     .AMX(180),AMY(180),AMXY(180)
7     COMMON DDX(30),DDY(30),DD1(30),DDXY(30),DH(30),DB(30)
8     COMMON FA(8,8,30),FG(8,8,30),FE(8,8,30),FZ(8,8,30)
9     COMMON AFA(8,6,30),AFG(8,8,30),AFE(8,8,30),AFZ(8,8,30)
10    COMMON SD4(6,5,6),SD2(6,3,6),SD1(6,3,6),SD(6,6)
11    COMMON DEX,DEY,DPX,DPY,DG,IDNUM,DLDX,DLDEXY,DLDY
12    COMMON MHC,MSKD,MSKB,MS3ID,MSZ90(250)
13    COMMON NW27,JENOX,JENUM,MS(31),ME(31)
14    COMMON ANG(30),SA
15
16    C
17    C     FORMULATION OF STIFFNESS MATRICES. THE FA,FG,FE,FZ MATRICES ARE
18    C     COMPILED FOR ONE STRIP AT A TIME IN ORDER GIVEN
19    C     DO 1000 I=1,IDNUM
20    C     THE ALPHA MATRIX *****
21    FA(3,3,I) = 13.*DB(I)*DDX(I)/75.
22    FA(4,4,I) = DB(I)**3.*DDX(I)/105.
23    FA(7,7,I) = 13.*DB(I)*DDX(I)/75.
24    FA(8,8,I) = DB(I)**3.*DDX(I)/105.
25    FA(4,7,I) = 11.*DB(I)**2.*DDX(I)/210.
26    FA(3,4,I) = FA(4,3,I)
27    FA(8,7,I) = -11.*DB(I)**2.*DDX(I)/210.
28    FA(7,4,I) = 13.*DB(I)**2.*DDX(I)/420.
29    FA(4,7,I) = FA(7,4,I)
30    FA(7,3,I) = 9.*DB(I)*DDX(I)/70.
31    FA(3,7,I) = FA(7,7,I)
32    FA(8,4,I) = -DB(I)**3.*DDX(I)/140.
33    FA(4,8,I) = FA(8,4,I)
34    FA(8,3,I) = -13.*DB(I)**2.*DDX(I)/420.
35    FA(3,8,I) = FA(8,3,I)
36    C     THE GAMMA MATRIX *****
37    FG(1,1,I) = -DH(I)*DLDX*DB(I)/3.
38    FG(2,2,I) = -DH(I)*DG*DB(I)/3.
39    FG(3,3,I) = -6.*DD1(I)/(5.*DB(I))-24.*DDXY(I)/(5.*DB(I))
40    .-6.*DD1(I)/(5.*DB(I))
41    FG(4,4,I) = -2.*DB(I)*DD1(I)/15.-8.*DB(I)*DDXY(I)/15.
42    .-2.*DB(I)*DD1(I)/15.
43    FG(5,5,I) = FG(1,1,I)
44    FG(6,6,I) = FG(2,2,I)
45    FG(7,7,I) = FG(3,3,I)
46    FG(8,8,I) = FG(4,4,I)
47    FG(4,3,I) = -DD1(I)/10.-4.*DDXY(I)/10.-11.*DD1(I)/10.
48    FG(3,4,I) = FG(4,3,I)
49    FG(8,7,I) = DD1(I)/10.+4.*DDXY(I)/10.+11.*DD1(I)/10.
50    FG(7,8,I) = FG(8,7,I)
51    FG(5,1,I) = -DH(I)*DLDX*DB(I)/6.
52    FG(1,5,I) = FG(5,1,I)
53    FG(6,2,I) = -DH(I)*DG*DB(I)/6.
54    FG(2,6,I) = FG(6,2,I)
55    FG(7,3,I) = 6.*DD1(I)/(5.*DB(I))+24.*DDXY(I)/(5.*DB(I))
56    .+6.*DD1(I)/(5.*DB(I))
57    FG(3,7,I) = FG(7,7,I)
58    FG(7,4,I) = DD1(I)/10.+4.*DDXY(I)/10.+DD1(I)/10.
59    FG(4,7,I) = FG(7,4,I)

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60      FG(8,4,I) = DB(I)*DD1(I)/30.+4.*DB(I)*DDXY(I)/30.
61      .+DB(I)*DD1(I)/30.
62      FG(4,8,I) = FG(8,4,I)
63      FG(8,3,I) = -DD1(I)/10.-4.*DDXY(I)/10.-DD1(I)/10.
64      FG(3,8,I) = FG(8,3,I)
65      C      THE EPSILON MATRIX *****
66      FE(2,1,I) = DH(I)*DLDX*( -1./2.)-DH(I)*DG*( -1./2.)
67      FE(1,2,I) = -FE(2,1,I)
68      FE(5,2,I) = DG*DH(I)/2.-DLDX*( -DH(I)/2.)
69      FE(2,5,I) = -FE(5,2,I)
70      FE(6,1,I) = DLDX*DH(I)/2.-DG*( -DH(I)/2.)
71      FE(1,6,I) = -FE(6,1,I)
72      FE(6,5,I) = DLDX*( DH(I)/2.)-DG*( DH(I)/2.)
73      FE(5,6,I) = -FE(6,5,I)
74      C      THE ZETA MATRIX *****
75      FZ(1,1,I)=DG*DH(I)/DB(I)
76      FZ(2,2,I) = DLDY*DH(I)/DB(I)
77      FZ(3,3,I) = 12.*DDY(I)/DB(I)**3.
78      FZ(4,4,I) = 4.*DDY(I)/DB(I)
79      FZ(5,5,I) = FZ(1,1,I)
80      FZ(6,6,I) = FZ(2,2,I)
81      FZ(7,7,I) = FZ(3,3,I)
82      FZ(8,8,I) = FZ(4,4,I)
83      FZ(4,3,I) = 6.*DDY(I)/DB(I)**2.
84      FZ(3,4,I) = FZ(4,3,I)
85      FZ(8,7,I) = -6.*DDY(I)/DB(I)**2.
86      FZ(7,8,I) = FZ(8,7,I)
87      FZ(7,4,I) = -6.*DDY(I)/DB(I)**2.
88      FZ(4,7,I) = FZ(7,4,I)
89      FZ(5,1,I)=-DG*DH(I)/DB(I)
90      FZ(1,5,I) = FZ(5,1,I)
91      FZ(6,2,I)=-DLDY*DH(I)/DB(I)
92      FZ(2,6,I) = FZ(6,2,I)
93      FZ(7,3,I) = -12.*DDY(I)/DB(I)**3.
94      FZ(3,7,I) = FZ(7,3,I)
95      FZ(8,4,I) = 2.*DDY(I)/DB(I)
96      FZ(4,8,I) = FZ(8,4,I)
97      FZ(8,3,I) = 6.*DDY(I)/DB(I)**2.
98      FZ(3,8,I) = FZ(8,7,I)
99      1000 CONTINUE
100     RETURN
101     END

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PROJECT*MENAI(1).ANGLE
1      COMPILER(XM=3)
2      SUBROUTINE ANGLE
3      COMMON/EXT/SKU(440,133),SD4CA(440,5),SD2CG(440,3),SDCZE(440),
4      .SDICE(440,3),CA(44,44),CG(44,44),CE(44,44),
5      .CZ(44,44),SIGMAX(180),SIGMAY(180),TAUXY(130),
6      .AMX(180),AMY(180),AMXY(180)
7      COMMON DDY(30),DD1(30),DDX(30),DDY(30),DH(30),DB(30)
8      COMMON FA(8,8,30),FG(8,8,30),FE(8,8,30),FZ(8,8,30)
9      COMMON AFA(8,8,30),AFG(8,8,30),AFE(8,8,30),AFZ(8,8,30)
10     COMMON SD4(6,5,6),SD2(8,3,6),SD1(6,3,6),SD(6,6)
11     COMMON DEX,DEY,DPX,DPY,DG,IDNUM,DLOX,DLDXY,DLDY
12     COMMON MHC,MSKD,MSKB,MSZIC,MSZEPO(250)
13     COMMON NW27,JENOX,JENUM,MS(31),ME(31)
14     COMMON ANG(20),SA
15     COMMON/B2/ATRANS
16     C
17     DIMENSION ASUM(8,8)
18     DOUBLE PRECISION ATRANS(8,8,30)
19     DOUBLE PRECISION ATTRAN(8,8,30)
20     C READ ANGLE OF STRIP: ANGLE IS MEASURED FROM GLOBAL
21     C Y AXIS TO LOCAL Y AXIS FOR STRIP, CLOCKWISE ANGLE IS
22     C CONSIDERED POSITIVE AND COUNTERCLOCKWISE ANGLE IS CONSIDERED
23     C NEGATIVE. ANGLE IS ENTERED IN DEGREES AND FRACTIONS OF A
24     C DEGREE. ORDER OF ENTRY IS AS FOR THICKNESS AND WIDTH CARDS
25     C ONE ANGLE PER CARD
26     C DO 2000 I=1, IDNUM
27     2000 READ (3,2001) ANG(I)
28     2001 FORMAT( )
29     C CHANGE ANGLE TO RADIANS
30     DO 2002 I=1, IDNUM
31     ANG(I) = ANG(I)/57.29577951
32     2002 CONTINUE
33     C FORMULATE TRANSFORMATION MATRICES
34     DO 2003 I = 1, IDNUM
35     ATRANS(1,1,I) = 1.
36     ATRANS(2,2,I) = COS(ANG(I))
37     ATRANS(3,3,I) = ATRANS(2,2,I)
38     ATRANS(4,4,I) = 1.
39     ATRANS(5,5,I) = 1.
40     ATRANS(6,6,I) = ATRANS(2,2,I)
41     ATRANS(7,7,I) = ATRANS(2,2,I)
42     ATRANS(8,8,I) = 1.
43     ATRANS(2,3,I) = SIN(ANG(I))
44     ATRANS(3,2,I) = -ATTRANS(2,3,I)
45     ATRANS(6,7,I) = ATRANS(2,3,I)
46     ATRANS(7,6,I) = -ATTRANS(2,3,I)
47     C FORMULATION OF THE TRANSFORMED TRANSFORMATION MATRICES
48     ATTRAN(1,1,I) = 1.
49     ATTRAN(2,2,I) = COS(ANG(I))
50     ATTRAN(3,3,I) = ATTRAN(2,2,I)
51     ATTRAN(4,4,I) = 1.
52     ATTRAN(5,5,I) = 1.
53     ATTRAN(6,6,I) = ATTRAN(2,2,I)
54     ATTRAN(7,7,I) = ATTRAN(2,2,I)
55     ATTRAN(8,8,I) = 1.
56     ATTRAN(3,2,I) = SIN(ANG(I))
57     ATTRAN(2,3,I) = -ATTRAN(3,2,I)
58     ATTRAN(6,7,I) = -ATTRAN(3,2,I)
59     ATTRAN(7,6,I) = ATTRAN(3,2,I)

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60 2003 CONTINUE
61 C
62 C TO ESTABLISH THE TRANSFORMED ALFA MATRICES (ATTRAN*FA*ATRANS)
63 C
64 DO 2003 I1 = 1, IDNUM
65 C TO CLEAR TEMP. MATRIX ASUM
66 DO 2004 I = 1, 8
67 DO 2004 J = 1, 8
68 ASUM(I, J) = 0.
69 2004 CONTINUE
70 C TO DO FA*ATRANS=ASUM
71 DO 2005 I2 = 1, 8
72 DO 2005 I3 = 1, 8
73 DO 2005 I4 = 1, 8
74 ASUM(I2, I3) = ASUM(I2, I3) + FA(I2, I4, I1) * ATRANS(I4, I3, I1)
75 2005 CONTINUE
76 C TO CLEAR FA( , , I1) SO SAME STORAGE MAY BE USED FOR AFA( , , I1)
77 DO 2006 L = 1, 8
78 DO 2006 M = 1, 8
79 FA(L, M, I1) = 0.
80 2006 CONTINUE
81 C TO DO ATTRAN*ASUM=AFA
82 DO 2007 K2 = 1, 8
83 DO 2007 K3 = 1, 8
84 DO 2007 K4 = 1, 8
85 AFA(K2, K3, I1) = AFA(K2, K3, I1) + ATTRAN(K2, K4, I1) * ASUM(K4, K3)
86 2007 CONTINUE
87 2023 CONTINUE
88 C
89 C TO ESTABLISH THE TRANSFORMED GAMMA MATRICES (ATTRAN*FG*ATRANS)
90 C
91 DO 2008 I1 = 1, IDNUM
92 C TO CLEAR TEMP. MATRIX ASUM
93 DO 2009 I = 1, 8
94 DO 2009 J = 1, 8
95 ASUM(I, J) = 0.
96 2009 CONTINUE
97 C TO DO FG*ATRANS=ASUM
98 DO 2010 I2 = 1, 8
99 DO 2010 I3 = 1, 8
100 DO 2010 I4 = 1, 8
101 ASUM(I2, I3) = ASUM(I2, I3) + FG(I2, I4, I1) * ATRANS(I4, I3, I1)
102 2010 CONTINUE
103 C TO CLEAR FG( , , I1) SO SAME STORAGE MAY BE USED FOR AFG( , , I1)
104 DO 2011 L = 1, 8
105 DO 2011 M = 1, 8
106 FG(L, M, I1) = 0.
107 2011 CONTINUE
108 C TO DO ATTRAN*ASUM = AFG
109 DO 2012 K2 = 1, 8
110 DO 2012 K3 = 1, 8
111 DO 2012 K4 = 1, 8
112 AFG(K2, K3, I1) = AFG(K2, K3, I1) + ATTRAN(K2, K4, I1) * ASUM(K4, K3)
113 2012 CONTINUE
114 2009 CONTINUE
115 C
116 C TO ESTABLISH THE TRANSFORMED EPSILON MATRICES (ATTRAN*FE*ATRANS)
117 DO 2013 I1 = 1, IDNUM
118 C
119 C TO CLEAR TEMP. MATRIX ASUM

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PROJECT* MENAI(1).CONTAC

```

1      COMPILER(XM=3)
2      SUBROUTINE CONTAC
3      COMMON/EXT/SKU(44C,13Z),SD4CA(44C,5),SD2CG(44C,3),SDGZE(44D),
4      .SD1CE(44D,3),CA(44,44),CG(44,44),CE(44,44),
5      .CZ(44,44),SIGMAX(18C),SIGMAY(18C),TAUXY(18C),
6      .AMX(18D),AMY(18D),AMXY(18D)
7      COMMON DDX(3D),DDY(3D),DD1(3C),DDXY(3C),DH(3C),DB(3C)
8      COMMON FA(3,3,3D),FG(3,3,3D),FE(3,3,3D),FZ(3,3,3D)
9      COMMON AFA(3,3,3D),AFG(3,3,3D),AFE(3,3,3D),AFZ(3,3,3D)
10     COMMON SD4(6,5,6),SD2(6,3,6),SD1(6,3,6),SD(6,6)
11     COMMON DEX,DEY,DFY,DPY,DG,IGNUM,DLDX,DLDXY,DLDY
12     COMMON MHC,MSKD,MSKB,MS31D,MSZFP0(25D)
13     COMMON NW27,JENOX,JENUM,MS(31),ME(31)
14     COMMON ANG(3D),SA
15
16     C      READ A CARD TELLING WHICH AF(A,G,E OR Z)(IGNUM) TO PICK(LC3);AND
17     C      ROW OR COLUMN NUMBER WHERE IT MUST START IN C(A,G,E OR Z)
18     C      MATRIX(LC4);ONE CARD PER STRIP.THE MATRICES THAT
19     C      DONT REQUIRE SUBDIVISION ARE DONE FIRST
20     3005 READ(8,3002) LC3,LC4
21     3002 FORMAT(      )
22     C      TEST FOR LAST CARD ; MUST BE 0 , 0 TO STOP PROCESS
23     IF(LC3.EQ.0 .AND. LC4 .EQ. 0 ) GO TO 3006
24     DO 3003 LC1 = 1,8
25     DO 3004 LC2 = 1,8
26     LC13 = (LC4-1+LC1)
27     LC14 = (LC4-1+LC2)
28     CA(LC13,LC14) = CA(LC13,LC14)+AFA(LC1,LC2,LC3)
29     CG(LC13,LC14) = CG(LC13,LC14)+AFG(LC1,LC2,LC3)
30     CE(LC13,LC14) = CE(LC13,LC14)+AFE(LC1,LC2,LC3)
31     CZ(LC13,LC14) = CZ(LC13,LC14)+AFZ(LC1,LC2,LC3)
32     3004 CONTINUE
33     3003 CONTINUE
34     GO TO 3005
35     3006 CONTINUE
36     C      PROCEED WITH *ODD-BALL* PLACEMENT
37     C      READ NUMBER OF *ODD-BALLS*
38     READ(8,3011) LC5
39     3011 FORMAT(      )
40     IF(LC5 .EQ. 0 ) GO TO 3007
41     C      THE ODD-BALLS ARE PLACED IN QUARTERS:1ST LH TOP QUARTER;
42     C      2ND RH TOP QUARTER;3RD LH BOTTOM QUARTER;4TH RH BOTTOM QUARTER
43     C
44     C      READ A CARD TELLING WHICH AF(A,G,E OR Z) (IGNUM) TO PICK
45     C      (LC6);AND ROW(LC7) AND COLUMN NUMBER(LC8) WHERE IT IS
46     C      TO START IN COMBINED MATRIX;THE ROW NUMBER WHERE TO START READING
47     C      IN SUB-MATRIX(LC11)AND THE COLUMN NUMBER WHERE TO START IN SUB-
48     C      MATRIX(LC12)
49     3010 READ(8,3008)LC6,LC7,LC8,LC11,LC12
50     C      TEST FOR LAST CARD ;MUST BE 0,0,0,0,0 TO STOP PROCESS
51     IF(LC6 .EQ. 0 ) GO TO 3007
52     3008 FORMAT(      )
53     C
54     DO 3009 LC9 = 1,4
55     DO 3009 LC10 = 1,4
56     LC15 = (LC7-1+LC9)
57     LC16 = (LC8-1+LC10)
58     LC17 = (LC11-1+LC11)
59     LC18 = (LC10-1+LC12)

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```
60      CA(LC15,LC16) = CA(LC15,LC16)+AFA(LC17,LC18,LC6)
61      CG(LC15,LC16) = CG(LC15,LC16)+AFG(LC17,LC18,LC6)
62      CE(LC15,LC16) = CE(LC15,LC16)+AFE(LC17,LC18,LC6)
63      CZ(LC15,LC16) = CZ(LC15,LC16)+AFZ(LC17,LC18,LC6)
64      3009  CONTINUE
65      GO TO 3010
66      3007  CONTINUE
67      RETURN
68      END
```

APRT.S MENAI.SILO

60 SD4(5,5,3)=1.
 61 SD4(6,3,3)=6.
 62 SD4(6,4,3)=-4.
 63 SD4(6,5,3)=1.
 64 C FOURTH FACE
 65 SD4(3,3,4)=6.
 66 SD4(3,4,4)=-4.
 67 SD4(4,3,4)=6.
 68 SD4(4,4,4)=-4.
 69 SD4(4,5,4)=1.
 70 SD4(5,3,4)=6.
 71 SD4(5,4,4)=-4.
 72 SD4(5,5,4)=1.
 73 SD4(6,3,4)=6.
 74 SD4(6,4,4)=-4.
 75 C FIFTH FACE
 76 SD4(3,3,5)=7.
 77 SD4(4,3,5)=5.
 78 SD4(4,4,5)=-2.
 79 SD4(5,3,5)=7.
 80 SD4(5,4,5)=-4.
 81 SD4(6,3,5)=5.
 82 C SIXTH FACE
 83 SD4(4,3,6)=1.
 84 SD4(5,3,6)=3.
 85 C
 86 C SECOND DERIVATIVE
 87 C
 88 C FIRST FACE
 89 SD2(2,2,1)=-1.
 90 SD2(2,3,1)=1.
 91 SD2(5,2,1)=-1.
 92 SD2(5,3,1)=1.
 93 C SECOND FACE
 94 SD2(1,2,2)=-2.
 95 SD2(1,3,2)=1.
 96 SD2(2,2,2)=-2.
 97 SD2(2,3,2)=1.
 98 SD2(3,2,2)=-2.
 99 SD2(3,3,2)=1.
 100 SD2(4,2,2)=-2.
 101 SD2(4,3,2)=1.
 102 SD2(5,2,2)=-2.
 103 SD2(5,3,2)=1.
 104 SD2(6,2,2)=-2.
 105 SD2(6,3,2)=1.
 106 C THIRD FACE
 107 SD2(1,2,3)=-2.
 108 SD2(1,3,3)=1.
 109 SD2(2,2,3)=-2.
 110 SD2(2,3,3)=1.
 111 SD2(3,2,3)=-2.
 112 SD2(3,3,3)=1.
 113 SD2(4,2,3)=-2.
 114 SD2(4,3,3)=1.
 115 SD2(5,2,3)=-2.
 116 SD2(5,3,3)=1.
 117 SD2(6,2,3)=-2.
 118 SD2(6,3,3)=1.
 119 C FOURTH FACE

120 SD2(1,2,4)=-2.
 121 SD2(1,3,4)=1.
 122 SD2(2,2,4)=-2.
 123 SD2(2,3,4)=1.
 124 SD2(3,2,4)=-2.
 125 SD2(3,3,4)=1.
 126 SD2(4,2,4)=-2.
 127 SD2(4,3,4)=1.
 128 SD2(5,2,4)=-2.
 129 SD2(5,3,4)=1.
 130 SD2(6,2,4)=-2.
 131 SD2(6,3,4)=1.
 132 C FIFTH FACE
 133 SD2(1,2,5)=-2.
 134 SD2(2,2,5)=-2.
 135 SD2(2,3,5)=1.
 136 SD2(3,2,5)=-2.
 137 SD2(4,2,5)=-2.
 138 SD2(4,3,5)=1.
 139 SD2(5,2,5)=-2.
 140 SD2(5,3,5)=1.
 141 SD2(6,2,5)=-2.
 142 SD2(6,3,5)=1.
 143 C SIXTH FACE
 144 SD2(2,2,6)=-1.
 145 SD2(5,2,6)=-1.
 146 C
 147 C FIRST DERIVATIVE
 148 C
 149 C FIRST FACE
 150 SD1(4,2,1)=-1.
 151 SD1(4,3,1)=1.
 152 SD1(6,3,1)=1.
 153 C SECOND FACE
 154 SD1(1,3,2)=1.
 155 SD1(2,3,2)=1.
 156 SD1(3,3,2)=1.
 157 SD1(4,3,2)=1.
 158 SD1(5,3,2)=1.
 159 SD1(6,3,2)=1.
 160 C THIRD FACE
 161 SD1(1,3,3)=1.
 162 SD1(2,3,3)=1.
 163 SD1(3,3,3)=1.
 164 SD1(4,3,3)=1.
 165 SD1(5,3,3)=1.
 166 SD1(6,3,3)=1.
 167 C FOURTH FACE
 168 SD1(1,3,4)=1.
 169 SD1(2,3,4)=1.
 170 SD1(3,3,4)=1.
 171 SD1(4,3,4)=1.
 172 SD1(5,3,4)=1.
 173 SD1(6,3,4)=1.
 174 C FIFTH FACE
 175 SD1(2,3,5)=1.
 176 SD1(4,3,5)=1.
 177 SD1(5,3,5)=1.
 178 SD1(6,3,5)=1.
 179 C SIXTH FACE

```

300      SD2CG(MS5,L)=SD2(MRS,L,J)
301      SD1CE(MS5,L)=SD1(MRS,L,J)
302      4009  CONTINUE
303      SD07E(MS5)=SD(MRS,J)
304      4007  CONTINUE
305      C     COPY IN V VALUES
306      DO 4018 J=1,3
307      MS7=((I*4-2)+(J-1)*MS4)
308      DO 4019 K=1,5
309      SD4CA(MS7,K)=SD4(MS6,K,J)
310      4019  CONTINUE
311      DO 4020 L=1,3
312      SD2CG(MS7,L)=SD2(MS6,L,J)
313      SD1CE(MS7,L)=SD1(MS6,L,J)
314      4020  CONTINUE
315      SD07E(MS7)=SD(MS6,J)
316      4018  CONTINUE
317      C     COPY IN W VALUES
318      DO 4028 J=1,3
319      MS10=((I*4-1)+(J-1)*MS4)
320      DO 4029 K=1,5
321      SD4CA(MS10,K)=SD4(MS9,K,J)
322      4029  CONTINUE
323      DO 4030 L=1,3
324      SD2CG(MS10,L)=SD2(MS9,L,J)
325      SD1CE(MS10,L)=SD1(MS9,L,J)
326      4030  CONTINUE
327      SD07E(MS10)=SD(MS9,J)
328      4028  CONTINUE
329      C     COPY IN THETA VALUES
330      DO 4033 J=1,3
331      MS13=((I*4)+(J-1)*MS4)
332      DO 4034 K=1,5
333      SD4CA(MS13,K)=SD4(MS12,K,J)
334      4034  CONTINUE
335      DO 4035 L=1,3
336      SD2CG(MS13,L)=SD2(MS12,L,J)
337      SD1CE(MS13,L)=SD1(MS12,L,J)
338      4035  CONTINUE
339      SD07E(MS13)=SD(MS12,J)
340      4033  CONTINUE
341      4003  CONTINUE
342      C     END OF STARTING CONDITIONS
343      C
344      C     END CONDITIONS
345      DO 4036 I=1,31
346      C     TEST FOR LAST ENTRY
347      IF (ME(I).EQ.0) GO TO 4036
348      C
349      IF (ME(I).EQ.1) GO TO 4037
350      IF (ME(I).EQ.3) GO TO 4039
351      IF (ME(I).EQ.5) GO TO 4044
352      IF (ME(I).EQ.7) GO TO 4046
353      4037  MS14=1
354      MS18=7
355      MS22=3
356      MS26=3
357      GO TO 4048
358      4039  MS14=1
359      MS18=5

```

```

420 SD4CA(MS27,K)=SD4(MS26,K,MS28)
421 4067 CONTINUE
422 DO 4068 L=1,3
423 SD2CG(MS27,L)=SD2(MS26,L,MS28)
424 SD1CE(MS27,L)=SD1(MS26,L,MS28)
425 4068 CONTINUE
426 SD0ZE(MS27)=SD(MS26,MS28)
427 4066 CONTINUE
428 4030 CONTINUE
429 C END OF END CONDITIONS
430 C STRUCTURE MUST HAVE AT LEAST 7 NODES (6 DIVISIONS) ALONG LENGTH.
431 C READ NODAL SPACING ,AND DIVIDE BY SA**4,SA**2,.22*SA.
432 READ(8,4069) SA
433 4069 FORMAT( )
434 SA4=SA**4
435 SA2=SA**2
436 SA3=SA*2.
437 DO 4070 I=1,MSKD
438 DO 4071 J=1,5
439 SD4CA(I,J)=SD4CA(I,J)/SA4
440 4071 CONTINUE
441 DO 4072 K=1,3
442 SD2CG(I,K)=SD2CG(I,K)/SA2
443 4072 CONTINUE
444 DO 4073 L=1,3
445 SD1CE(I,L)=SD1CE(I,L)/(SA3)
446 4073 CONTINUE
447 4070 CONTINUE
448 C
449 C ROWS AND COLUMNS ARE REDUCED TO ZERO IN THE SKU
450 C MATRIX TO ALLOW FOR SUPPORT CONDITIONS.
451 C
452 C READ NUMBER OF ROWS TO BE REDUCED TO ZERO.
453 READ(8,4400) MS31C
454 4400 FORMAT( )
455 C ROW NUMBERS TO BE ZEROED ARE ENTERED 10 TO A CARD IN FREE FORMAT
456 C FIRST INDICATE HOW MANY CARDS TO BE READ. (ZEROS TO BE USED
457 C TO FILL FIELD)
458 C READ NUMBER OF CARDS TO BE READ(MS401)
459 READ(9,4400) MS401
460 NN=1
461 4401 IF(NN.GT.MS401) GO TO 4402
462 MS403=10*NN-9
463 MS402=10*NN
464 READ(8,4400) (MSZERO(I),I=MS403,MS402)
465 NN=NN+1
466 GO TO 4401
467 4402 CONTINUE
468 C
469 C TO FACILITATE MATRIX MULTIPLICATION,THE SD4CA,SD2CG,&SD1CE
470 C MATRICES HAVE COLUMNS SHIFTED UP ON L.H. SIDE OF MAIN
471 C DIAGONAL AND DOWN ON RH SIDE OF MAIN DIAGONAL.
472 C
473 C 4TH DERIVATIVE
474 MS35=2*MS4
475 MS36=1*MS4
476 MS38=MSKD-MS35
477 DO 4074 I=1,MS38
478 MS37=I+MS35
479 SD4CA(I,1)=SD4CA(MS37,1)

```

```
500 4317 CONTINUE
501      DO 4323 K=1,MS310
502      MS324=MSZERO(K)
503      SKU(MS324,1)=1.
504 4323 CONTINUE
505      RETURN
506      END
```

APRT,S MENAI.WHEEL

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60      SKU(NW5,NW27)=WW(J)
61      SKU(NW6,NW27)=WTHETA(J)
62      5002 CONTINUE
63      C
64      C FAR END
65      NW7=MSKD-NW1+1
66      DO 5003 K=1,NW1
67      NW8=NW7+K-1
68      SKU(NW8,NW27)=SKU(K,NW27)
69      5003 CONTINUE
70      C
71      C CENTRAL SECTION
72      C
73      C DOUBLE END VALUES OF LOADING.
74      DO 5004 L=1,NW2
75      WU(L)=2.*WU(L)
76      WV(L)=2.*WV(L)
77      WW(L)=2.*WW(L)
78      WTHETA(L)=2.*WTHETA(L)
79      5004 CONTINUE
80      C SUBSTITUTE INTO CENTRAL SECTION OF SKU( ,NW27)
81      NW9=(MSKD-2*NW1)/NW1
82      DO 5005 M=1,NW9
83      C STARTING ADDRESS IN SKU( ,NW27)
84      NW10=NW1+1
85      DO 5006 N=1,NW2
86      NW15=NW10+(M-1)*4+NW2+(N-1)*4
87      NW16=NW15+1
88      NW17=NW15+2
89      NW18=NW15+3
90      SKU(NW15,NW27)=WU(N)
91      SKU(NW16,NW27)=WV(N)
92      SKU(NW17,NW27)=WW(N)
93      SKU(NW18,NW27)=WTHETA(N)
94      5006 CONTINUE
95      5005 CONTINUE
96      C
97      C LIVE LOADS
98      C
99      C READ NUMBER OF NODES HAVING LIVE LOADS.
100     READ(8,5007) NW19
101     5007 FORMAT( )
102     C NW20 IS NODE NUMBER,NW21 IS CROSS-SECTION NUMBER; OTHERS ARE
103     C GLOBAL LIVE LOADS.
104     DO 5011 I=1,NW19
105     READ(8,5009)NW20(I),NW21(I),WLU(I),WLV(I),WLW(I),WLTHE(I)
106     5009 FORMAT( )
107     5011 CONTINUE
108     C COPY INTO SKU( ,NW27)
109     I=1
110     5016 NW22=(4*NW20(I)-3)+(NW21(I)-1)*NW1
111     NW23=NW22+1
112     NW24=NW22+2
113     NW25=NW22+3
114     SKU(NW22,NW27)=SKU(NW22,NW27)+WLU(I)
115     SKU(NW23,NW27)=SKU(NW23,NW27)+WLV(I)
116     SKU(NW24,NW27)=SKU(NW24,NW27)+WLW(I)
117     SKU(NW25,NW27)=SKU(NW25,NW27)+WLTHE(I)
118     I=I+1
119     IF(I.GT.NW19) GO TO 5010

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```
120          GO TO 5016
121      5010  CONTINUE
122      C
123      C      PUT LOADS AT SUPPORT TO ZERO.
124      C
125          DO 5015 K=1,MS310
126          NW28=MSZERO(K)
127          SKU(NW28,NW27)=0.
128      5015  CONTINUE
129          RETURN
130          END
```

@PRT: S MENAI.GRAFT

PROJECT*MEFNAI(1).ECHO

```

1      COMPILER(XM=3)
2      SUBROUTINE ECHO (EDV, JE1)
3      COMMON/EXT/SKU(440,133),SD4CA(440,5),SD2CG(440,3),SDCZE(440),
4      .SDICE(440,3),CA(44,44),CG(44,44),CE(44,44),
5      .CZ(44,44),SIGMAX(180),SIGMAY(180),TAUXY(180),
6      .AMX(180),AMY(180),AMXY(180)
7      COMMON DDY(30),DDY(30),DB1(30),DDXY(30),DH(30),DB(30)
8      COMMON FA(8,8,30),FG(8,8,30),FE(8,8,30),FZ(8,8,30)
9      COMMON AFA(8,8,30),AFG(8,8,30),AFE(8,8,30),AFZ(8,8,30)
10     COMMON SD4(6,5,6),SD2(6,3,6),SD1(6,3,6),SD(6,6)
11     COMMON DEX,DEY,DPY,DPY,DEG, IDNUM,BLDX,BLDXY,DLDY
12     COMMON MHC,MSKD,MSKB,MS3IC,MSZERO(253)
13     COMMON NW27,JENOX,JENUM,MS(31),ME(31)
14     COMMON ANG(30),SA
15     COMMON/B2/ATRANS
16     DIMENSION JETRID(16),JEST(16),JEMID(16),JEEND(16)
17     DIMENSION EDV(JE1)
18     DOUBLE PRECISION ATRANS(8,8,30)
19     DIMENSION EGDIS(8),ELDIS(8)
20     DOUBLE PRECISION EDUX(3),EDVY(3),EDVX(3),EDUY(3),ED2WX2(3)
21     DOUBLE PRECISION ED2HY2(3),ED2WXY(3)
22     C      READ NUMBER OF TWO STRIP SUB-STRUCTURES:JENUM
23     READ(6,7000) JENUM
24     7000  FORMAT( )
25     C      FOR EACH SUB-STRUCTURE READ:ONE STRIP NUMBER WITH SAME
26     C      ANGLE(JETRID); NUMBER IN SKU, ON FIRST CROSS-SECTION, WHERE
27     C      FIRST NODE DISPLACEMENTS BEGIV (JEST);CONTINUE(JEMID);
28     C      AND END(JEEND)
29     DO 7001 I=1,JENUM
30     READ(8,7000) JETRID(I),JEST(I),JEMID(I),JEEND(I)
31     7001  CONTINUE
32     C      NUMBER OF CROSS-SECTIONS IN STRUCTURE(JENOX)
33     JENOX=3*MSKD/MSKB
34     C
35     C      IF FICTICIOUS NODES USED, JENOX IS REDUCED
36     IF(MS(1).EQ.6) JENOX=JENOX-1
37     IF(ME(1).EQ.6) JENOX=JENOX-1
38     C      SPACE TO ALLOW IN NEW DISPLACEMENT VECTOR EDV IS
39     C      4*3*NUMBER OF SUB-STRUCTURES*NUMBER OF CROSS-SECTIONS
40     C      IN COMPLETE STRUCTURE (THIS IS CALCULATED AND PLACED
41     C      IN LIST IN MAIN PROGRAM).NUMBER IS JE1.
42     C
43     C      NUMBER OF DEGREES OF FREEDOM PER CROSS-SECTION IN NEW EDV
44     JE2=JE1/JENOX
45     C      NUMBER OF DEGREES OF FREEDOM IN CROSS-SECTION IN OLD SKU
46     JES=MSKB/7
47     JA=C
48     IF(MS(1).EQ.6) JA=1
49     C
50     C      PERFORM TRANSFORMATIONS (CALCULATE LOCAL DISPLACEMENTS)
51     C
52     C      FOR NUMBER OF CROSS-SECTIONS IN STRUCTURE(JENOX)
53     DO 7002 J=1,JENOX
54     C      FOR NUMBER OF TWO STRIP SUB-STRUCTURES(JENUM)
55     DO 7003 K=1,JENUM
56     C      LOAD GLOBAL DISPLACEMENTS INTO EGDIS
57     C      FIRST FOUR
58     DO 7004 L=1,4
59     JF3=L

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60      JE4=JEST(K)+(J-1+JA)*JES*L-1
61      EGDIS(JE3)=SKU(JE4,NW27)
62      7004  CONTINUE
63      C      LAST FOUR
64      DO 7005 M=1,4
65      JES=4+M
66      JEC=JEMID(K)+(J-1+JA)*JES*M-1
67      EGDIS(JE5)=SKU(JE6,NW27)
68      7005  CONTINUE
69      C      MULTIPLY: ATRANS*EGDIS
70      DO 7006 N=1,8
71      DO 7006 I=1,8
72      JE7=JETRID(K)
73      ELDIS(N)=ELDIS(N)+ATrans(N,I,JE7)*EGDIS(I)
74      7006  CONTINUE
75      C      COPY INTO EDV
76      DO 7007 I2=1,8
77      JES=(J-1)*JE2+(12*K-11)+I2-1
78      EDV(JE8)=ELDIS(I2)
79      7007  CONTINUE
80      C      CLEAR ELDIS AND EGDIS
81      DO 7008 J2=1,8
82      ELDIS(J2)=0.
83      EGDIS(J2)=0.
84      7008  CONTINUE
85      C      LOAD LAST OF SET OF THE THREE INTO EGDIS
86      DO 7009 K2=1,4
87      JE10=JEEND(K)+(J-1+JA)*JES+K2-1
88      EGDIS(K2)=SKU(JE10,NW27)
89      7009  CONTINUE
90      C      MULTIPLY: ATRANS*EGDIS
91      DO 7010 L2=1,4
92      DO 7010 M2=1,4
93      JE11=JETRID(K)
94      ELDIS(L2)=ELDIS(L2)+ATrans(L2,M2,JE11)*EGDIS(M2)
95      7010  CONTINUE
96      C      COPY INTO EDV
97      DO 7011 N2=1,4
98      JE12=(J-1)*JE2+12*K-3+N2-1
99      EDV(JE12)=ELDIS(N2)
100     7011  CONTINUE
101     C      CLEAR ELDIS AND EGDIS
102     DO 7012 I3=1,8
103     ELDIS(I3)=0.
104     EGDIS(I3)=0.
105     7012  CONTINUE
106     7003  CONTINUE
107     7002  CONTINUE
108     C      NUMBER OF CROSS-SECTIONS EXCLUDING FIRST AND LAST
109     JE16=JEVOX-?
110     C      STRESS PACKAGES REQUIRED IN CENTRAL SECTION
111     JE17=JE16+JENUM
112     C
113     C      STRESS CALCULATIONS FOR FIRST CROSS-SECTION
114     C
115     DO 7013 I=1,JENUM
116     C      TO GET TO STARTING DISPLACEMENT IN EDV
117     JES*DV=12*I-11
118     C
119     C      L.H.NODE

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180 C CALCULATE ED2WX2(2)
181 JE272=JESTDV+6
182 JE273=JESTDV+6+JE2
183 JE274=JESTDV+6+2*JE2
184 JE275=JESTDV+6+3*JE2
185 ED2WX2(2)=(2.*EDV(JE272)-5.*EDV(JE273)+4.*EDV(JE274)
186 -1.*EDV(JE275))/(SA**2.)
187 C CALCULATE ED2WY2(2)
188 JE47=JESTDV+2
189 JE48=JESTDV+6
190 JE49=JESTDV+10
191 ED2WY2(2)=(1.*EDV(JE47)-2.*EDV(JE48)+1.*EDV(JE49))/(DB(JE23)**2)
192 C CALCULATE ED2WXY(2)
193 JE50=JESTDV+7
194 JE51=JESTDV+7+JE2
195 JE52=JESTDV+7+2*JE2
196 ED2WXY(2)=(-3.*EDV(JE50)+4.*EDV(JE51)-1.*EDV(JE52))/
197 *(2.*SA)
198 C
199 C R.H. NOJE
200 C
201 C CALCULATE EDUX(3)
202 JE54=JESTDV+8
203 JE55=JE54+JE2
204 JE56=JE54+JE2*2
205 EDUX(3)=(-3.*EDV(JE54)+4.*EDV(JE55)-1.*EDV(JE56))/(2.*SA)
206 C CALCULATE EDVY(3)
207 JE57=JESTDV+1
208 JE58=JESTDV+5
209 JE59=JESTDV+9
210 EDVY(3)=(1.*EDV(JE57)-4.*EDV(JE58)+3.*EDV(JE59))/(2.*DB(JE23))
211 C CALCULATE EDVX(3)
212 JE60=JESTDV+9
213 JE61=JE60+JE2
214 JE62=JESTDV+9+2*JE2
215 EDVX(3)=(-3.*EDV(JE60)+4.*EDV(JE61)-1.*EDV(JE62))/(2.*SA)
216 C CALCULATE EDUY(3)
217 JE63=JESTDV
218 JE64=JESTDV+4
219 JE65=JESTDV+8
220 EDUY(3)=(1.*EDV(JE63)-4.*EDV(JE64)+3.*EDV(JE65))/(2.*DB(JE23))
221 C CALCULATE ED2WX2(3)
222 JE276=JESTDV+10
223 JE277=JESTDV+10+JE2
224 JE278=JESTDV+10+2*JE2
225 JE279=JESTDV+10+3*JE2
226 ED2WX2(3)=(2.*EDV(JE276)-5.*EDV(JE277)+4.*EDV(JE278)
227 -1.*EDV(JE279))/(SA**2.)
228 C CALCULATE ED2WY2(3)
229 JE253=JESTDV+3
230 JE254=JESTDV+7
231 JE255=JESTDV+11
232 ED2WY2(3)=(1.*EDV(JE253)-4.*EDV(JE254)+3.*EDV(JE255))
233 /((DB(JE23)*2.)
234 C CALCULATE ED2WXY(3)
235 JE72=JESTDV+11
236 JE73=JESTDV+11+JE2
237 JE74=JESTDV+11+2*JE2
238 ED2WXY(3)=(-3.*EDV(JE72)+4.*EDV(JE73)-1.*EDV(JE74))/
239 *(2.*SA)

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360      JE122=JECOV+4
361      JE123=JECOV+8
362      EDUY(3)=(1.*EDV(JE121)-4.*EDV(JE122)+3.*EDV(JE123))/(2.*DB(JE82))
363      C      CALCULATE ED2WX2(3)
364      JE124=JECOV+10-JE2
365      JE125=JECOV+10
366      JE126=JECOV+10+JE2
367      ED2WX2(3)=(1.*EDV(JE124)-2.*EDV(JE125)+1.*EDV(JE126))/(SA**2)
368      C      CALCULATE ED2WY2(3)
369      JE259=JECOV+3
370      JE260=JECOV+7
371      JE261=JECOV+11
372      ED2WY2(3)=(1.*EDV(JE259)-4.*EDV(JE260)+3.*EDV(JE261))
373      ./((DB(JE82)+2.))
374      C      CALCULATE ED2WXY(3)
375      JE130=JECOV+11-JE2
376      JE131=JECOV+11+JE2
377      ED2WXY(3)=(-1.*EDV(JE130)+1.*EDV(JE131))/(2.*SA)
378      C
379      C      CALCULATE STRESSES
380      C
381      DO 7C17, M=1,3
382      JE133=JENUM+3+(LS-1)*JENUM+3+3*(L-1)+M
383      SIGMAX(JE133)=DLDY*EDUX(M)+CLOXY*EDVY(M)
384      SIGMAY(JE133)=DLDY*EDUX(M)+DLDY*EDVY(M)
385      TAUXY(JE133)=DG*(EDVX(M)+EDUY(M))
386      C
387      AMX(JE133)=-((DDY(JE82)*ED2WX2(M)+DD1(JE82)*ED2WY2(M))
388      AMY(JE133)=-((DD1(JE82)*ED2WX2(M)+DDY(JE82)*ED2WY2(M))
389      AMXY(JE133)=DDXY(JE82)*2.*ED2WXY(M)
390      7C17 CONTINUE
391      C      CLEAR DERIVATIVE VECTORS.
392      DO 7C18, N=1,3
393      EDUX(N)=0.
394      EDVY(N)=0.
395      EDVX(N)=0.
396      EDUY(N)=0.
397      ED2WX2(N)=0.
398      ED2WY2(N)=0.
399      ED2WXY(N)=0.
400      7C18 CONTINUE
401      7C16 CONTINUE
402      7C22 CONTINUE
403      C
404      C      STRESS CALCULATIONS FOR LAST CROSS SECTION.
405      C
406      DO 7C19, I=1,JENUM
407      C      TO GET TO STARTING ADDRESS
408      JEEDV=JE1-JE2+1+I*12-11-1
409      C
410      C      L.H. NODE.
411      C
412      C      CALCULATE EDUX(1)
413      JE134=JEEDV-2+JE2
414      JE135=JEEDV-JE2
415      JE136=JEEDV
416      EDUX(1)=(1.*EDV(JE134)-4.*EDV(JE135)+3.*EDV(JE136))/(2.*SA)
417      C      CALCULATE EDVY(1)
418      JE137=JEEDV+1
419      JE138=JEEDV+5

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.....
 MEMAT: TRANSLATIONAL SHELL PROGRAM

DATE: 23. IX. 76

PROJECT: EX. No: 2

CLIENT: U.C.T.

GENERAL DESCRIPTION OF STRUCTURE: Thin walled
 roof structure

LOADING CASE CONSIDERED: Uniformly distributed loading

.....

ALL UNITS USED ARE N.M.S.

YOUNG'S MODULUS X: 2.524430+10
 YOUNG'S MODULUS Y: 2.524430+10
 POISSON RATIO X: .000
 POISSON RATIO Y: .000
 SHEAR MODULUS G: 1.262215+10

STRIP NO	STRIP WIDTH	STRIP THICKNESS	STRIP ANGLE (RAD)
1	.381	.102	.000000
2	.381	.102	.000000
3	1.078	.102	.7853982
4	1.078	.102	.7853982
5	1.078	.102	.7853982
6	1.078	.102	.7853982
7	.762	.102	.000000
8	.762	.102	.000000
9	1.078	.102	-.7853982
10	1.078	.102	-.7853982

BOUNDARY CONDITION AT START IS SIMPLY SUPPORTED

BOUNDARY CONDITION AT END IS SIMPLY SUPPORTED

NODAL SPACINGS = 4.87E

CROSS-SECTION	NODE	GLOBAL DISPLACEMENTS			THETA (RAD)
		U	V	W	

1					
1	6	6.713910-C4	0.000000	0.000000	0.000000
2	5	5.738274-C4	0.000000	0.000000	0.000000
3	6	6.790395-C4	0.000000	0.000000	0.000000
4	3	3.042237-C4	0.000000	0.000000	0.000000
5	1	1.244441-C4	0.000000	0.000000	0.000000
6	-1	-1.424428-C4	0.000000	0.000000	0.000000
7	-4	-4.223833-C4	0.000000	0.000000	0.000000
8	-4	-4.204766-C4	0.000000	0.000000	0.000000
9	-4	-4.309293-C4	0.000000	0.000000	0.000000
10	1	1.173533-C4	0.000000	0.000000	0.000000
11	5	5.875770-C4	0.000000	0.000000	0.000000

2					
1	5	5.360041-C4	0.000000	1.713799-C3	0.000000
2	5	5.371815-C4	-2.688409-C6	1.781796-C3	3.735671-C4
3	5	5.407196-C4	-5.317646-C6	2.023797-C3	9.467509-C4
4	3	3.230387-C4	-1.077437-C3	3.088147-C3	1.358996-C3
5	1	1.157687-C4	-1.600011-C3	3.607521-C3	-1.344166-C4

6	-3.C99204-C5	-9.148399-04	2.924230-C3	-1.463429-C3
7	-3.C47533-C4	6.769562-C5	1.948491-C3	-5.515919-C4
8	-3.C42383-C4	7.297957-C5	2.169852-C3	1.065552-C3
9	-3.122381-C4	7.861925-C5	3.674173-C7	3.C86015-C3
10	1.C20991-C4	3.475441-C3	7.062693-C3	5.323881-C3
11	5.179552-C4	7.708732-C3	1.1129326-C2	5.665945-C3
1	2.161337-C4	C.C00000	2.740214-C3	C.C00000
2	2.161003-C4	-2.702747-C5	2.809725-C3	3.811459-C4
3	2.162189-C4	-5.318717-C6	3.C55387-C3	9.571027-C4
4	1.546967-C4	-1.078331-C3	4.120557-C3	1.341143-C3
5	9.355229-C5	-1.562524-C3	4.601414-C3	-2.149166-C4
6	3.250743-C5	-8.186408-C4	3.859541-C3	-1.505669-C3
7	-2.865323-C5	1.062227-C4	2.541675-C3	-3.116564-C4
8	3.110686-C5	1.110335-C4	3.466108-C3	1.629470-C3
9	-3.361555-C5	1.173849-C4	5.535163-C3	4.C04687-C3
10	6.704078-C5	4.379402-C3	9.789774-C3	6.536443-C7
11	1.676781-C4	9.680650-C3	1.508747-C2	7.111088-C3
1	-1.112290-C4	C.C00000	2.496516-C3	C.C00000
2	-1.122719-C4	-2.735372-C5	2.568805-C3	3.957494-C4
3	-1.152920-C4	-5.37217-C6	2.822855-C3	9.866008-C4
4	-1.307332-C5	-1.11512-C3	3.521034-C3	1.391016-C3
5	7.C27725-C5	1.C27987-C3	4.434011-C3	-1.197275-C4
6	1.575748-C4	-8.835941-C4	3.691461-C3	-1.542335-C3
7	2.510605-C4	1.074070-C4	2.707132-C3	-4.670308-C4
8	2.472400-C4	1.127225-C4	3.093359-C3	1.422810-C3
9	2.C14121-C4	1.185867-C4	5.002349-C3	3.802123-C3
10	3.407303-C5	4.241921-C3	9.117022-C3	6.469384-C3
11	-1.843003-C4	9.421050-C3	1.429327-C2	6.953793-C3
1	-2.685055-C4	C.C00000	1.241445-C3	C.C00000
2	-2.709534-C4	-2.482995-C5	1.314165-C3	3.983864-C4
3	-2.779759-C4	-5.040679-C6	1.570412-C3	9.568251-C4
4	-3.936151-C5	-1.133907-C3	2.692195-C3	1.447674-C3
5	5.792675-C5	-1.716741-C3	3.272541-C3	-7.641179-C5
6	2.134452-C4	-1.C38399-C3	2.4595548-C3	-1.520252-C3
7	3.844460-C4	6.511901-C5	1.497749-C3	-8.350152-C4
8	1.755572-C4	6.997213-C5	1.441305-C3	6.416317-C4
9	3.837765-C4	7.599415-C5	2.611291-C3	2.653218-C7
10	2.412792-C5	7.167470-C3	5.694932-C3	4.951219-C3
11	-3.364479-C4	7.128014-C3	9.653012-C3	5.316571-C3
1	-9.687506-C5	C.C00000	0.C00000	C.C00000
2	-1.168109-C5	1.168109-C5	0.000000	2.646914-C9
3	-9.774591-C5	2.328058-C5	C.C00000	-2.657754-C7
4	-9.807916-C6	7.210935-C5	0.000000	-2.904651-C5
5	6.632953-C5	8.966197-C5	C.C00000	-8.294943-C8
6	1.424033-C4	7.519926-C5	0.000000	2.297917-C5
7	2.200557-C4	2.885482-C5	C.C00000	1.734194-C5
8	2.110533-C4	5.287794-C6	0.000000	1.740755-C6
9	2.034332-C4	-2.665600-C5	0.C00000	-2.218002-C5
10	5.761953-C5	-7.618770-C5	0.000000	-1.237761-C5
11	-8.877468-C5	-9.361292-C5	0.C00000	-1.185675-C5
1	1.305823-C4	C.C00000	1.779773-C4	0.C00000
2	1.325089-C4	-2.510152-C6	2.547040-C4	4.194171-C4
3	1.383276-C4	-5.C95052-C6	5.229632-C4	1.C38838-C3
4	9.809013-C5	-1.179551-C3	1.690202-C3	1.513097-C3
5	7.581594-C5	-1.1798394-C3	2.306359-C3	-5.890875-C9

6	5.497119-05	-1.090598-C3	1.599626-C3	-1.026923-C3
7	2.176159-05	1.650049-C4	3.494121-C4	-1.140497-C3
8	1.729418-05	1.697678-C4	6.391228-C6	2.081192-C4
9	-1.003444-C6	1.752176-C4	8.222939-C4	2.170722-C3
10	2.692964-C5	2.895426-C3	3.533882-C3	4.458821-C3
11	1.947189-C4	6.479163-C3	7.114885-C3	4.620774-C2
1	7.933516-C5	0.000000	5.921125-C4	0.000000
2	8.057816-C5	-2.796135-C6	6.726673-C4	4.391561-C4
3	9.244223-C5	-5.430869-C6	9.515018-C4	1.073839-C3
4	7.221228-C5	-1.208018-C3	2.145897-C3	1.533666-C3
5	6.732393-C5	-1.802234-C3	2.743213-C3	-1.314709-C4
6	6.300328-C5	-1.027486-C3	1.902777-C7	-1.722522-C3
7	5.470070-C5	2.417491-C4	6.995927-C4	-1.034909-C3
8	5.945087-C5	2.469435-C4	5.342406-C4	5.766193-C4
9	5.775141-C5	2.528171-C4	1.742606-C3	2.834215-C3
10	1.025688-C4	3.615853-C3	5.796556-C7	5.445560-C3
11	1.503963-C4	9.004011-C3	9.481738-C3	5.908505-C3
1	-7.205888-C5	0.000000	4.632338-C4	0.000000
2	-7.250740-C5	-2.729866-C6	5.372128-C4	4.049072-C4
3	-7.405771-C5	-5.401256-C6	7.971692-C4	1.009627-C3
4	-8.570914-C6	-1.146247-C3	1.930056-C3	1.458784-C3
5	5.232231-C5	-1.726192-C3	2.505479-C3	-1.012614-C4
6	1.128701-C4	-3.008585-C3	1.790371-C3	-1.599344-C3
7	1.757097-C4	1.744293-C4	6.136251-C4	-9.595065-C4
8	1.802561-C4	1.795669-C4	4.524039-C4	4.892519-C4
9	2.093242-C4	1.852685-C4	1.492901-C3	2.464310-C3
10	0.520712-C5	3.115524-C3	4.414315-C3	4.717124-C3
11	-1.771573-C5	6.887735-C3	8.187890-C3	5.002229-C3
1	-1.515697-C4	0.000000	0.000000	0.000000
2	-1.526485-C4	0.000000	0.000000	0.000000
3	-1.533398-C4	0.000000	0.000000	0.000000
4	-5.055295-C5	0.000000	0.000000	0.000000
5	4.552607-C5	0.000000	0.000000	0.000000
6	1.4000458-C4	0.000000	0.000000	0.000000
7	2.405376-C4	0.000000	0.000000	0.000000
8	2.594505-C4	0.000000	0.000000	0.000000
9	2.353690-C4	0.000000	0.000000	0.000000
10	9.042328-C5	0.000000	0.000000	0.000000
11	-1.073899-C4	0.000000	0.000000	0.000000

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10

CROSS-SECTION SUB-STRUCTURE NODE

LOCAL STRESSES

M-X M-Y M-ZY

SIGMA-X SIGMA-Y SIGMA-ZY

SIGMA-X SIGMA-Y SIGMA-ZY

M-X M-Y M-ZY

CROSS-SECTION	SUB-STRUCTURE	NODE	SIGMA-X	SIGMA-Y	SIGMA-ZY	M-X	M-Y	M-ZY
1	1	1	-2.27023+05	0.00000	-2.00068+02	9.82633+00	0.00000	0.00000
		2	-2.28670+05	0.00000	1.05817+01	2.24309+01	0.00000	0.00000
		3	-2.33610+05	0.00000	2.14029+05	6.76338+01	0.00000	0.00000
	2	1	-2.37050+05	0.00000	1.17679+06	4.85342+01	0.00000	0.00000
		2	-1.17040+05	0.00000	1.33040+06	3.30289+02	0.00000	0.00000
		3	-1.39345+04	0.00000	1.49662+06	4.94244+02	0.00000	0.00000
	3	1	-1.17040+05	0.00000	1.40676+06	3.36289+02	0.00000	0.00000
		2	-1.39345+04	0.00000	1.42026+06	4.94244+02	0.00000	0.00000
		3	8.76323+04	0.00000	1.44765+06	3.35825+02	0.00000	0.00000
4	1	-1.39345+04	0.00000	1.49662+06	4.94244+02	0.00000	0.00000	
	2	8.76323+04	0.00000	1.42026+06	4.94244+02	0.00000	0.00000	
	3	1.99298+05	0.00000	1.76081+06	4.79734+01	0.00000	0.00000	
5	1	1.99298+05	0.00000	3.46116+05	6.41591+01	0.00000	0.00000	
	2	1.94000+05	0.00000	1.63338+05	7.34140+00	0.00000	0.00000	
	3	2.00227+05	0.00000	-1.79390+04	1.15735+02	0.00000	0.00000	
6	1	2.00227+05	0.00000	-1.79390+04	1.15735+02	0.00000	0.00000	
	2	-2.77092+04	0.00000	-1.75055+06	6.22583+02	0.00000	0.00000	
	3	-2.55985+05	0.00000	-1.71806+06	1.30474+03	0.00000	0.00000	
7	1	-1.17971+00	-1.80089+05	-9.03886+01	6.45436+01	-1.60406+03	0.00000	
	2	-1.18416+06	-1.76168+05	7.45955+04	7.07965+01	-2.67602+03	8.72523+01	
	3	-1.19755+06	-1.72248+05	1.49384+05	9.31660+01	-3.94342+03	2.19103+02	
8	1	-1.13755+06	-1.66503+05	1.84052+05	6.62313+01	-2.82698+03	2.19103+02	
	2	-5.20046+05	-9.03278+04	2.94119+05	2.07970+02	1.48680+03	3.07018+02	
	3	-8.00374+04	-1.51525+04	4.08272+05	2.62257+02	5.06606+03	-4.91593+01	
9	1	-6.20046+05	-9.03278+04	3.45056+05	2.07970+02	3.26310+03	3.07018+02	
	2	-8.00374+04	-1.05144+04	3.57334+05	2.32257+02	3.27425+03	-4.91593+01	
	3	4.55470+05	7.44372+04	3.74503+05	1.59191+02	2.58193+03	-3.44682+02	
10	1	-8.00374+04	-7.91632+03	4.08073+05	2.82257+02	5.07210+03	-4.91593+01	
	2	4.55470+05	7.19391+04	3.23005+05	1.93191+02	8.01341+02	3.44682+02	
	3	1.01922+06	1.51595+05	2.44011+05	6.14983+01	-4.20862+03	-7.13452+01	
11	1	1.01922+06	1.69160+05	2.10003+05	8.97105+01	-4.14405+03	-7.13452+01	
	2	1.00793+06	1.80944+05	8.20761+04	9.20286+01	-4.93271+03	3.73023+02	
	3	1.02772+06	1.92729+05	-4.54103+04	1.70254+02	-6.51315+03	9.16764+02	
12	1	1.02772+06	1.82956+05	-1.15908+05	1.23034+02	-6.59747+03	9.16764+02	
	2	-1.30240+05	9.13199+04	-9.04414+04	4.58566+02	-2.28013+03	1.51923+03	
	3	-1.29421+06	-1.31640+03	-7.01490+04	8.76791+02	1.15464+03	1.62789+03	
13	1	-1.67545+06	-1.81953+05	2.03683+01	1.19261+02	-1.66257+03	0.00000	
	2	-1.68119+06	-1.76204+05	3.05615+03	1.13142+02	-2.70906+03	5.07903+00	

3	-1.69843+C6	-1.70454+C5	6.03641+C3	1.13698+C2	-3.94554+C3	9.12113+C0
1	-1.69843+C6	-1.67204+C5	7.09242+C3	8.39285+C1	-2.81422+C3	9.12113+C0
2	-8.83065+C5	-9.25464+C4	1.22015+C4	7.96514+C1	1.53359+C3	7.33004+C0
3	-1.15725+C5	-1.73395+C4	1.72070+C4	7.02697+C1	5.23138+C3	-1.20327+C1
1	-8.30065+C5	-9.94593+C4	1.39420+C4	7.96514+C1	3.49719+C3	7.33004+C0
2	-1.16725+C5	-1.09756+C4	1.54864+C4	7.02697+C1	3.32919+C3	-1.20327+C1
3	6.43447+C5	7.75781+C4	1.71843+C4	6.25573+C1	2.39833+C3	-1.79947+C1
1	-1.16725+C5	-7.20836+C3	1.72012+C4	7.02697+C1	5.24593+C3	-1.20327+C1
2	6.43447+C5	7.37408+C4	1.54495+C4	6.25573+C1	4.85051+C2	-1.79947+C1
3	1.44027+C6	1.54690+C5	1.37092+C4	7.93232+C1	-5.04559+C3	2.16472+C1
1	1.44027+C6	1.73611+C5	1.28642+C4	1.15262+C2	-5.05109+C3	2.16472+C1
2	1.42759+C6	1.84897+C5	8.75306+C7	1.56716+C2	-5.93836+C3	8.18991+C1
3	1.45580+C6	1.96183+C5	4.76230+C3	2.24773+C2	-7.59463+C3	1.63933+C2
1	1.45580+C6	1.86021+C5	3.19333+C3	1.61432+C2	-7.68377+C3	1.63933+C2
2	-1.70094+C5	9.23985+C4	9.55191+C2	2.94750+C2	-2.83162+C3	2.62232+C2
3	-1.81788+C6	-1.22406+C3	-1.45782+C3	4.52840+C2	1.25065+C3	2.94813+C2
1	-1.25455+C6	-1.84339+C5	-9.99534+C1	9.49657+C1	-1.74728+C7	0.00000
2	-1.26077+C6	-1.78142+C5	6.86713+C4	9.51860+C1	-2.79534+C3	3.94675+C0
3	-1.27355+C6	-1.71945+C5	-1.37462+C5	9.57674+C1	-4.03367+C3	9.09335+C0
1	-1.27955+C6	-1.68635+C5	-1.67235+C5	6.77440+C1	-2.89048+C3	9.09335+C0
2	-6.58956+C5	-9.48336+C4	-2.71144+C5	6.50590+C1	4.95994+C3	3.43712+C1
3	-9.21514+C4	-2.10318+C4	-3.74895+C5	6.44554+C1	5.32139+C3	3.17069+C1
1	-6.58956+C5	-1.02159+C5	-3.17127+C5	6.92590+C1	3.49978+C3	2.43712+C1
2	-9.21514+C4	-1.37062+C4	-3.28212+C5	6.44554+C1	3.41829+C3	3.17069+C1
3	4.69380+C5	7.47468+C4	-3.42035+C5	5.56584+C1	2.57434+C3	-3.33822+C0
1	-9.21514+C4	-9.46019+C3	-3.75034+C5	6.44554+C1	5.33203+C3	3.17069+C1
2	4.62200+C5	7.05007+C4	-2.95943+C5	5.56384+C1	6.63329+C2	-3.33822+C0
3	1.06936+C6	1.50462+C5	-2.17306+C5	6.19455+C1	-4.77329+C3	-1.19309+C2
1	1.06936+C6	1.70320+C5	-1.87551+C5	9.15447+C1	-4.77577+C3	-1.19309+C2
2	1.05299+C6	1.85187+C5	-6.66643+C4	1.20124+C2	-5.85470+C3	-2.26127+C2
3	1.09099+C6	2.00053+C5	5.45177+C4	1.74486+C2	-7.70240+C3	-3.09382+C2
1	1.08099+C6	1.91324+C5	1.21127+C5	1.26328+C2	-7.78395+C3	-3.09382+C2
2	-1.11025+C5	9.55220+C4	9.41637+C4	2.44822+C2	-2.87626+C3	-3.85786+C2
3	-1.30499+C6	-2.80115+C2	6.71020+C4	3.90347+C2	1.25708+C3	-4.10906+C2
1	6.30500+C4	-1.52044+C5	6.45097+C2	-1.27945+C3	-1.74823+C3	0.00000
2	6.51377+C4	1.66993+C5	-1.78214+C5	5.58922+C0	-2.82230+C3	-9.05900+C1
3	7.15675+C4	-1.71941+C5	-2.77297+C5	2.39565+C1	-4.09253+C3	-2.25916+C2
1	7.15675+C4	-1.55566+C5	-3.52766+C5	2.29698+C1	-2.97835+C3	-2.25916+C2
2	2.14424+C4	-7.92388+C4	-5.38813+C5	1.78721+C2	1.47720+C3	-3.25085+C2
3	-1.01933+C4	-2.81172+C3	-7.57541+C5	2.65996+C2	5.20094+C3	4.23415+C1
1	2.14424+C4	-7.27047+C4	-6.44521+C5	1.78721+C2	3.23914+C3	-3.25085+C2
2	-1.01933+C4	-9.34586+C3	6.51892+C5	2.65996+C2	3.42118+C3	4.23415+C1

3	-3.22734+C4	5.40130+C4	-6.93343+C5	1.93785+C2	2.90706+C3	3.58267+C2
1	-1.01973+C4	-1.38497+C4	-7.55703+C5	2.65996+C2	5.19466+C3	4.28416+C1
2	-3.92734+C4	5.85168+C4	-5.89412+C5	1.83785+C2	1.14919+C3	3.58267+C2
3	-8.17614+C4	1.30883+C5	4.56641+C5	1.95462+C1	-3.62365+C3	1.07679+C2
1	-8.17614+C4	1.51355+C5	-3.86559+C5	2.70769+C1	-3.54286+C3	1.07679+C2
2	-9.36738+C4	1.70202+C5	-1.44664+C5	-1.97989+C1	-4.71533+C3	-3.25315+C2
3	-1.20912+C5	1.89049+C5	9.39707+C4	2.06775+C1	-6.67672+C3	-8.75470+C2
1	-1.20912+C5	1.87220+C5	2.45156+C5	1.81309+C1	-6.76017+C3	-8.75470+C2
2	6.09531+C4	3.18094+C4	1.75550+C5	2.94933+C2	-2.36794+C3	-1.48382+C3
3	2.47280+C5	-3.60111+C3	1.46113+C5	6.60089+C2	1.24457+C3	-1.59460+C3
1	1.03209+C6	7.76669+C5	8.13528-C1	-1.33281+C2	-1.08884+C0	C.00000
2	1.04418+C6	7.71262+C5	-1.44933+C4	-1.47313+C2	0.00000	4.81440+C0
3	1.07765+C6	7.65856+C5	-2.89883+C4	-1.90503+C2	2.64614+C0	9.61763+C0
1	1.07705+C6	1.06746+C6	-3.56473+C4	-1.42750+C2	1.19395+C2	9.61763+C0
2	5.13992+C5	5.49599+C5	-5.07245+C4	-4.54151+C2	-4.24830+C1	1.49929+C1
3	4.67094+C4	3.11735+C4	-7.78839+C4	-6.15711+C2	-1.19773+C2	3.77792+C0
1	5.13952+C5	5.55827+C5	-6.67112+C4	-4.54151+C2	-6.60920+C1	1.49929+C1
2	4.63094+C4	2.57011+C4	-6.78092+C4	-6.15711+C2	-4.35040+C1	3.77792+C0
3	-4.10220+C5	-5.04810+C5	6.91288+C4	-4.23082+C2	-4.16491+C1	-2.44419+C1
1	4.63084+C4	2.41453+C4	-7.79126+C4	-6.15711+C2	-8.16188+C1	3.77792+C0
2	-4.10230+C5	-5.03448+C5	-5.91052+C4	-4.29882+C2	-4.32812+C1	-2.44194+C1
3	-9.38855+C5	-1.03104+C6	-4.03383+C4	-1.11196+C2	5.38189+C1	-6.99318+C1
1	-9.38855+C5	-7.41348+C5	-3.22584+C4	-1.73244+C2	1.59418+C1	-6.99318+C1
2	-9.27436+C5	-8.20156+C5	-8.83614+C3	-1.35935+C2	C.00000	-9.92523+C1
3	-9.06005+C5	-8.98965+C5	1.47293+C4	-7.22405+C2	8.81280+C1	-1.10454+C2
1	-9.96005+C5	-1.23499+C6	2.47609+C4	-2.47358+C2	-2.99128+C1	-1.10454+C2
2	1.88457+C5	-6.04023+C5	1.81428+C4	-1.32542+C3	-5.17613+C1	-1.12721+C2
3	1.37499+C6	2.69409+C4	1.14095+C4	-2.02922+C3	8.53825+C0	-1.13499+C2
1	4.31856+C5	-1.63821+C5	-6.43300+C2	-2.21747+C1	-1.87160+C3	C.00000
2	4.34034+C5	-1.63914+C5	1.09558+C5	-1.53293+C1	-2.94561+C3	1.00527+C2
3	4.40554+C5	-1.73806+C5	2.19987+C5	8.85875+C0	-4.21543+C3	2.45887+C2
1	4.40554+C5	-1.56988+C5	2.82459+C5	8.12137+C0	-3.10202+C3	2.45887+C2
2	2.12320+C5	-8.19328+C4	4.26399+C5	1.53181+C2	1.50314+C3	3.57603+C2
3	2.54815+C3	-6.99738+C3	6.02901+C5	2.40760+C2	5.37744+C3	-3.00777+C1
1	2.12320+C5	-7.55460+C4	5.12550+C5	1.63181+C2	3.26378+C3	3.57603+C2
2	2.54815+C3	-1.34442+C4	5.16749+C5	2.43766+C2	3.59914+C3	-3.00777+C1
3	-2.00382+C5	4.80576+C4	5.55029+C5	1.53690+C2	3.23896+C3	-3.99585+C2
1	2.54815+C3	-1.82106+C4	6.01055+C5	2.40760+C2	5.37142+C3	-3.00777+C1
2	-2.00382+C5	5.34240+C4	4.70724+C5	1.53690+C2	1.49241+C3	-3.99585+C2
3	-4.29146+C5	1.25059+C5	3.73889+C5	-3.99533+C0	-3.13361+C3	-2.39969+C2
1	-4.28146+C5	1.46401+C5	3.19451+C5	-7.21715-C2	-3.05167+C3	-2.39969+C2
2	-3.05033+C5	1.69167+C5	1.24191+C5	-4.03963+C1	-4.49502+C3	1.81648+C2

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7

6	3	-3.77116+C5	1.91934+C5	-6.71107+C4	-9.20367+C3	-5.04920+C3	6.53994+C2
	1	-3.77116+C5	1.91504+C5	-1.96961+C5	1.34505+C3	-6.73292+C3	5.53994+C2
	2	1.18946+C5	9.40089+C4	-1.39603+C5	2.80346+C2	-2.35404+C3	1.24954+C3
	3	6.20394+C5	-3.48653+C3	-1.22636+C5	6.50410+C2	1.24487+C3	1.35533+C3
1	1	-5.24588+C5	-1.89414+C5	8.17294+C1	5.03978+C1	-2.00033+C3	0.00000
	2	-5.30806+C5	-1.82172+C5	4.09126+C4	5.19639+C1	-3.05073+C3	-3.30791+C0
	3	-5.49812+C5	-1.75931+C5	8.19155+C4	5.47453+C1	-4.29174+C3	-6.69695+C0
2	1	-5.49812+C5	-1.71494+C5	9.96258+C4	3.97440+C1	-3.15010+C3	-5.69695+C0
	2	-2.77675+C5	-1.00213+C5	1.61489+C5	5.05578+C1	1.62778+C3	-1.24335+C1
	3	-6.01432+C4	-2.89329+C4	2.23270+C5	5.03563+C1	5.64620+C3	-9.46657+C0
3	1	-2.77675+C5	-1.07724+C5	1.89035+C5	5.05578+C1	3.52332+C3	-1.24335+C1
	2	-6.01432+C4	-2.14223+C4	1.95674+C5	5.05568+C1	3.75018+C7	-9.46657+C0
	3	1.49379+C5	6.48794+C4	2.03424+C5	3.26237+C1	3.21995+C3	5.31342+C0
4	1	-6.01432+C4	-1.75035+C4	2.23366+C5	5.03568+C1	5.65441+C3	-9.46657+C0
	2	1.49879+C5	6.09606+C4	1.75732+C5	3.26233+C1	1.31797+C3	6.31342+C0
	3	3.98514+C5	1.39425+C5	1.28496+C5	1.63918+C1	-3.78246+C3	4.14737+C1
5	1	3.98514+C5	1.60830+C5	1.11145+C5	4.09534+C1	-3.77611+C3	4.14237+C1
	2	4.47734+C5	1.83336+C5	3.79501+C4	5.72481+C1	-5.28165+C3	6.43577+C1
	3	5.42428+C5	2.05941+C5	-3.54300+C4	1.39862+C2	-7.55938+C3	6.73464+C1
6	1	5.42428+C5	1.99697+C5	-7.68678+C4	8.73214+C1	-7.63403+C3	5.73464+C1
	2	-4.22598+C3	9.98633+C4	-5.92802+C4	2.30124+C2	-2.79326+C3	5.91314+C1
	3	-5.49914+C5	3.90185+C1	74.16172+C4	4.18680+C2	1.25721+C3	5.52743+C1
7	1	-5.99329+C5	-1.82913+C5	7.84903+C1	3.13952+C1	-1.78784+C3	0.00000
	2	-6.07777+C5	-1.78938+C5	-2.84900+C4	3.77246+C1	-2.86018+C7	-1.00533+C2
	3	-6.16979+C5	-1.75364+C5	-5.91805+C4	6.03534+C1	-4.12803+C7	-2.45926+C2
8	1	-6.16979+C5	-1.68942+C5	-7.35504+C4	4.39284+C1	-3.01062+C3	-2.45926+C2
	2	-3.17796+C5	-9.50459+C4	-1.16900+C5	1.85821+C2	1.51784+C3	-3.50954+C2
	3	-5.64254+C4	-2.15501+C4	-1.64139+C5	2.59799+C2	5.31118+C3	3.00367+C1
9	1	-3.17796+C5	-9.98529+C4	-1.37263+C5	1.85821+C2	3.29489+C3	-3.50954+C2
	2	-5.64254+C4	-1.67432+C4	-1.43827+C5	2.59799+C2	3.51839+C3	3.00367+C1
	3	1.99279+C5	6.63555+C4	-1.55101+C5	1.73137+C2	3.03825+C7	3.91324+C2
10	1	-5.64264+C4	-1.45105+C4	-1.63832+C5	2.59799+C2	5.31610+C3	3.00367+C1
	2	1.99279+C5	6.41338+C4	-1.35155+C5	1.73137+C2	1.25792+C3	3.94324+C2
	3	4.81218+C5	1.42778+C5	-1.10815+C5	2.79252+C1	-3.53866+C3	2.38914+C2
11	1	4.81218+C5	1.61954+C5	-9.95395+C4	4.95458+C1	-3.47291+C3	2.38914+C2
	2	5.20313+C5	1.79877+C5	-4.77616+C4	3.47954+C1	-4.62035+C3	-1.32047+C2
	3	5.30771+C5	1.97901+C5	3.13605+C3	1.10733+C2	-6.55992+C3	-5.49816+C2
12	1	5.90771+C5	1.91149+C5	3.62924+C4	9.03590+C1	-6.63890+C3	-5.49816+C2
	2	-3.66176+C4	9.50172+C4	3.06253+C4	4.21402+C2	-2.29641+C3	-1.24670+C3
	3	-6.68582+C5	-1.11458+C3	3.05036+C4	8.40540+C2	1.26005+C3	-1.35261+C3

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10

1	-2.23953+05	0.00000	1.97838+02	1.18026+01	0.00000	0.00000
2	-2.25464+05	0.00000	-6.12236+04	2.34852+01	0.00000	-2.70292+02
3	-2.30353+05	0.00000	-1.22836+05	6.59615+01	0.00000	-6.70680+02
1	-2.30353+05	0.00000	-7.44984+05	4.73129+01	0.00000	-6.78680+02
2	-1.16918+05	0.00000	-8.31827+05	3.21064+02	0.00000	-9.84341+02
3	-1.66374+04	0.00000	-9.31000+05	4.68642+02	0.00000	6.26275+01
1	-1.16918+05	0.00000	-9.76979+05	3.21064+02	0.00000	-9.84841+02
2	-1.66374+04	0.00000	-8.85888+05	4.68642+02	0.00000	6.26275+01
3	8.21123+04	0.00000	-9.08442+05	3.13650+02	0.00000	1.07019+03
1	-1.66374+04	0.00000	-9.30337+05	4.68642+02	0.00000	6.41734+02
2	8.21123+04	0.00000	-7.86339+05	3.13650+02	0.00000	1.07019+03
3	1.90665+05	0.00000	-8.10700+05	3.84587+01	0.00000	6.41734+02
1	1.90665+05	0.00000	-3.41280+05	5.81383+01	0.00000	6.41734+02
2	1.95126+05	0.00000	-2.34264+05	1.23427+01	0.00000	-3.15955+02
3	2.11020+05	0.00000	-1.29204+05	1.23605+02	0.00000	-1.60028+03
1	2.11020+05	0.00000	1.12383+06	9.33906+01	0.00000	-1.60028+03
2	-2.42032+04	0.00000	1.09735+06	6.12880+02	0.00000	-3.07272+03
3	-2.45995+05	0.00000	1.08660+06	1.26213+03	0.00000	-3.28282+03

6FIN

PUNTD: BARKER ACCT: ACECS-3033 PROJECT: PROJCTY
 TIME: TOTAL: CC:02:31.799 I/O: CC:00:02.257
 CPU: CC:02:27.125 WAIT: CC:00:00.000
 CC/PR: CC:00:02.41E
 IMAGES READ: 76 PAGES: 11
 START: C9:15:44 SEP 22,1976 FIN: C9:18:47 SEP 22,1976

APPENDIX F:MENAI - USERS' MANUAL

The program is liberally documented and the READ statements are explained in the various sub-routines. All data is entered in FREE FORMAT, units being combinations of Newtons and metres.

MAIN PROGRAM: Calls the sub-routine in a specific sequence, and monitors all the output. No data is read by this program.

Dimension parameters indicating the size of large matrices must be changed to suit each structure:

<u>Sequence No.</u>	<u>Parameter</u>
25	MHC
26	MSKD
27	MSKB
28	NW27
29	JE1

where:

- MHC = Size of "Stiffness" matrix for 1 cross-section (= Degrees of freedom per section).
- MSKD = Size of Diagonal in total Stiffness matrix (= Degrees of freedom in complete structure).
- MSKB = Half Band Width in Stiffness matrix (= $3 \times \text{MHC}$)
- NW27 = Column in Stiffness matrix for placing the load vector. (= $\text{MSKB} + 1$).
- JE1 = Size of displacement vector to accommodate transformed global displacements. (= (Number of 2-strip sub-structures) \times 12 \times (Number of cross-sections in Structure)).

The common storage block "EXT" must be changed for each different structure as indicated:

```
COMMON/EXT/SKU(MSKD,NW27),SD4CA(MSKD,5),SD2CG(MSKD,3),SDOZE(MSKD),
.SD1CE(MSKD,3),CA(MHC,MHC),CG(MHC,MHC),CE(MHC,MHC),
```

.CZ(MHC,MHC),SIGMAX(JE1/4),SIGMAY(JE1/4),TAUXY(JE1/4),
 .AMX(JE1/4),AMY(JE1/4),AMXY(JE1/4)

SUB-ROUTINE DEES: Formulates the elastic constants.

<u>Sequence No.</u>	<u>Variables Read</u>
16	DEX,DEY,DPX,DPY
19	IDNUM
23	DH(I),DB(I) (I = 1,IDUM)

where:

DEX = Young's Modulus in the X axis direction.
 DEY = " " " " Y " "
 DPX = Poisson's Ratio in the X axis direction.
 DPY = " " " " Y " "
 IDNUM = Number of strips in structure (Max. = 30).
 DH(I) = Strip thickness.
 DB(I) = Strip widths.

SUB-ROUTINE FORMA: Formulates the basic strip stiffness matrices. No data is read by this sub-program.

SUB-ROUTINE ANGLE: Compiles the transformation matrices for individual strips and produces global stiffness matrices.

<u>Sequence No.</u>	<u>Variable Read</u>
27	ANG(I) (I = 1,IDUM)

where:

ANG(I) = Strip angle to global co-ordinate directions. One strip angle per card, entry sequence same as thickness and width cards.

SUB-ROUTINE CONTAC: Formulates the combined cross-section stiffness matrices.

<u>Sequence No.</u>	<u>Variable Read</u>
20	LC3,LC4
38	LC5
49	LC6,LC7,LC8,LC11,LC12

where:

- LC3 = Strip number.
- LC4 = Row number where the L.H. top corner of the strip stiffness matrices belong in the combined matrices. Repeat for all 'normal' strips. Place 0,0 after last entry to stop process.
- LC5 = Number of strip stiffness matrices that require 'split' placing. (If split placing is not required, enter 0 and program automatically skips this process).
- LC6 = Strip number.
- LC7 = Row number in combined matrices where strip stiffness sub-matrices belong.
- LC8 = Column number (ditto).
- LC11 = Row number in sub-matrices where the copy process starts.
- LC12 = Column number in sub-matrices where copy process starts.

Repeat step 49 four (4) times for every strip stiffness matrix that requires 'split' placing. Information given first for L.H. top section (4×4), then R.H. top section, then L.H. bottom section, and finally R.H. bottom section (4×4). Place 0,0,0,0,0 after last entry to stop process.

SUB-ROUTINE SILO: Formulates the Finite Difference equations, multiplies them with the combined cross-sectional stiffness matrices to give the upper half of the banded stiffness matrix.

<u>Sequence No.</u>	<u>Variable Read</u>
247	MS(I) (I = 1,16)
249	MS(I) (I = 17,31)
254	ME(I) (I = 1,16)
256	ME(I) (I = 17,31)
432	SA (Min No. nodes in longitudinal direction = 7)
453	MS 310
459	MS 401
464	MSZERO(I)

where:

MS(I) = Single digit numbers indicating boundary conditions at start of structure. 1 = Fixed, 3 = Guided, 7 = Simply Supported. (Zeroes to be used to fill field).

ME(I) = Ditto ... at end of structure.
 SA = Longitudinal nodal spacing (metres).
 MS310 = Number of rows to be reduced to zero in stiffness and Load Matrix (Columns are automatically done as well)
 MS401 = Number of cards to be read with ten row numbers per card. (Zeroes are used to fill field).
 MSZERO(I) = Vector of numbers entered on MS401 cards (Max. = 250)

SUB-ROUTINE WHEEL: Formulates the load matrix.

<u>Sequence No.</u>	<u>Variable Read</u>
46	WU(I), WV(I), WW(I), WTHETA(I) (I = 1, to number of nodes per cross-section)
100	NW19
105	NW10(I), NW21(I), WLU(I), WLV(I), WLW(I), WLTHE(I) (I = 1, NW19)

where:

WU(I) = Cross-section dead load in global X direction.
 WV(I) = Half cross-section dead load (N/m) in global Y direction.
 WW(I) = Half cross-section dead load (N/m) in global Z direction.
 WTHETA(I) = Half cross-section twisting dead load about X axis (Nm/m).

One card is read for every nodal point on cross-section.

NW19 = Number of nodes having global live loads. (Max. number = 50)
 If no live loads - use dummy.
 NW20 = Node number for load.
 NW21 = Cross-section number for load.
 WLU(I) = Global live load in X direction (N).
 WLV(I) = " " " " Y " (N/m).
 WLW(I) = " " " " Z " (N/m).
 WLTHE(I) = Global twisting live load about X axis (Nm/m).

One card read for each node having global live loads.

SUB-ROUTINE GRAFT: Solves the linear algebraic equations to give global displacements. No data is read by this sub-program.

SUB-ROUTINE ECHO: Solves stresses in local axes co-ordinates. The structure is now thought of as a combination of two-strip sub-structures to facilitate the application of the Finite Difference operator patterns. Sections may have an odd number of strips; this merely leads to a duplication of stress and moment calculations for a particular node or the cross-section. Strips belonging to the same sub-structure must have the same thickness and width.

<u>Sequence No.</u>	<u>Variable Read</u>
23	JENUM
30	JETRID(I), JEST(I), JEMID(I), JEEND(I) (I = 1, JENUM)

where:

- JENUM = Number of two-strip sub-structures.
- JETRID(I) = Strip number for transformation of displacements into local axis directions.
- JEST(I) = Row number in SKU(,NW27) matrix, on first cross-section, where first node displacements begin.
- JEMID(I) = Row number in SKU(,NW27) matrix, on first cross-section, where second node displacements begin.
- JEEND(I) = Row number in SKU(,NW27) matrix, on first cross-section, where third node displacements begin.

SIGN CONVENTION FOR OUTPUT

STRESSES: (In local co-ordinate directions)

- SIGMA - X: Positive indicates tensile stress. (N/m^2)
- SIGMA - Y: Positive indicates tensile stress. (N/m^2)
- TAU - XY : Positive as indicated.
- M - X : Positive induces sagging of strip.
- M - Y : Positive induces sagging of strip.
- M - XY : Positive as indicated (= - (M - YX)).

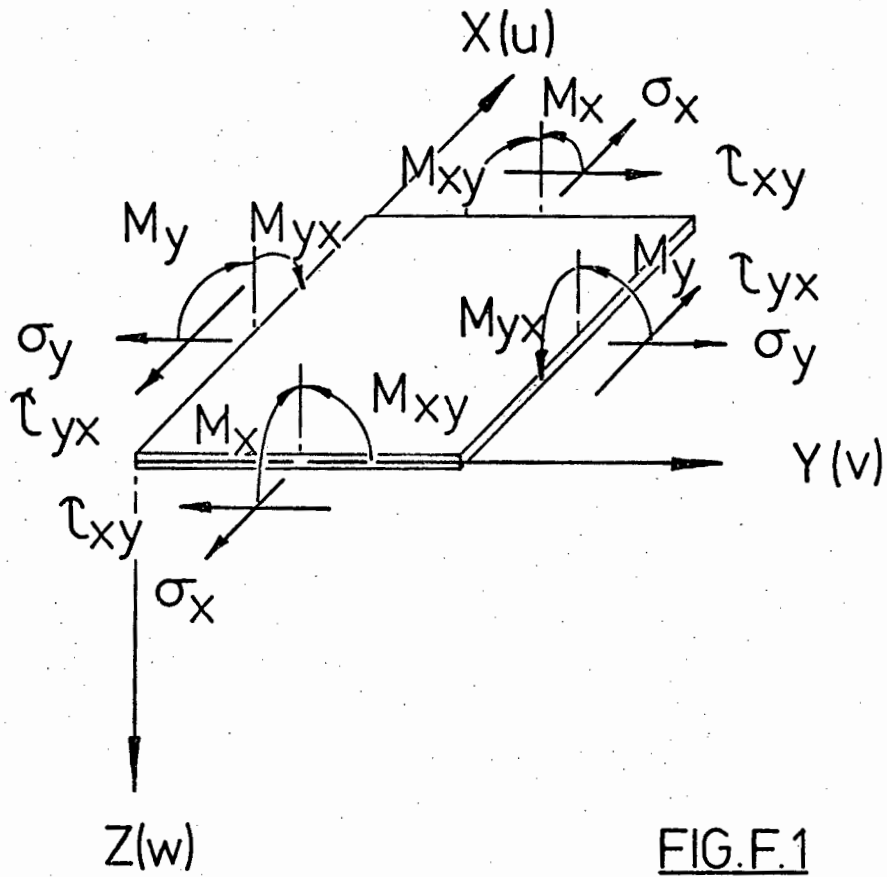


FIG.F.1